# EVALUATION OF CRYOGENIC NITROGEN AS A FIRE-EXTINGUISHING AGENT FOR AIRCRAFT POWERPLANT INSTALLATIONS

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# FINAL REPORT

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#### 16. Abstract

Proposals have been made to carry relatively large quantities of liquid nitrogen (LN2) aboard commercial aircraft for the purpose of fuel tank inerting. Secondary uses, such as powerplant fire extinguishment, have been suggested. Testing was conducted at the National Aviation Facilities Experimental Center to determine the feasibility of using LN2 as an aircraft powerplant fire-extinguishing agent and also to determine the characteristics of LN2 when used as an extinguishant. These tests were conducted in a fire test facility using a full-scale aircraft turbojet engine and nacelle for subsonic lowaltitude flight condition simulation and also in a mockup engine/ nacelle facility where nacelle volume and airflow could be varied. For all tests, the  $LN_2$  was delivered from a dewar where it was stored under pressure as a saturated liquid. All fire tests were conducted using JP-4 jet fuel which was spray released and spark ignited. In addition to the preliminary feasibility study, this report describes the experimentation conducted to determine the design criteria required for an effective agent quantity, discharge rate, discharge conditions, and distribution system configuration. The report also describes the effects of an inadvertent discharge on engine components, the effects of a damaged cowling, and the cooling of potential reignition sources.

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#### LIST OF SYMBOLS

An	=	Nozzle cross-sectional discharge area, in2
At	=	Nominal inside cross-sectional area of tubing, in2
FR	= = ; : : : :	Dewar weight-fill ratio. Rated liquid nitrogen capacity (100 lbs) divided by actual fill weight, percent.
GN <sub>2</sub>	=	Gaseous nitrogen
LN <sub>2</sub>	=	Liquid nitrogen
N <sub>2</sub>	=	Gaseous and liquid nitrogen mixture
$P_{D}$	=	Dewar pressure, psig
$P_{D_S}$	=	Dewar saturation pressure, psig
$P_{D}$	=	Average dewar pressure for specifically grouped data, psig
Pn	=	Static pressure immediately upstream of discharge nozzle, psig
P1,2,a,b,	.=	Pressure at various designated stations in distribution system, psig
$V_{\mathbf{V}}$	=	Void volume of test article, ft <sup>3</sup>
ŴA	=	Weight rate of flow of nacelle air, lb/sec
ŴF	=	Rate of fuel flow, gpm
WGN <sub>2</sub>	=	Weight rate of flow of gaseous nitrogen, lb/sec
$W_{LN_2}$	=	Weight of liquid nitrogen in dewar, lbs
WLN <sub>2</sub>	=	Weight of flow of liquid nitrogen from the dewar, lbs/sec
W <sub>N2</sub>	=	Weight rate of flow of gaseous and liquid nitrogen mixture, lbs/sec
x	=	Mixture quality, ratio of gaseous nitrogen weight to total mixture weight
X	= 1	Average mixture quality for specifically grouped data
$\overline{x}_1$	=	Average mixture quality at Station P <sub>l</sub> for specifically grouped data

#### INTRODUCTION

#### Purpose

The purpose of this project was to determine the feasibility of using cryogenic nitrogen as an aircraft powerplant fire extinguishing agent and to provide fundamental design criteria for an effective extinguishing system utilizing that agent.

#### Background

Liquid nitrogen (LN2) inerting systems have been used for several years on military aircraft, such as the B-52, SR-71, and B-70, to inert the fuel systems in an attempt to prevent ignition in the fuel tanks. Hardware has been developed for installation and flight tests in a C-135 and a C-141 aircraft. Adoption of fire protection for fuel systems in commercial aircraft is being considered, and liquid nitrogen inerting is among the methods being considered for this purpose. Such a system would require several hundred pounds of LN2 to be carried aboard the aircraft. Due to the availability of large quantities of LN2 for inerting when such a system is used, additional uses are being investigated, such as for extinguishing powerplant fires.

On July 1, 1968, Flight Standards Service issued request No. FS-100-68-92 for an R, D, and E effort to investigate the extinguishing properties of cryogenic nitrogen and to determine the best method of using it in an installed fireextinguishing system. At the request of Aircraft Development Service, a project was initiated at the National Aviation Facilities Experimental Center (NAFEC) in August 1968 to provide the required information. The first phase of testing under this project was initiated on September 3, 1968, and was completed on November 5, 1968. A complete description of the first phase of the project is presented in Appendix A. The test results indicated that- (1) LN2 was effective in extinguishing fires in aircraft powerplant compartments, (2) the reserve quantity of LN2 (approximately 100 pounds) expected to be available from an LN2 fuel tank inerting system in a large commercial transport aircraft would be sufficient to extinguish fires, and (3) on aircraft where a large quantity of LN2 is available, an LN2 fireextinguishing system could provide greater in-flight powerplant fire protection than could the limited quantity of agent available in a conventional high-rate-discharge extinguisher system.

Based upon the determination that the use of  $LN_2$  was feasible for powerplant fire protection, a second phase of the project was initiated in September 1969. The second phase was conducted to experimentally define the requirements for an effective extinguishing system as influenced by nacelle ventilation and free volume and in terms of agent quantity, discharge rate, discharge conditions and distribution provisions. The effects of an inadvertent discharge, damaged cowling, and the cooling of potential reignition sources were also investigated. The second phase of the project, which finished the assigned test program, was completed in July 1970.

#### DISCUSSION

# Characteristics of LN2 as a Fire-Extinguishing Agent

Like carbon dioxide, the effectiveness of LN2 in extinguishing fires is dependent upon (1) oxygen dilution to the level that will no longer support combustion, and (2) cooling to reduce the temperature of the combustible below its ignition temperature or the point at which it vaporizes. A comparison of the physical properties of LN2, carbon dioxide, and the two most common halogenated fireextinguishing agents (CBr<sub>2</sub>F<sub>2</sub> and CBrF<sub>3</sub>) currently in use on U. S. military and commercial aircraft is made in Table 1. Since nitrogen at atmospheric pressure has a lower boiling point than the other three agents and a higher heat of vaporization than the two halogenated agents, the amount of cooling during an LN2 discharge can be expected to be greater when compared on a weight basis. Likewise, since the expansion ratio of nitrogen when converted from a liquid to a gas is considerably higher than the other three agents, nitrogen produces the greatest amount of oxygen dilution. The overall effectiveness of LN2 as a fire-extinguishing agent, however, cannot be expected to be as great as the highly effective halogenated agents. These agents do not depend primarily on oxygen dilution and cooling, but on a chemical interference with the combustion process. The lower effectiveness of nitrogen does not eliminate it from consideration as a fireextinguishing agent on aircraft where large quantities can be made available from the reserve supply of LN2 stored for inerting fuel tanks and other purposes.

TABLE 1. - PHYSICAL PROPERTIES OF SEVERAL EXTINGUISHANTS

LN <sub>2</sub>	CO <sub>2</sub>	CBrF3	CBr <sub>2</sub> F <sub>2</sub>
-320	-109	-72	76
85	113	48	53
14	9	3	3
	-320 85 14 696:1	-320 -109 85 113 14 9 696:1 403:1	-320 -109 -72 85 113 48 14 9 3

#### Test Facilities and System Installation

Two wind-tunnel-type facilities at NAFEC were utilized in the conduct of the tests. These facilities were the 5-foot Fire Test Facility, utilizing a JT-12 turbojet engine with a left-hand, in board, C-140 engine/nacelle installation as the test article, and a mockup engine/nacelle facility.

Five-Foot Fire Test Facility: The Five-Foot Fire Test Facility, shown in Figure 1, is described in detail in Systems Research and Development Service (SRDS) Handbook RD P 6000.2, entitled "Technical Facilities at NAFEC." Airflow through the facility's 20-foot-long by 5-foot-diameter cylindrical test section was induced by the ejector pumping action of two J-57 turbojet engines located downstream of the test section. The airflow simulated subsonic, low-altitude flight conditions around the C-140 engine nacelle, which is shown installed in the test section in Figure 2.

The nacelle was divided into two fire zones by a vertical transverse fire seal. The LN2 extinguishing system and instrumentation were installed in the nacelle accessory and compressor section (Zone II). The void volume within Zone II was approximately 12.6 cubic feet. For the majority of the tests, the source of airflow within the nacelle was the engine compressor interstage air bleed. The bleed air discharged from a series of orifices around the periphery of the engine at the fourth compressor stage. The flow was a function of the engine primary airflow rate. The compressor interstage ports were open from engine start to approximately 81-percent compressor rotational speed (N1).

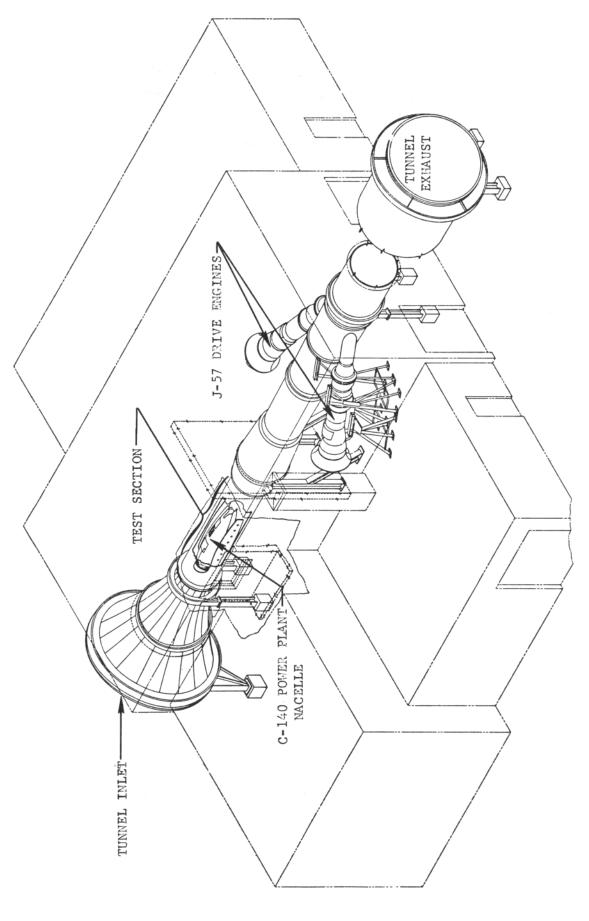


FIGURE 1 FIVE-FOOT FIRE TEST FACILITY

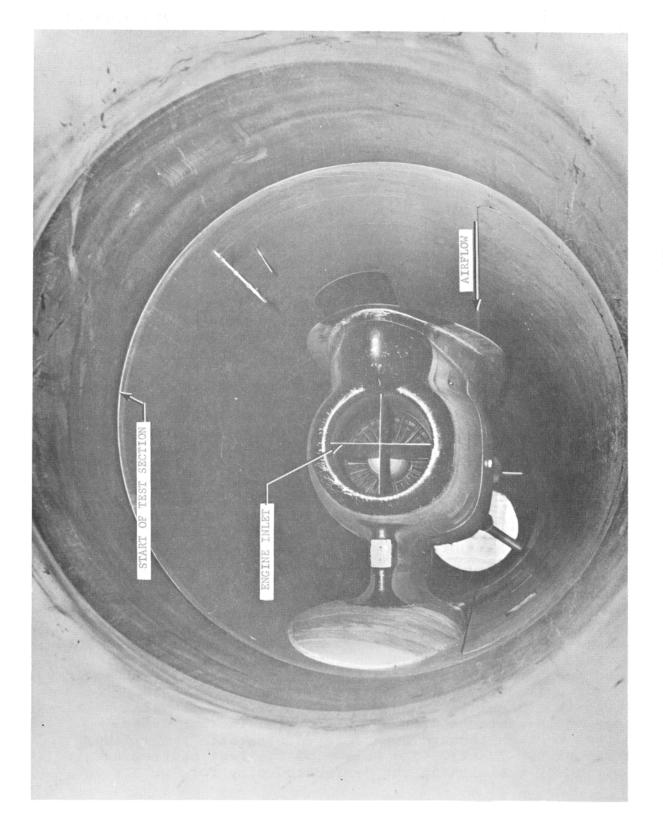


FIGURE 2 TEST ENGINE AND NACELLE INSTALLED IN FIRE TEST FACILITY

LN<sub>2</sub> extinguisher systems were developed and used in this facility to extinguish test fires in the compressor and accessory compartment of the turbojet powerplant nacelle. The LN<sub>2</sub> storage container (dewar) and a typical distribution system used during the tests are shown in the Fire Test Facility in Figure 3. LN<sub>2</sub> was routed from the cryogenic container by operating a control valve, through a l-inch tube system and was discharged into the nacelle through either four fog nozzles or open-end tube systems. A bottom view of the test engine installation showing the fog-nozzle location is pictured in Figure 4.

A diagrammatic view of the dewar is shown in Figure 5. The dewar was rated at 100-pound liquid nitrogen capacity plus an approximate 1/3-cubic-foot vapor space. During some tests in the program, a portion of the vapor space was filled with liquid, thus accounting for dewar fill ratios reported in excess of 100 percent. The locations of the dewar valving, gauges, discharge temperature probe, and liquid withdrawal tube are also shown. A schematic drawing showing the distribution systems and the associated flow control orifices and instrumentation pickups is illustrated in Figure 6.

The nitrogen was stored under pressure in the dewar as a saturated liquid. All the test fires resulted from spray releasing and spark igniting JP-4 fuel. A surveillance-type, radiation sensing, flame detector was installed within the nacelle in the vicinity of the test fire fuel spray nozzle. The fuel spray nozzle and flame detector are shown in Figure 7. The detector output signal was recorded by an oscillograph to indicate ignition and extinguishment times for the test fires. The test fires were located in a remote area relative to the LN2 discharge location to avoid the effects of localized high concentration of nitrogen in the area of the fire.

Mockup Engine/Nacelle Facility: The second facility, shown in Figure 8, is a boiler plate mockup of an engine nacelle. Outside air is drawn into the tunnel circuit by an axial-flow fan and fed through a perforated plate into the test section. The air flows through the annular passage formed by an elliptically domed cylinder positioned within a larger cylinder to simulate a cowled engine. The airflow through this annulus is made turbulent by ribs installed alternately on the outer and inner cylinders. The air exits the test section through a perforated ring into the exhaust section of the tunnel. The volume of the test section can be varied by positioning the perforated ring fore or aft on

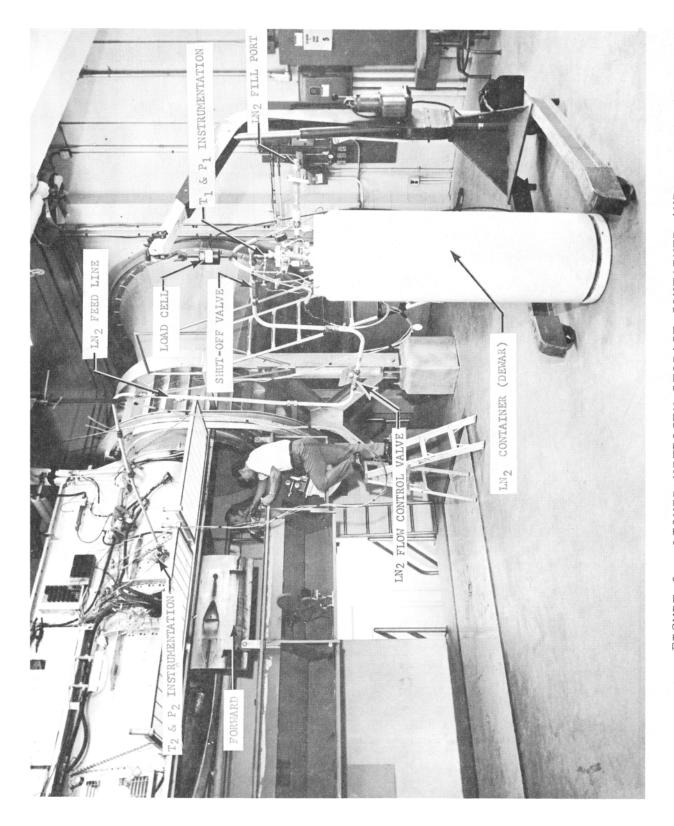


FIGURE 3 LIQUID NITROGEN STORAGE CONTAINER AND TYPICAL DISTRIBUTION SYSTEM

FIGURE 4 BOTTOM VIEW OF POWERPLANT AND NACELLE SHOWING FOG NOZZLE DISCHARGE SYSTEM

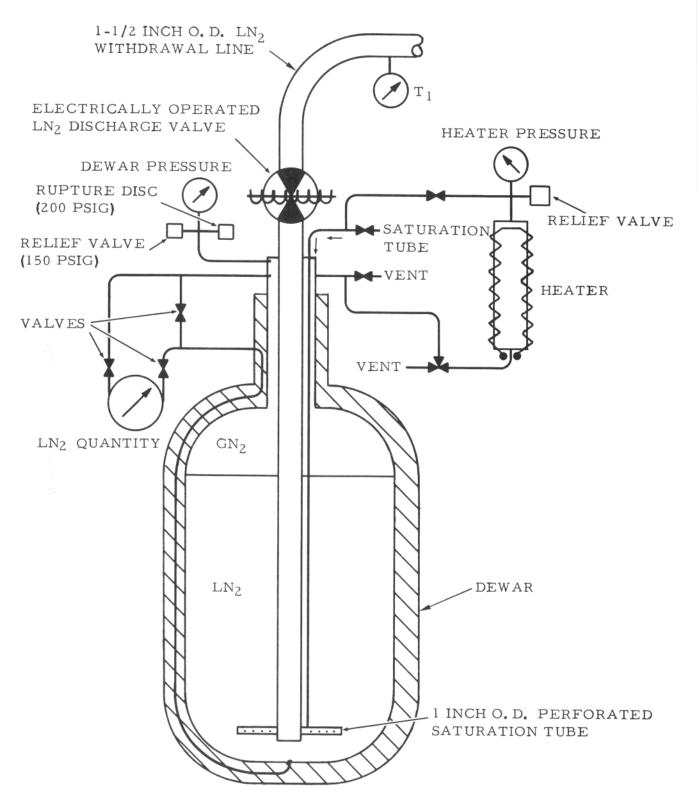
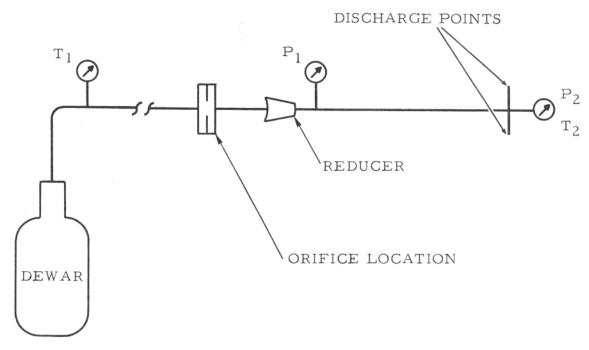
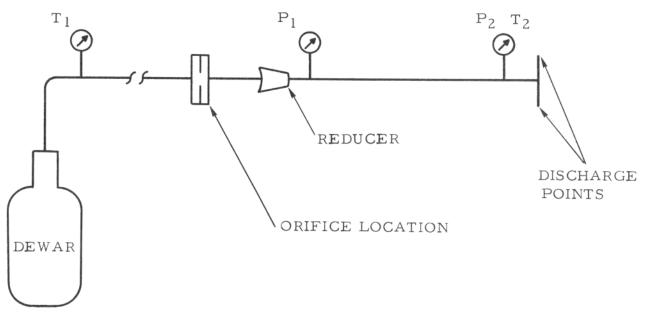


FIGURE 5 DIAGRAMMATIC VIEW OF DEWAR ASSEMBLY AND INSTRUMENTATION

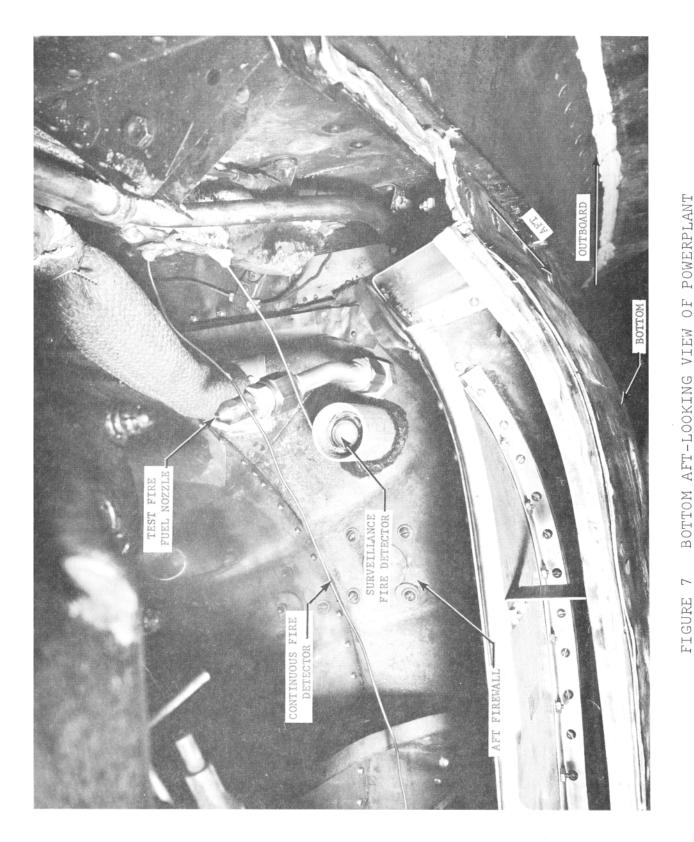


(a)  $LN_2$  DISTRIBUTION SYSTEM 1



(b)  $LN_2$  DISTRIBUTION SYSTEM 2

FIGURE 6 LN2 DISTRIBUTION SYSTEMS AND INSTRUMENTATION SCHEMATIC



JRE 7 BOTTOM AFT-LOOKING VIEW OF POWERPLANT NACELLE SHOWING FUEL-TO-FIRE NOZZLE AND FIRE SENSOR

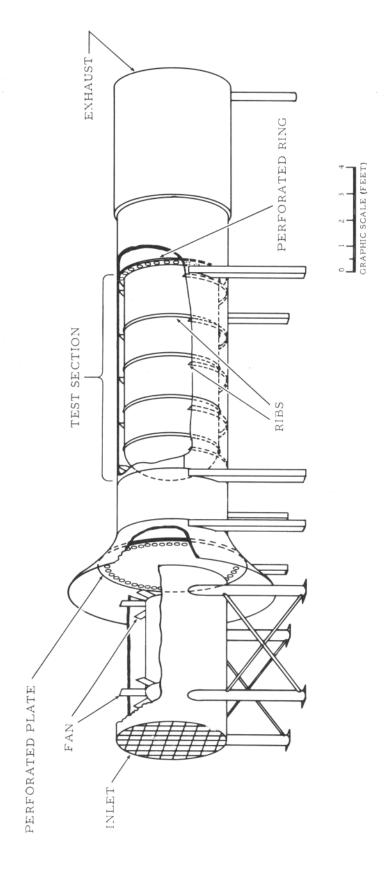


FIGURE 8 MOCKUP ENGINE/NACELLE FACILITY

the inner cylinder. The LN2 distribution systems used in both facilities were similar. LN2 was discharged into the second facility through either open-end tube systems or a perforated tube system. In both facilities, no attempt was made to optimize the type of discharge and the distribution within the test compartments. Test fire ignition and extinguishment were also monitored by a radiation-type flame detector in this facility.

#### Effect of System Pressure Losses and LN2 Flashing

Objective: An objective of this work was to experimentally define the effects of pressure losses and the associated rapid conversion from the liquid state to the gaseous state (flashing) in the LN2 distribution systems on (1) the nitrogen quantity requirements for extinguishing fires and (2) the size of the distribution system. Since the two-phase flow problem encountered in a transfer system being supplied with a saturated liquid is extremely complex, no attempt was made to establish design information and procedures related to predicting the quality (x), quantity, and cool-down time for any given distribution system. The investigation was limited to determining whether the amount of flashing and cooling that occurs in such a system has a significant effect on (1) the quantity requirements for extinguishing a fire, (2) the discharge rate through a given size system, and (3) the time required to extinguish the fire (system response).

As a saturated liquid flows through an uninsulated tube, a portion of the liquid is converted to vapor. The amount of vapor produced is a function of (1) the pressure losses in the tube which lower the local static pressure below the existing vapor pressure and produce flashing, and (2) the amount of heat transfer through the wall as the tube is cooled. This section of the report deals primarily with the flashing and system response effects. The cool-down phenomena associated with long-line lengths will be discussed in a following section of the report.

Method: The amount of flashing was controlled by inserting various sizes of orifice plates in the distribution system at the dewar outlet. The nitrogen flow rate was controlled by varying the size of the nozzles at the outlets. The distribution systems used for this investigation and the location of the orifice plates are shown in Figure 6. Nitrogen was saturated at approximately 100 pounds per square inch gage (psig), plumbed through 21 feet of either 1/2-, 3/4-, or 1-inch tubing, and discharged through a standard AN-834 bulkhead tee fitting. The size of the tee fitting corresponded to the size of tubing being used.

Changes in discharge nozzle size were accomplished by attaching either AN-894 reducer bushings or drilled AN-820 caps to the AN-834 tee fitting. Initial testing involved calibration discharges into an open laboratory area to determine nitrogen flow rates as a function of nozzle, orifice, and tube size. The results of these calibration tests are presented in Figures 4-1, 4-2, and 4-3 of Appendix D. After the flow calibration was complete, fire tests were conducted in the engine installation with the discharge tee and nozzles positioned in the forward section of Zone II at 3:30 o'clock as shown in Figure 5-1 of Appendix E. The discharge was directed annularly over and under the engine case.

The flow calibration tests consisted of short duration discharges of LN2 during which time, mass flow rates, and distribution system temperatures and pressures were measured. The test procedure consisted of spark igniting the fuel spray at nozzle location B, shown in Figure 5-1 of Appendix E, and a 10-second duration discharge from the l-inch-diameter LN2 system. The bleed airflow was maintained between 1.7 and 1.9 pounds per second. The surveillance-type, radiation sensing, flame detector was utilized to determine whether the fire was extinguished and the time of extinguishment. The minimum LN2 flow rate required for extinguishment was determined for controlled amounts of flashing up to 19 percent, on a weight basis, at the discharge tee.

Results: The results of the tests in this series (Nos. 1 through 40) are summarized in Table 2, Figure 9, and Appendix D, Figures 4-1 through 4-3. The liquid lost due to the flashing of liquid nitrogen, shown in Table 2 and Figure 9, was calculated from a temperature-entropy diagram prepared by the National Bureau of Standards. The pressure readings given in Table 2 are for a period three seconds after LN2 discharge, which was the nominal time required for flow stabilization to occur.

A comparison of the nitrogen discharge rates at the various degrees of flashing shown in Table 2, indicates that the pressure losses and flashing of LN2 in the distribution system did not substantially affect the discharge rate requirements for extinguishing the fire. At nitrogen discharge rates above 1.05 pounds per second, all the fires were extinguished regardless of the amount of flashing. Conversely, at rates less than 1.05 pounds per second, none of the test fires were extinguished. As would be expected, the amount of flashing did substantially affect the nitrogen discharge rate. As the percent lost by flashing increased from two percent to 16 1/2 percent, the nozzle size required to allow sufficient flow for extinguishment increased from 0.24 to 1.12 square inches.

Figure 9 shows the calculated mass flow of nitrogen per unit nozzle area as affected by the amount of flashing that occurs within the distribution system. This figure shows the least squares fit in the form of an exponential function for the combined data from the tests with 1/2-, 3/4-, and inch tube systems.

The system response was not substantially affected by the amount of flashing. All the fires were extinguished in a 3-to-7 second period after initiating the LN $_2$  system discharge regardless of the quality of the nitrogen.

TABLE 2. - EFFECT OF PRESSURE LOSSES AND FLASHING ON LN2 SYSTEM REQUIREMENTS

Test		Liquid Lost by Flashing	Pl (at 3 sec) (after discharge)	Pn (at 3 sec) (after discharge)	w <sub>LN<sub>2</sub></sub>	Time Fire Ext.
	(in2)	(%)	(psig)	(psig)	(lb/sec)	(sec)
31	0.138	1.5	103.0	101.5	1.02	Non-Ext
32	0.240	2.5	100.0	94.0	1.76	4.3
33	0.186	2	87.5	83.3	1.26	6.1
34	0.368	11.5	54.5	45.0	1.09	6.6
35	0.138	7.5	57.0	56.5	0.51	Non-Ext
36	0.240	6	70.5	61.5	1.06	6.7
3 7	0.240	7.	59.5	55.8	0.91	Non-Ext
38	1.118	16.5	45.3	21.5	1.16	3.4
39	0.582	15.5	37.5	25.0	0.91	Non-Ext
40	1.118	18.5	31.5	13.5	0.91	Non-Ext

### Effect of Fire Size

Objective: The objective of this effort was to determine the effect of the amount of fuel on the fire size and the  $\text{LN}_2$  requirements. The basic concept used throughout the portion of the investigation, in which  $\text{LN}_2$  was used to extinguish fires on the test engine installation, was to create a large, severe

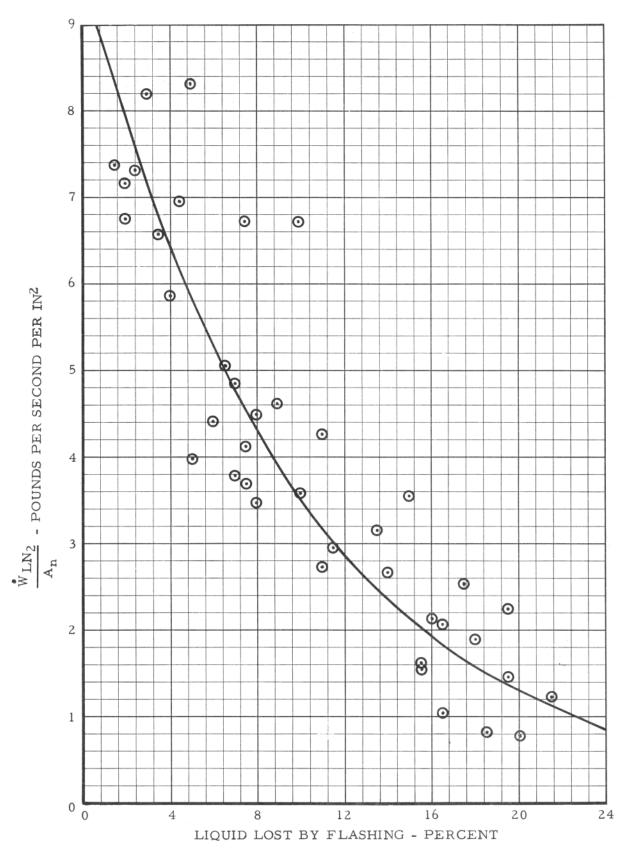


FIGURE 9 LIQUID NITROGEN DISCHARGE RATE AS A FUNCTION OF FLASHING IN THE DISTRIBUTION SYSTEM

fire requiring maximum quantity of LN2 without causing extensive damage to the nacelle. This was accomplished by minimizing the duration of the fire and increasing the flow of fuel to the fire until the LN2 requirements for extinguishment no longer increased.

Method: The investigation to determine the effect of the amount of fuel on the fire size and the LN<sub>2</sub> requirements was conducted in Zone II of the test engine installation with a nominal bleed airflow of 2.5 pounds per second. The nitrogen was saturated at 100 psig, plumbed through 21 feet of l-inch tubing, and discharged through a standard AN-824 bulkhead tee fitting. The tee was positioned in the forward section of the Zone at 3:30 o'clock to direct the discharge annularly over and under the engine case.

The test procedure consisted of spark igniting the fuel spray at nozzle location B, with the engine operating at military rated thrust (MRT). The engine was retarded to cutoff 5 seconds after initiating the fuel release. This was followed 10 seconds later with a 10-second duration discharge from the LN2 system. A radiation sensor was utilized to determine whether the fire was extinguished and the time of extinguishment. The minimum LN2 flow rate required for extinguishment was determined for fuel flows ranging from 0.1 to 0.7 gallon per minute (gpm).

Results: The results of the eight tests in this series are summarized in Table 3.

The time listed for each test in which the fire was extinguished is the period between initiating the LN2 system discharge and the clearing of the radiation sensor. The fuel flow rate is seen to have affected the discharge rate of nitrogen required for extinguishing the test fires. As the fuel flow was increased from 0.1 to 0.3 gpm, the nitrogen discharge rate requirements increased, and unburned fuel started to accumulate within the compartment. As the fuel flow was increased beyond this point, burning occurred outside the compartment at a location downstream of the top air exit louvers, and the required nitrogen discharge rate no longer increased. As a result of this test series, all remaining fire testing with bleed air flowing into Zone II of the test nacelle at rates above 1.5 pounds per second were normally conducted with a fuel-to-fire flow of 0.3 gpm. Similarly, because of unburned fuel accumulations and external fire with 0.3 gpm fuel flows at bleed airflows below 1.5 pounds per second, remaining tests in this airflow range were normally conducted with 0.1-gpm fuel flows.

TABLE 3. - EFFECT OF FUEL-TO-FIRE FLOW RATE ON LN2 FLOW RATE REQUIREMENTS

Test No.	Fuel Flow WF	Liquid Nitrogen Flow W <sub>LN2</sub>	Time Fire Extinct
	(gpm)	(lb/sec)	(sec)
41	0.1	1.45	4.2
42	0.1	1.22	6.0
43	0.1	0.91	Non-Ext
44	0.3	1.15	Non-Ext
45	0.3	1.33 (Ext at 4.8,	Non-Ext Flashed back at 12.8)
46	0.3	1.38	3.6
47	0.5	1.33	5.8
48	0.7	1.42	3.5

# A Comparison of Gaseous Nitrogen and Liquid Nitrogen

Objective: The specified objective of this phase of the project was to determine the effectiveness of  $\mathrm{GN}_2$  as compared to  $\mathrm{LN}_2$  in extinguishing fires. However, an extensive failure of the test equipment caused the cancellation of this designated objective and no data were obtained.

Alternate Objective: The alternate objective of this phase was to study the effects of long distribution lines and discharge valve location on the LN2 requirements. The tests were conducted in the Five-Foot Fire Test Facility using the JT-12 installation as the test article. Standard turbine engine and wind-tunnel instrumentation were utilized to record the JT-12 and tunnel facility operational parameters during the tests.

Method: A long line distribution system was fabricated from 1-inch-outside diameter tubing with a 0.040-inch wall thickness. A line length of 80 feet was selected to approximate the tubing required between a dewar mounted in the center of the fuselage and the outboard nacelle of a typical large transport aircraft. Open-end tee discharge nozzles were used for the tests. In addition to the electrically operated discharge valve, located as shown in Figure 5, a manually operated ball valve was located in the system approximately 5 feet from the nacelle discharge nozzle. The distribution system configuration and instrumentation, with the exception of the manual ball valve, are shown in Figure 6 as "LN2 Distribution System 2."

A total of 8 tests was conducted with the 80-foot lines. For seven of these tests, the electrically operated valve at the dewar was opened, and the line from the dewar to the closed manually operated valve was filled with GN2, at the dewar saturation pressure, approximately 5 minutes before discharge. The manual valve was then used to discharge the agent for the test. For the eighth test, the manual valve was placed in the open position, and the discharge was controlled by the electrical valve at the dewar, thus leaving the long line unpressurized.

For all tests in this series, Test Event Schedule E was utilized, as described in Appendix C. At the time of LN2 discharge, the engine power level was at cutoff and the compressor interstage bleed ports were open. The test section Mach number was stabilized at 0.50, and the fuel flow to the fire was 0.30 gpm. The nominal secondary airflow within the nacelle was 3 pounds per second.

Results: A tabular record of Tests Nos. 241 through 252 inclusive, is presented in Appendix B. A comparative time versus event illustration of several of the pressurized 80-foot lines with the discharge valve near the discharge point, the unpressurized 80-foot line with the discharge valve at the dewar, and a typical unpressurized 21-foot line with the discharge valve at the dewar is presented in Figure 10. Comparison of the pressurized 80-foot lines with the valve near the discharge point and the unpressurized 80-foot line with the valve at the dewar shows little significant difference in the time from "LN2 ON" to "FIRE OUT." Comparing Tests Nos. 247, 249, and 250, which were all pressurized 80-foot lines with the valve near the discharge point, indicates that as the time from "IGNITION" to "LN2 ON" (preburn time) increased, the time from "LN2 ON" to FIRE OUT" (extinguishment time) also increased.

An event versus time schedule is also shown in Figure 10 for a typical 21-foot system having the discharge valve at the dewar, and test conditions similar to those for 80-foot lines. The time for extinguishment with the typical 21-foot line having a test fire preburn time of 14.2 seconds, was approximately one-third that of the 80-foot line having a test fire preburn time of 14.3 seconds (Test No. 250). The flow rate buildup to the nominal rate existing 2.5 seconds before and after "FIRE OUT" was also greater for the 21-foot line.

Figure 11 illustrates the time history of line pressures, temperatures, and dewar weight for the pressurized and unpressurized 80-foot lines with the discharge valves in different locations. For comparison, a similar history is presented for a similar unpressurized 21-foot length configuration with the valve at the dewar. This plot also indicates that the discharge and flow parameters are essentially the same for the pressurized and unpressurized 80-foot line configurations. A tabular presentation of time versus line pressures, temperatures, and dewar weight for selected runs in the 80-foot line length series of tests, and two comparable 21-foot line length tests is presented in Appendix F.

It should be noted that some difficulty was encounterd with the pressurized 80-foot line having the discharge valve near the discharge point. Possibly due to the combination of long-line length and support arrangement, system oscillation and vibration, and extreme temperature changes causing rapid contraction and expansion of system fittings, a fitting loosened during the testing causing the charged line to leak. Eventually pressure and LN2 loss from the dewar would have occurred. This system did have a relatively large number of fittings; however, the number could be considered representative of the number found on a large aircraft with a similar line length. Hence, in service, the system with the discharge valve located remotely from the dewar would have the greater potential for possible system leaks.

The oscillograph records showed that, as the discharge tee outlets in the 1-inch 80-foot length lines were decreased from AN-834-16 fittings to -12 to -8 fittings, large sinusoidal pressure oscillations with a magnitude of 15-20 psi were recorded at the P<sub>1</sub> and P<sub>2</sub> probes. The smaller the outlet fitting, the greater was the duration and magnitude of the oscillations. The oscillations decreased in magnitude and frequency until they were dampened out after 13 seconds of the 20-second discharge. Little effect was noted in discharge rate due to these oscillations.

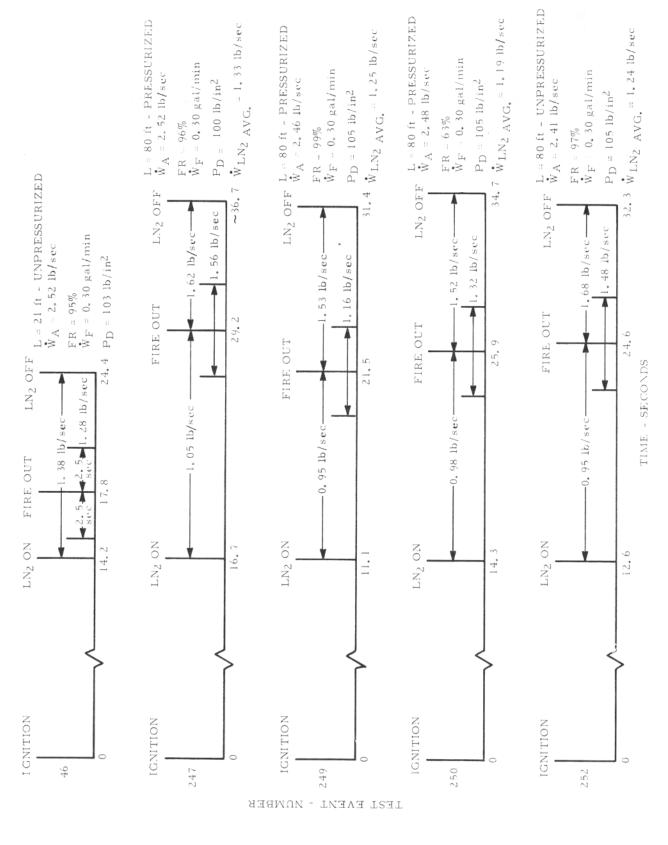


FIGURE 10 - HISTORY OF TEST EVENTS FOR 80-FOOT-LONG DISTRIBUTION LINES AND A COMPARABLE 21-FOOT-LONG DISTRIBUTION LINE

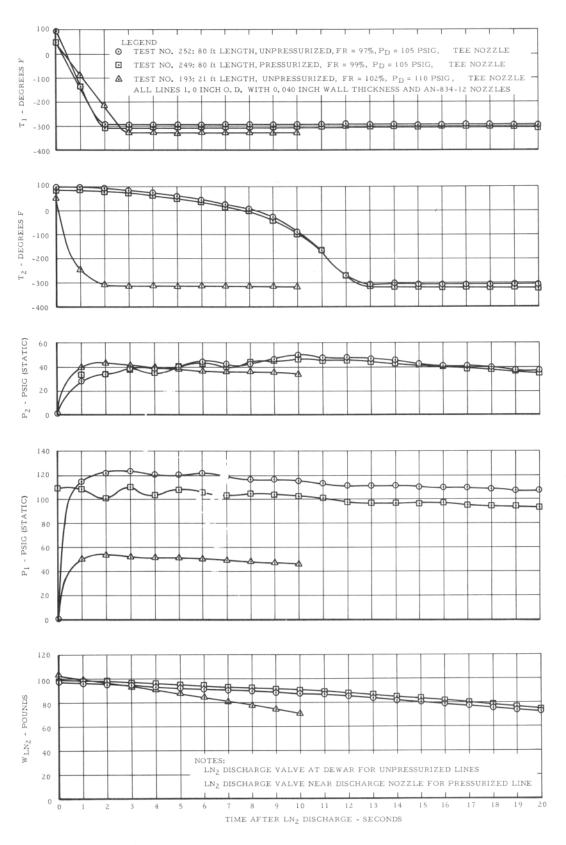


FIGURE 11 - DISTRIBUTION SYSTEM PRESSURES, TEMPERATURES, AND NITROGEN FLOW FOR AN 80-FOOT PRESSURIZED LINE AND 80- AND 21-FOOT UNPRESSURIZED LINES

#### Effect of Compartment Airflow

Objective: A large portion of this work was devoted to experimentally defining the nitrogen requirements for extinguishing fires as a function of compartmental airflow. This relationship was explored during the first phase of testing as reported in Appendix A. Testing under this initial phase involved spray releasing and spark igniting JP-4 jet fuel at a rate of 0.1 gpm. These tests were conducted with nitrogen being throttled to control the discharge rate. This resulted in large quantities of the nitrogen being converted to a gas in the distribution line. It had been theorized that the required discharge rates would be somewhat lowered if the nitrogen in the line was maintained in the liquid state (Appendix 1, Reference 1). The test fires at the higher airflow were considered to have been burning lean, and it was expected that at higher fuel release rates, the required nitrogen discharge rates Therefore, additional tests were scheduled would increase. during the second phase of testing to supplement the initial data by increasing both the fuel flow and the airflow, and by maintaining liquid flow in the discharge line during tests with low nitrogen flow rates. At the same time, tests were also scheduled to investigate the effects of the type of discharge and the compartmental volume on the relation between nitrogen fire-extinguishing requirements and compartmental airflow.

Method: This phase of the investigation involved fire tests in the JT-12 installation and the 40- and 53-cubic-foot simulated nacelle installation. The 21-foot 1-inch tube nitrogen distribution system was utilized with discharge through either the AN-834 nozzles or the perforated tube.

The test procedure consisted of spark igniting the fuel spray at nozzle location B in the JT-12 installation and a comparable location in the simulated nacelle. In the JT-12 installation, the engine power was retarded from military rated thrust (MRT) to cutoff 5 seconds after initiating the fuel release, and 10 seconds prior to discharging the nitrogen. The fan power and airflow for the simulated nacelle were maintained constant throughout each test run. The duration of the discharge from the LN2 extinguisher system was 10 seconds for all the tests in this series. As in previous tests, a surveillance-type radiation sensor was utilized to determine the time of extinguishment. The minimum LN2 flow rate required for extinguishing the test fires was determined at various airflow rates up to a maximum rate of 9 pounds per second.

Results: The results of the tests in this series are summarized in Figures 12 and 13. The airflow rate, shown in Figures 12 and 13, is seen to have substantially affected the discharge rate requirements of nitrogen for extinguishing the fires in each of the three different volume compartments and with the two types of discharges. The required nitrogen flow rate increased in direct proportion to the JT-12 compartment bleed airflow. Figure 13 shows the least squares fit in the form of an exponential function for the combined data from the tests with 40- and 53-cubic-foot volume configurations of the simulated nacelle using both open-tube nozzle and perforated tube-type discharges. Variations in the volume size and type of discharge did not affect the nitrogen flow rate requirements substantially from the least squares curve. The differences between the linear relationship, shown in Figure 12, and the exponential relationship, shown in Figure 13, are attributed to the non-uniform airflows and possible inadequate nitrogen distribution in the simulated nacelle at the high airflow rates.

#### Effect of LN2-Induced Cooling

Objective: An objective of this work was to experimentally determine the effectiveness of a nitrogen fire-extinguishing system in post-fire cooling of the compartment and potential reignition sources. As determined in a previous investigation, long-duration fires may heat small exposed metal components of the engine and nacelle sufficiently to reignite the fuel after the extinguishing agent dissipates (Appendix I, Reference These reignitions were further reported to be explosive when the quantity of extinguishing agent was marginal to the extent that a long-duration fire would be only momentarily extinguished. Therefore, an investigation was made to determine the feasibility of increasing the discharge rate and prolonging the duration of the discharge of a nitrogen fire-extinguishing system to force-cool components below the ignition temperatures of the flammable fluids present, and to keep the compartment inert while this cooling is taking place.

Method: The test engine installation was instrumented with thermocouples to measure ambient and metal temperatures within the accessory compartment. The metal components selected as being typical low-thermal mass items, found on powerplant installations, consisted of a continuous-type fire

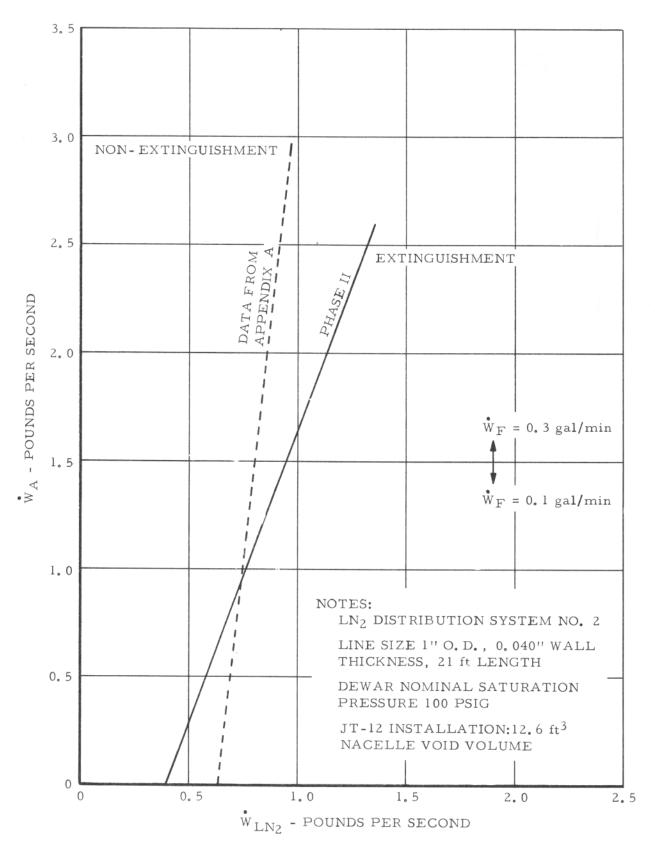


FIGURE 12 LN<sub>2</sub> FLOW REQUIREMENTS AS A FUNCTION OF COMPARTMENT AIRFLOW IN THE JET ENGINE INSTALLATION

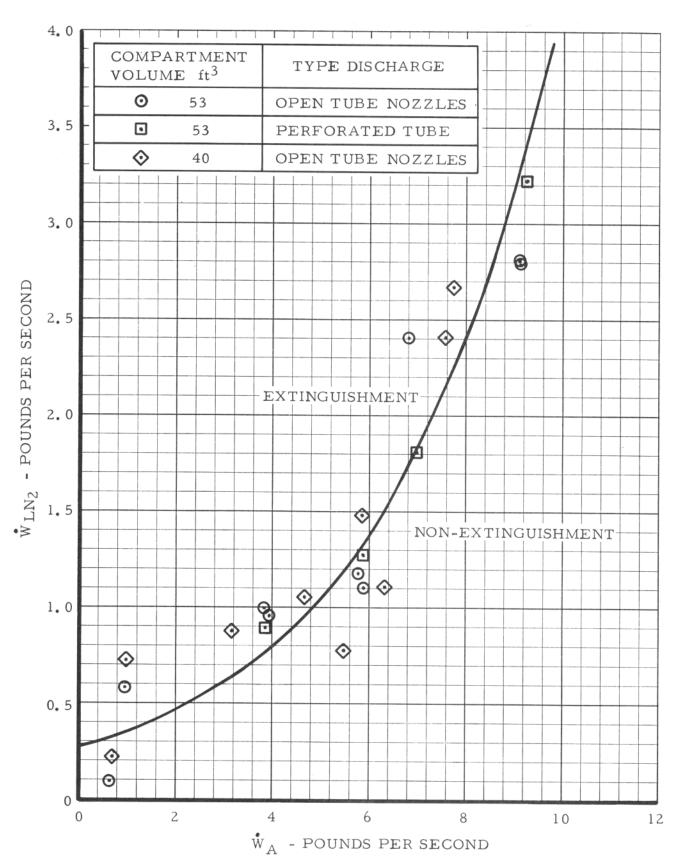


FIGURE 13 LN<sub>2</sub> FLOW REQUIREMENTS AS A FUNCTION OF COMPARTMENT AIRFLOW IN THE MOCKUP ENGINE/NACELLE FACILITY

detector element; 0.041-inch-diameter, twisted, stainless steel, safety wires; and a door-latch bracket. These components were remotely located relative to the nitrogen outlets and were in the immediate vicinity of the fire. The nitrogen distribution system utilized for this series of tests consisted of a 21-foot length of l-inch tubing with two open-tube discharge nozzles.

The test procedures followed consisted of spark igniting the fuel spray at nozzle locations B, B\*, or C, as shown in Figure 5-1 of Appendix E. The engine was retarded from MRT to cutoff 5 seconds after initiating the fuel release. For tests involving a nitrogen discharge, this was followed 10 seconds later with a discharge from this nitrogen system of 9- to 14-seconds duration. In order to determine the degree of cooling resulting from the air entering the compartment and from the nitrogen, tests involving metal temperature measurements were duplicated with and without nitrogen discharges. The fuel flow was reduced to 0.03 gpm for these tests so the fire would self-extinguish as the fuel was shut off. At higher fuel flows, the fire would relocate away from the instrumented components and continue to burn after fuel shutoff.

Results: As shown in Figure 14, significant cooling was apparent during the nitrogen discharges. The measured ambient temperature resulted from a 1-pound-per-second discharge of nitrogen, initiated at zero time, into the fire environment of the test engine installation. The rapid decay in the ambient temperature is typical of the effect that the extinguishment of a fire with a nitrogen discharge has on the ambient environment in an area of a compartment remote to the location of the discharge nozzles. As would be expected, the temperatures in the area of the nitrogen outlet decreased at a higher rate, and to a lower level, during the discharge.

The metal components were likewise substantially cooled by the nitrogen. As shown in Figure 15, the twisted safety wire was heated to temperatures from 1400° to 1640°F during three test fires. In the first test, the fuel-to-fire was shut off, and the wire cooled from 1400° to 500°F in 13 seconds. The 500°F temperature is considered to be a relatively safe temperature from the standpoint of ignition

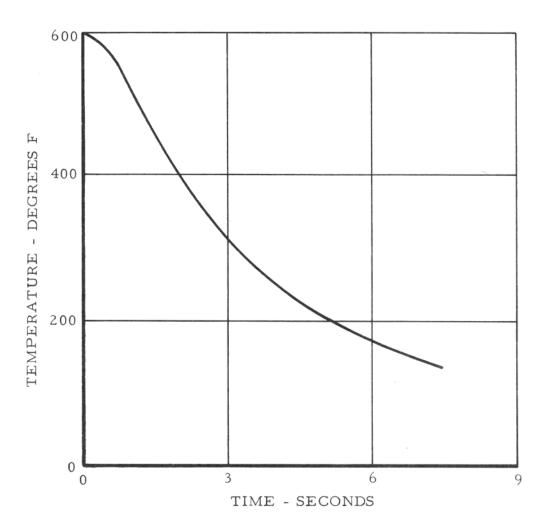


FIGURE 14 AMBIENT TEMPERATURE IN JET ENGINE INSTALLATION DURING NITROGEN DISCHARGE

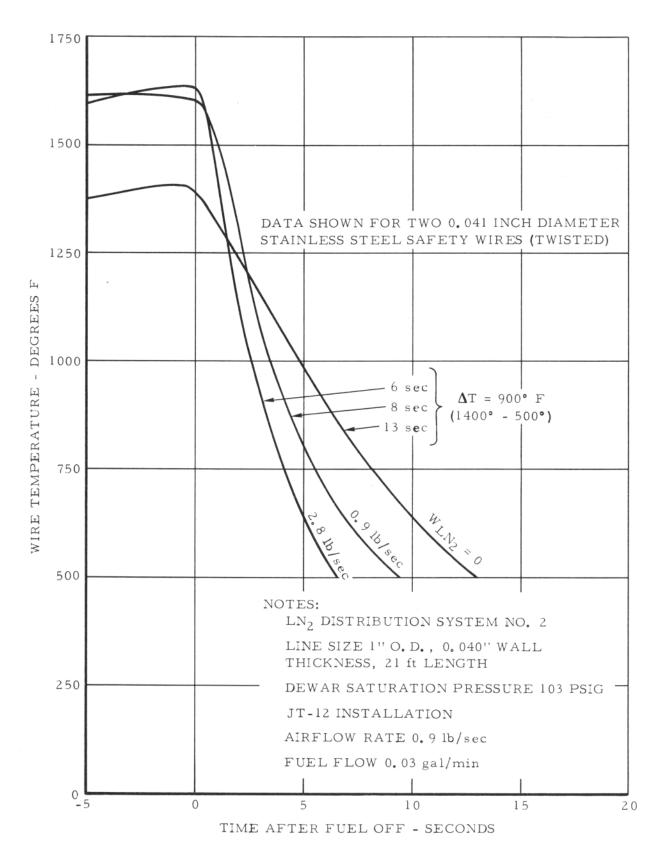


FIGURE 15 TEMPERATURE DECAY OF TWISTED SAFETY WIRE WITH AND WITHOUT NITROGEN DISCHARGE

of fuel vapors. In the next two tests, the nitrogen was discharged at 0.9 and 2.8 pounds per second. The times required for the wire temperature to decrease from 1600° to 500°F were 9.4 and 6.5 seconds for the low and high nitrogen discharge rates, respectively. In addition to the possible elimination of potential reignition sources prior to the end of the discharge, this cooling is considered to be beneficial in decreasing the rate at which remaining fuel in the compartment is vaporized. An item which makes the nitrogen cooling effect more significant is the fact that the fire-extinguishing systems in use on current aircraft may dissipate the agent in a half second after reaching the concentration required to extinguish a fire. With normal cooling, the temperature of the safety wires a half second after the fire was selfextinguished was lowered 80° to 1320°F. In the case of the nitrogen extinguishing system, it is possible that with the availability of large quantities of nitrogen for fuel tank inerting, the discharge duration could be prolonged for 30 seconds or longer to assure that all hot surface reignition sources are eliminated, and to allow for the dissipation of fuel vapors.

# Effect of Inadvertent LN2 Discharge

Objective: An operating turbojet engine was selected for evaluating the effectiveness of cryogenic nitrogen as an agent for aircraft powerplant fire-extinguishing systems in order that the thermal effects on the engine and nacelle components could be observed. In addition to the effects of a discharge with the engine shut down according to fire emergency procedures, there was concern about the thermal effects of an inadvertent nitrogen discharge while the engine was operating.

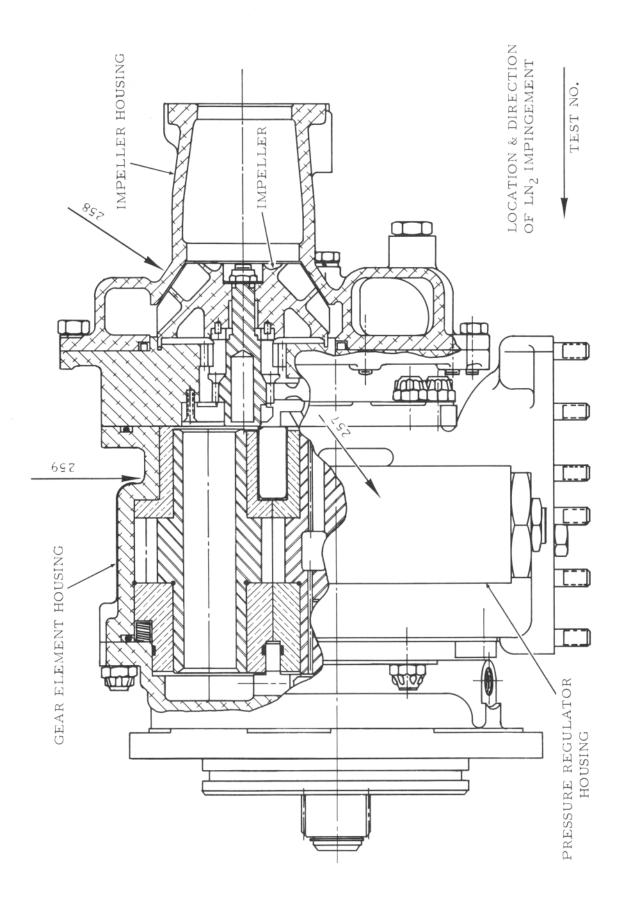
Method: Special tests were not performed to evaluate the thermal effects of extinguishing nacelle fires with a cryogenic nitrogen system. This information was obtained during tests on the JT-12 installation, which were designed for other specific program objectives. The effects of an inadvertent discharge were limited to tests in which an operating engine fuel pump was thermally shocked by large quantities of nitrogen. A single-gear-type fuel pump with a centrifugal booster from a JT-4 engine was mounted in a closed cubical compartment 20 inches in length on each side. The pump was connected to a 20-horsepower electric motor located outside the compartment. The primary materials used in the pump are nitralloy steel for the gears and shafts, lead-bronze for the bearings, and 355T6 aluminum for the housings.

The pump was operated at 3450 rpm with a discharge pressure of 470 psig and a delivery rate of 36 gpm. The nitrogen was distributed through 21 feet of 1-inch tubing and discharged through an open-end tube into the compartment. Four positions were selected for the discharge tube, with three of these positions selected so the nitrogen would impinge directly on critical areas of the pump. The first position directed the nitrogen onto the floor of the compartment so that the nitrogen did not directly impinge on the pump. The three remaining positions were directed to impinge the nitrogen on the pressure regulator housing, the booster element housing, and the spur-gear type element housing, as shown in Figure 16. The pump was instrumented with surface thermocouples at each of the three impingement locations on the pump. The compartment ambient temperature, pump discharge temperature, and fuel flow and pressure, were also measured and recorded. The current flow to the drive motor was measured for an indication of any pump seizure during the nitrogen discharge.

The procedure for these tests consisted of operating the pump until the outlet conditions stabilized. The nitrogen was then discharged from a full dewar, saturated at 100 psig, into the compartment with the pump operating. The discharges lasted from 40 to 46 seconds and expended from 60 to 76 pounds of nitrogen at rates from 1.4 to 1.7 pounds per second.

Results: The results of the four fuel pump tests are summarized in Table 4. In each test, the pump continued to operate without any apparent adverse effects. The fuel flow and discharge temperature did not measurably change during the tests. The pump outlet pressure showed only minor fluctuations of 10 to 20 psig. The amperage measurements showed a gradual increase of from 1 to 2 amperes during the first three tests and of 6 amperes during the last test with the nitrogen impinging on the gear housing. During the first and second tests, a short duration (less than 1 second) current surge of 2 amperes was recorded, respectively, at 38 and 31 seconds into the nitrogen discharges. In all four tests, the current returned to a normal level shortly after the nitrogen discharge was terminated, and the pump continued to operate satisfactorily.

The JT-12 engine installation was subjected to nearly 100 nitrogen discharges in extinguishing fires during the two phases of this program. As much as 50 pounds of nitrogen were discharged into the accessory compartment during a single test at rates as high as 3.2 pounds per second, and for durations from 3 to 20 seconds. There were no observed failures or deteriorations of the engine and nacelle components throughout the program which could be directly attributed to the thermal effects of the cryogenic nitrogen fire extinguisher-system.



LOCATION OF LN2 IMPINGEMENT ON TYPICAL TURBINE AIRCRAFT FUEL PUMP FIGURE 16

TABLE 4. - RESULTS OF LN2 DISCHARGE ON TYPICAL TURBINE AIRCRAFT FUEL PUMP

Time (sec)	Tamb (°F)	T <sub>case</sub> (°F)	Drive Motor Current (amp)	W <sub>LN2</sub>	
	TEST 256	(Compan	etment Floor)		
0 10 20 30 38.3 40 45.5	84 -170 -279 -317  -320 END of	79 42 38 30  20 LN <sub>2</sub>	49.3 50.2 50.8 51.0 53.0 51.1 DISCHARGE	0 16.8 32.8 50.4  66.8 75.7	
	TEST 257	(Pressu	ure Regulator)		
0 10 20 30 30.1 40	75 36 -270 -293  -304 END of	96 NR* NR NR NR NR NR	50.3 50.5 50.6 51.2 53.3 50.8 DISCHARGE	0 13.2 27.0 41.0  54.8	
	TEST 258	(Impelle	er Housing)		
0 10 20 30 40 44.2	65 -188 -288 -315 -320 END of	64 43 41 42 12 LN <sub>2</sub>	49.5 50.2 51.0 51.1 51.5 DISCHARGE	0 14.6 30.6 46.6 62.4	
	TEST 259	(Gear Ho	ousing)		
0 10 20 30 39.8 39.83	NR NR NR NR NR END of	+79 NA** NA NA NA NA LN <sub>2</sub>	49.7 54.5 55.7 54.2 54.6 DISCHARGE	0 15.0 29.6 45.4 60.0	

<sup>\*</sup> NR - Not Recorded
\*\* NA - Not Applicable

# Effect of Damaged Cowling

Objective: The objective of this effort was to determine the effectiveness of an LN2 system in extinguishing fires after the powerplant installation had been damaged by a fire, and the compartment was no longer intact. The criteria used in the design and evaluation of conventional powerplant fireextinguishing systems is that the protected compartment will be intact at the time the system is utilized. A previous FAA project effort indicated that a fire which is not rapidly detected and extinguished may produce abnormally high air leakage into the compartment or may create openings at seams, seals, or near normal air exits (Appendix I, Reference 2). Therefore, an investigation was conducted to determine the effects of high localized inflows of air and large openings in the cowling on the requirements of an LN2 fire-extinguishing system. The investigation would provide information necessary to determine the feasibility of furnishing addtional protection, in the event of such a failure, by providing additional quantities of  ${
m LN}_2$  from the normal supply expected to be available on large aircraft.

Method: The fire damage to the nacelle was simulated by two separate methods: (1) the starter/generator cooling air duct in the accessory compartment of the JT-12 installation was disconnected to allow ram air into the compartment through the 3-inch-diameter duct, and (2) a 3.9- by 3.5-inch rectangular, static-type opening was made in the cowl door at station 90 at 5:30 o'clock, where the external nacelle pressure was equal to a static pressure corresponding to the tunnel's pressure altitude. The 21-foot-long, 1-inch-tube nitrogen distribution system was utilized with the two opentube discharge nozzles to extinguish the fires in the damaged nacelle.

The test procedures followed consisted of spark igniting the fuel spray at nozzle location B. The engine power was retarded from MRT to cutoff 5 seconds after initiating the fuel release and 10 seconds prior to discharging the nitrogen. The duration of the discharge from the LN2 system was 10 seconds. The minimum LN2 flow rates for extinguishing the fires were determined for a normal nacelle configuration, a damaged nacelle with approximately 3/4 pound per second of air entering the compartment from the disconnected cooling air duct, and a damaged nacelle simulating a burned-out area exposed only to static-pressure differentials. The bleed airflow was maintained at approximately 2 1/2 pounds per second for all three nacelle configurations. The fuel-to-fire flow was increased from 0.3 to 0.5 gpm for the tests with the disconnected cooling air duct in an attempt to

create a condition where the fire was not burning lean and, as previously discussed, the fire size was of sufficient intensity to require a maximum amount of nitrogen for extinguishment.

Results: Table 5 summarizes the results of the tests in this series. The localized high air leakage from the 3-inch duct entered a 9 o'clock position in the forward section of the compartment with a downward directed flow. The air discharged from this duct was approximately 30 percent, by weight, of the air entering the compartment from the bleed air system. This 30-percent increase in airflow resulted in an approximate 14-percent, by weight, increase in the nitrogen flow rate requirements when compared to the undamaged nacelle nitrogen fire extinguishing requirements.

TABLE 5. - EFFECT OF SIMULATED DAMAGED COWLING ON NITROGEN FLOW REQUIREMENTS

Cest No	Bleed Air Flow	Ram Air Flow	Total Air Flow	$\dot{\mathbf{w}}_{\mathrm{LN}_2}$	Time Fire Extinct	
	(lb/sec)	(lb/sec)	(lb/sec)	(lb/sec	) (sec)	
	STARTER/GENI	ERATOR COOLI	NG AIR DU	CT DISCO	NNECTED	
235 236 237 238 239 240	2.28 2.50 2.31 2.42 2.39 2.43	0.70 0.71 0.75 0.69 0.69	2.98 3.21 3.06 3.11 3.08 3.13	2.23 1.05 1.46 1.45 1.21 1.30	1.50 Non-Ext 1.70 3.35 Non-Ext Non-Ext	
	No	ORMAL NACELL	E CONFIGU	RATION		
241 242 243 244	2.64 2.43 2.36 2.48			1.15 1.45 1.36 1.27	Non-Ext 4.29 5.07 4.58	
	SIMULA	TED BURN-OUT	IN NACEL	LE COWLI	NG	
253 254 255	2.48 2.49 2.50			1.79 1.60 1.35	3.48 5.01 Non-Ext	

Although compartmental pressure measurements did not indicate a substantial increase in the compartment airflow, the airflow pattern, as evidenced by a change in the flame path, was changed when the nacelle was modified to simulate a burned-out area in the bottom aft portion of the compartment. This modification resulted in a 22-percent increase in the required weight rate of nitrogen discharged into the nacelle when compared to the undamaged nacelle nitrogen fire extinguishing requirements.

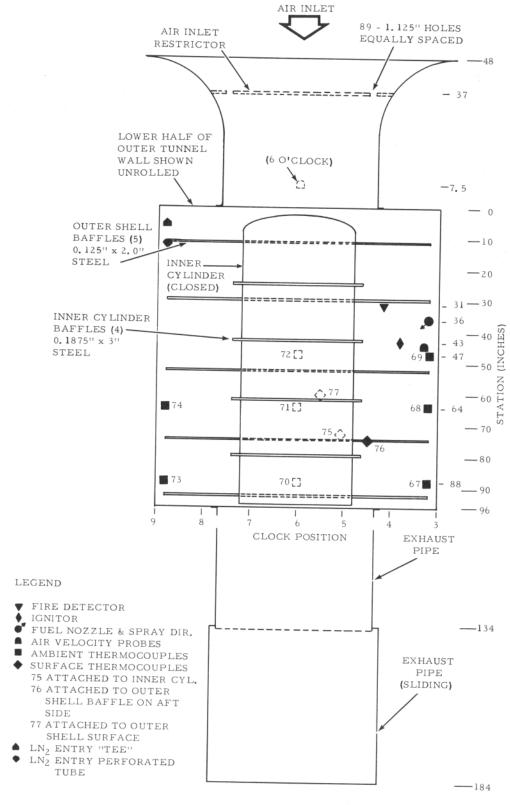
# Effect of Installation Volume and Type Discharge

Objective: The objective of this portion of the project was to determine the effects of (a) compartment volume, and (b) type of discharge on the quantity of LN2 required to extinguish fires.

Method: The majority of these tests were conducted in the Mockup Engine/Nacelle Facility. A number of applicable tests undertaken for other phases of the project were conducted in the Five-Foot Fire Test Facility.

The applicable tests in the Five-Foot Fire Test Facility were conducted using the standard JT-12 engine and nacelle with a 12.6-cubic-foot void volume within the nacelle. Standard turbine engine and wind tunnel instrumentation were utilized to record the JT-12 engine and tunnel facility operational parameters during the tests.

Tests Nos. 70 through 203, inclusive, were conducted in the Mockup Engine/Nacelle Facility. The basic facility is illustrated in Figure 8. To create a variable volume compartmentized test section and to control the airflow, a circular steel baffle was fabricated to fit the space between the inner cylinder wall and the outer cylinder wall. The baffle contained two rings of equally spaced 3/4-inch-diameter holes. The location of the facility test instrumentation and test equipment is shown in a plan view of the facility in Figure 17. For comparative purposes, the locations of the fuel-to-fire nozzle, ignitor, and LN2 nozzle were selected to duplicate the corresponding locations in the JT-12 test engine installation. Facility design required that these locations be exactly opposite, as viewed from aft, as those in the JT-12 installation.



INSTRUMENTATION AND TEST EQUIPMENT

FIGURE 17 PLAN VIEW OF MOCKUP ENGINE/NACELLE FACILITY SHOWING INSTRUMENTATION AND TEST EQUIPMENT

The relative locations of the fuel and  $LN_2$  nozzles with respect to each other were, however, the same for both facilities. A total of 12 thermocouples was installed, as shown in Figure 17. Eight of the thermocouples were used to record ambient air temperatures within the facility, and four of the thermocouples were used to record surface or skin temperatures of various metal samples within the facility. Test fire ignition and extinguishment were monitored by a radiation-type flame sensor as in the JT-12 installation.

The LN<sub>2</sub> distribution system was fabricated from 21 feet of 1-inch tubing with a 0.040-inch wall thickness and generally conformed to the Five-Foot Fire Test Facility system configuration. The distribution system and instrumentation are shown in Figure 6 as "LN<sub>2</sub> Distribution System 2." Discharge was controlled by an electrically operated valve located at the dewar. Two types of discharge nozzle, or system, were used within the test section. One type of nozzle was a standard AN bulkhead tee, directed to discharge in a vertical plane. The other system was a perforated loop welded shut at the end. The perforated loop installation and description are illustrated in Figure 18. The LN<sub>2</sub> distribution system and facility test section are shown in Figures 19 and 20.

Tests Nos. 70 through 75, inclusive, were conducted using Test Event Schedule D as described in Appendix C. This schedule included a 15-second airflow stabilization period in the test section prior to test fire ignition, a 20-second test fire preburn period prior to LN2 on, and a 10-second LN2 discharge. These tests were conducted without the aft baffle in place, thus creating an uncompartmentized "straight-through" test section. The average test section airflow was 10.3 pounds per second. Fuel flows were varied from 2.5 gpm at 42 psig, to 1.0 gpm at 55 psig, to 0.7 gpm at 40 psig. The LN2 flow was varied from 1.8 to 4.0 pounds per second. All discharges were from AN-824 nozzles.

Following this series of tests, the aft perforated baffle was installed at Station 96, which is shown in Figure 17. This essentially created a definable nacelle-type compartment with a void volume of 53 cubic feet. Test section airflow was also effectively decreased. All remaining tests in this phase were conducted using Test Event Schedules, D,  $D_1$  and  $D_2$  as described in Appendix C.

Tests Nos. 76 to 119 and 182 to 197, inclusive, were conducted with AN-824 discharge nozzles. The fuel-to-fire flows tested were 0.1 gpm at 20 and 40 psig, 0.169 gpm at 50 psig, 0.7 gpm at 40 psig, and 1.0 gpm at 55 psig. Airflow

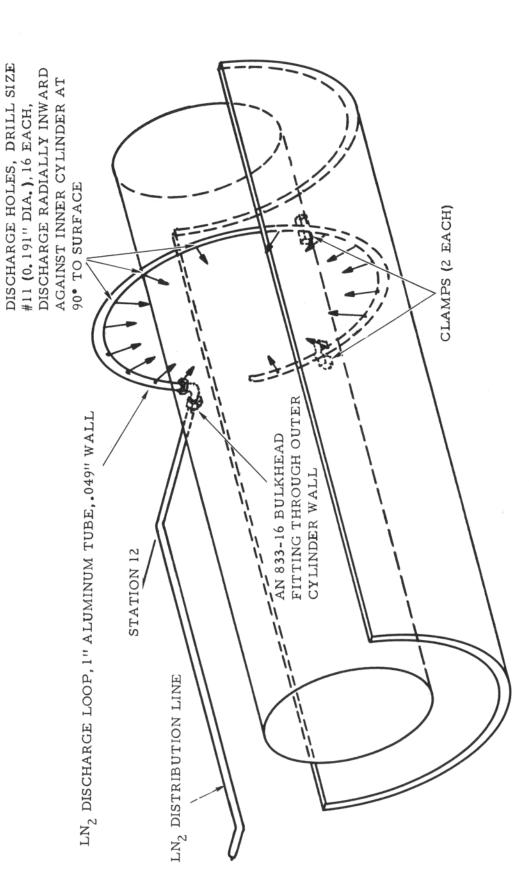


FIGURE 18 PERFORATED LOOP DISTRIBUTION SYSTEM

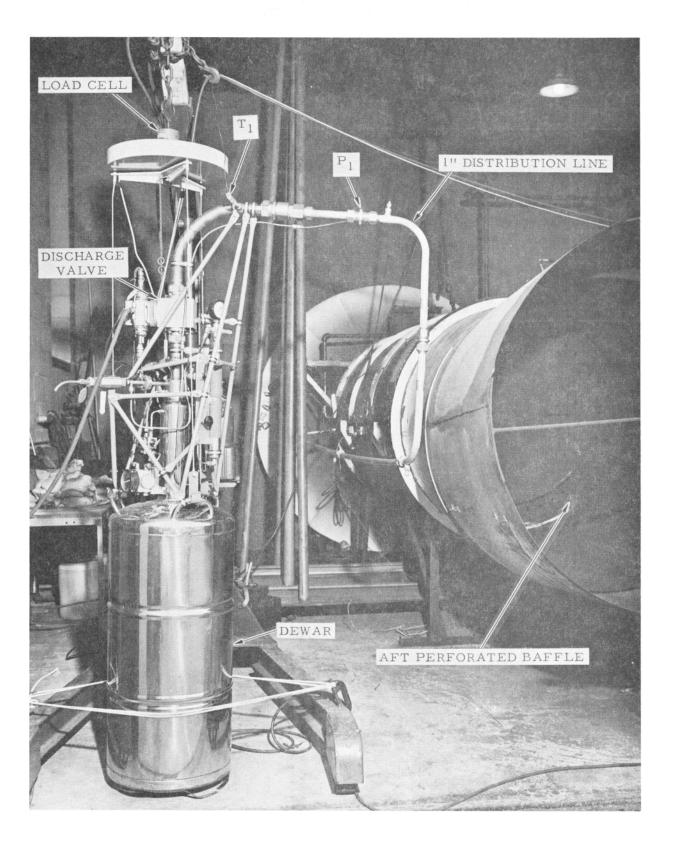
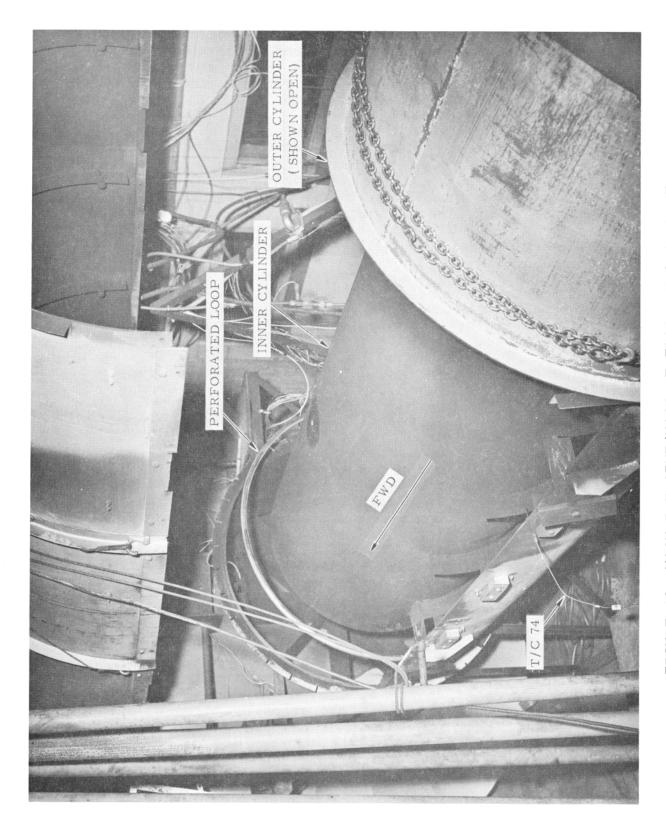


FIGURE 19 AFT VIEW OF TEST INSTALLATION IN MOCKUP ENGINE/NACELLE FACILITY



MOCKUP ENGINE/NACELLE FACILITY SHOWING TEST SECTION AND PERFORATED LOOP SYSTEM FIGURE 20

through the test section was varied by symmetrically blocking various numbers and combinations of the 89 holes (each was 1 square inch in area) in the "Air Inlet Restrictor" shown in Figure 17. Airflow values tested ranged from 0.64 to 9.01 pounds per second. LN2 discharge rates were varied from 0.9 to 3.61 pounds per second.

Tests Nos. 120 to 139 and 198 to 203, inclusive, were conducted using a perforated loop discharge system as shown in Figure 18. Compartment volume was 53 cubic feet. Fuel flows were varied from 0.421 gpm at 55 psig to 1.0 gpm at 55 psig. Airflows tested ranged from 3.69 to 9.12 pounds per second. LN2 flows were tested from 0.70 to 3.22 pounds per second.

Following this series of tests, the aft baffle was moved forward approximately 24 inches, thus establishing a compartment with a void volume of 40 cubic feet. All tests in this series were conducted with LN<sub>2</sub> discharge occurring from AN-824 tee nozzles. Fuel flows were 0.1 gpm at 40 psig, 0.421 gpm at 55 psig, and 1.0 gpm at 55 psig. Test section airflows were varied from 0.68 to 7.62 pounds per second. LN<sub>2</sub> flows ranged from 0.21 to 2.66 pounds per second.

Results: Table 6 presents the test results applicable to the determination of the effect of compartment volume and type of discharge (tee nozzle or perforated tube) on the LN2 discharge rate required for fire extinguishment. Figure 21 defines the effects of engine compartment air changes for a standard day on the LN2 discharge rate requirements. This plot was derived from the conversion of curves of WLN2 versus WA as a function of the tested compartment volumes with data obtained from Table 6. Figure 21 shows that the mass flow rate of LN2 required for extinguishment for each value of compartment airflow was influenced by the compartment volume. ' The required flow rate of LN2 increased as the volume of the compartment increased for all values of compartment air changes. For compartment air change values below 60, the volume of the compartment had a linear effect on the LN2 requirements. The slopes of all three compartment volume curves appear to be equal below approximately 60 air changes per minute. The LN2 rate requirements do not appear to increase in proportion to the ratio of compartment volume at rates below 60 air changes per minute. When the number of compartment air changes was increased beyond 60 per minute, the effect of compartment volume became increasingly significant.

TABLE 6. - SUMMARY OF NITROGEN FLOW REQUIREMENTS AS A FUNCTION OF AIRFLOW, FUEL FLOW, COMPARTMENT VOLUME, AND DISCHARGE NOZZLE

Test	Fuel Flow	Air Flow	w <sub>LN2</sub>	Time Fire Exting	Type Discharge Nozzle	Compart- ment Volume
	(gpm)	.lb/sec)	(lb/sec	(sec)		(ft3)
4 5 4 6	0.30	2.55	1.33	Non-Ext. 3.6	Open-end tee	12.6
49 50	0.30	1.87	1.41	4.0 Non-Ext.		
51 52	0.30	1.45	0.98	4.1 Non-Ext.		
54	0.10	1.04	0.73	Non-Ext.		
5 5	0.10	0.83	0.71	4.0		
5 7 5 9	0.10	0.58	0.60	3.8 Non-Ext.	Open-end tee	12.6
8 5 8 6	0.421	3.93	0.92	Non-Ext. 5.2	Open-End tee	53
89 91	1.0	3.87 3.81	0.91	Non-Ext. 3.7		
9 4 9 5	1.0	5.73 5.71	1.01	Non-Ext. 3.3		
101	0.421	5.89	1.01	Non-Ext. 2.9		
L17 L19	1.0	6.74	2.41	2.4 Non-Ext.		
L86 L87	0.10 0.10	0.64	0.09	1.6 Non-Ext.		
L89 L90	0.10 0.10	0.95	0.58	6.7 Non-Ext.		
L92 L94	1.0	8.99	2.81	1.2 Non-Ext.		
L96 L97	0.421	9.00	2.80	1.0 Non-Ext.	Open-end tee	53

TABLE 6. (continued)

Test No	Fuel Flow	Air Flow	w <sub>LN2</sub>	Time Fire Exting•	Type Discharge Nozzle	Volume
	(gpm)	(lb/sec)	(lb/sec)	(sec)		(£13)
127	1.0	6.85 6.85	1.80	1.6 Non-Ext.	Perforated Tube	53
137	1.0	3.79 3.79	0.89	2.7 Non-Ext.	Î	
198 199	0.421	5.77 5.77	1.27	1.9 Non-Ext.		
202	0.421	9.08	3.22 2.81	0.8 Non-Ext.	Perforated Tube	5 3
140 141	0.421	3.14 3.14	0.86 0.51	3.6 Non-Ext.	Open-End Tee	40
143 144	0.421	4.68	0.68	Non-Ext. 3.2	1	
148 149	0.421 0.421	5.41 5.41	0.76 0.83	3.7 Non-Ext.		
152 153	0.421	6.24	0.91	Non-Ext. 2.1		
155 156	0.421	5.77 5.77	0.94 1.47	Non-Ext. 2.0		
163 164	0.421	7.48 7.46	2.40	1.6 Non-Ext.		
168 169	1.0	7.46 7.62	2.37	Non-Ext.		
174 175	0.1	0.68	0.21 0.25	2.0 Non-Ext.		
179 180	0.1	0.99	0.35 0.71	Non-Ext.	Open-End Tee	40 .

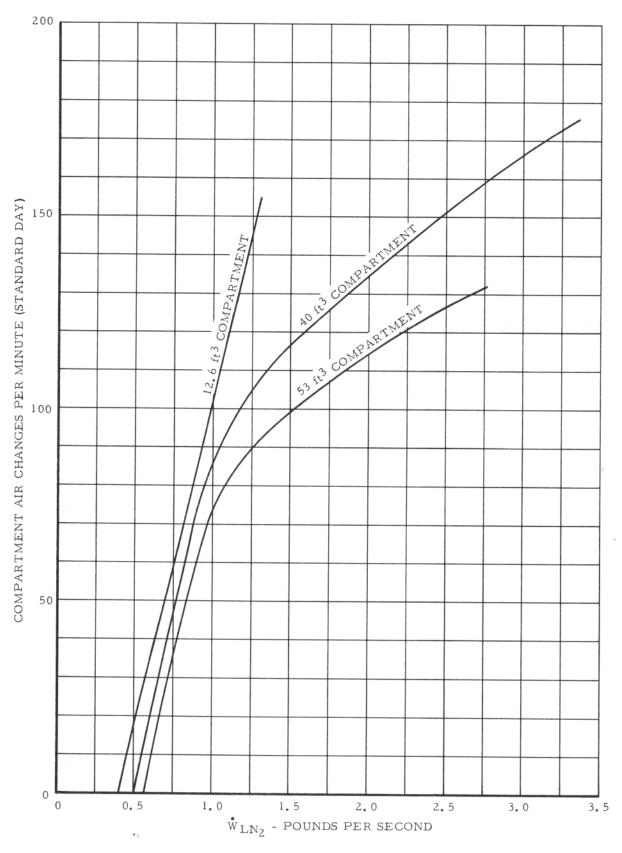


FIGURE 21 - INFLUENCE OF COMPARTMENT VOID VOLUME ON NITROGEN FLOW REQUIRED FOR FIRE EXTINGUISHMENT

The effects of the type of discharge nozzle, or system, are presented in Table 7. In the JT-12 installation, the fog nozzles and the open-end tee nozzles provided basically equal extinguishing capabilities in the low-airflow ranges in which they were tested. When tested in the simulated engine facility with a volume of 53 cubic feet, the open-end tee and the perforated tube systems provided essentially comparable capabilities at test section airflows of 4, 6, and 9 pounds per second. The slightly better performance of the open-end tee nozzle at the high airflow values might be due in part to a greater degree of airflow interruption and disruption caused by discharge from two points rather than the 16 points in the perforated tube. No attempt was made to optimize either system.

# Nitrogen Flow Characteristics for the Tested Systems

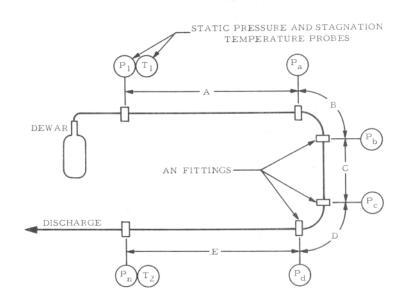
Objective: The objective of this phase of the project was to determine the effects of dewar pressure, dewar fill ratio, line size, line length, nitrogen flashing, type discharge, and fittings on the flow rate of LN<sub>2</sub> through a distribution system.

Method: The tests were conducted in the laboratory area of the Fire Test Facility Building. A description of the systems tested is presented in Figure 22. Discharge from the dewar was initiated from the facility control room by means of an electrically operated cryogenic ball valve located between the dewar and the P1/T1 probe positions. Two test distribution systems were utilized; one system was fabricated from 1-inchdiameter tubing, and the other system was fabricated from 1/2-inch-diameter tubing. Each system was composed of two 12-foot and one 4-foot straight sections, one 90° bend section with a 2-foot radius, and one 90° bend section with 1-foot radius. The individual sections of the systems were connected with standard AN fittings. The tubing was uninsulated for all tests. The l-inch-diameter tubing system was tested with openend, spray, and perforated tube outlets. The 1/2-inch-diameter tubing system was tested with an open-end outlet. Static wall pressures were recorded by pressure transducers located at each of the AN fittings as shown in Figure 22. Stagnation temperatures were recorded by thermocouples located at the first and last AN fittings as shown in Figure 22.

For the tests, the nominal  $\rm LN_2$  fill weights in the dewar were 33, 67, and 86 pounds, and the nominal dewar saturation pressures were 40, 70, and 100 psig. Nominal discharge duration was 16 seconds to allow stabilization of the  $\rm LN_2$  flow.

- EFFECT OF TYPE OF DISCHARGE ON NITROGEN FLOW RATE REQUIREMENTS FOR FIRE EXTINGUISHMENT TABLE 7.

		LN2 D	LN2 DISCHARGE RATE FOR:	FOR:	
Engine Installation	LN2 System	WA≝l lb/sec	4 lb/sec	6 lb/sec	9 lb/sec
JT-12 (12.6 cu ft vol)	Fog Nozzle Open-end Tee	0.8 lb/sec 0.8 lb/sec			
Simulated Engine (53 cu ft vol)	Open-end Tee Perforated Tube	0.6 lb/sec	1.0 lb/sec 0.9 lb/sec	1.2 lb/sec 1.3 lb/sec	2.8 lb/sec 3.0 lb/sec



SYSTE	М	TUBE O.D. (INCHES)	TUBE WALL THICKNESS (INCHES)	DIMENSIONS A & E (INCHES)	CENTERLINE DIMENSION B (INCHES)	DIMENSION C (INCHES)	CENTERLINE DIMENSION D (INCHES)	TUBE FITTINGS
1,2,3		1.00	0.065	142.5	37.7	48	18.8	AN 815-16 AN 818-16 AN 819-16
4		0.50	0.035	142.5	37.7	48	18.8	AN 815-8 AN 818-8 AN 819-8

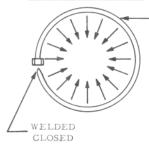
#### SYSTEM | DISCHARGE

1" OUTSIDE DIAMETER WITH 0.065" WALL OPEN-END TUBE. NOZZLE AREA = 0.595 in<sup>2</sup>

# SYSTEM 3 DISCHARGE

FOUR #1-1/4 H 12 "FULLJET" SPRAY NOZZLES (SPRAYING SYSTEMS COMPANY) TOTAL NOZZLE AREA = 0.645in<sup>2</sup>

### SYSTEM 2 DISCHARGE



1" OUTSIDE DIAMETER WITH
0.049" WALL ALUMINUM TUBE.
DISCHARGE DIRECTED RADIALLY
INWARD BY 16 EQUISPACED 0.191"
DIA. HOLES. TOTAL NOZZLE
AREA = 0.458in<sup>2</sup>.

## SYSTEM 4 DISCHARGE

1/2" OUTSIDE DIAMETER WITH 0.035" WALL OPEN-END TUBE. NOZZLE AREA = 0.145in<sup>2</sup>

FIGURE 22 - SYSTEM CONFIGURATIONS AND INSTRUMENTATION USED FOR NITROGEN FLOW CHARACTERISTIC STUDY

In addition to the tests conducted specifically for this portion of the program, data from Tests Nos. 1 through 203, inclusive, were used to determine the relation between nitrogen flow and nitrogen quality.

Results: An overall summary of the system and component pressures and temperatures resulting from this series of tests is presented in Appendix G. Flow rates for the various systems tested are presented in Table 8. Figure 23 illustrates that a relationship exists between nitrogen flow rate, dewar saturation pressure, and dewar fill ratio. The data are shown for a l-inch outside-diameter tube system with a 0.065-inch wall thickness. The nitrogen was discharged through the open-end of the last tube in the system. The tubing system component connector fittings were of the same inside diameter as the tubing components, thus essentially creating a straight-through system with no nozzle restriction.

A similar relationship between flow rate, saturation pressure, and fill ratio is presented in Figure 24. The 1-inch system was the same as described for Figure 23, with the exception that the nitrogen was discharged through the perforated loop. A comparative relationship is also shown in Figure 23 for a 1/2-inch-diameter open-end nozzle system. These figures illustrate that for any initial dewar saturation pressure, the nitrogen flow rate is influenced by the quantity of nitrogen in the dewar. Thus, on aircraft where quantities of LN2 are normally expended during flight for inerting fuel tanks or for other purposes, the available LN2 flow rate would decrease as the LN<sub>2</sub> is withdrawn from the dewar. Extrapolation of the curves to zero flow at a zero fill ratio indicate increasingly greater drops in flow rates as the dewar is emptied. The  ${\tt LN}_2$ flow rate would also be lowered by inadvertently saturating the dewar below the design saturation pressure during the filling operation.

As previously discussed, the amount of flashing in the nitrogen distribution system was controlled by inserting various sizes of orifices in the line at the dewar outlet. The pressure drops through the 21 feet of tubing between the orifice, and the discharge nozzles were recorded throughout the test program. The pressure loss for each size of tubing tested was determined to be primarily a function of the quality of the nitrogen downstream of the orifice and the weight flow rate of nitrogen. This is shown in Figure 25 for a 1-inch tube system with pressures measured 3 seconds after initiating the nitrogen discharge. The quality of the nitrogen was determined from Figure 26 as a function of the dewar saturation pressure and the pressure drop between the dewar and downstream of the

TABLE 8. - NITROGEN FLOW RATE AS A FUNCTION OF SYSTEM CONFIGURATION, DEWAR SATURATION PRESSURE, AND DEWAR FILL RATIO

204 205	contradit det	Pressure at Nozzle (Pr)	Saturation	Dewar Fill Ratio	LN2 flow Rate	Flow Rate (WLN2/A nozzle)
0			(psig)	(%)	(lb/sec)	(lb/sec-in²)
0		34.4	105	101		•
	ı —	39.1	105	19	2.67	64.4
0		°	105	35	$\sim$	
0		00	7.0	98	$\sim$	
0		19.8	7.0	34	7	
0		7.	7.0	49	$\sim$	
$\vdash$		9	0+1	98	.60/1.	
$\vdash$		9	0+1	67		2.
$\vdash$		2.	0+1	34	.28/1.20)	2.
$\vdash$		38.3	100	8.5	$\overline{}$	•
$\vdash$	2	m	0	98		
$\vdash$		i,	0	99	$\sim$	•
Н		49.1	100	34	$\sim$	
$\vdash$		5	0	33	$\alpha$	
217		+	70	79	_	
$\vdash$		39.2	70	88	$\sim$	90.4
218		6	70	33	(0	•
$\vdash$		0	0+	85	$\sim$	•
2		28.4	0 †	8 9	$\sim$	•
221	_	24.9	0 †1	33	$\sim$	•

TABLE 8 (continued)

Nozzle LN2 Flow Rate (W <sub>LN2</sub> /A nozzle)	$(lb/sec-in^2)$	!	!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!!	1 1 1	1 1	1 1 1	1 1	3.03	2		0	י ע	1 C	• •
System LN2 Flow Rate	(lb/sec)	2.50	2.18	•	1.52		1.04	44.0	•	•	0.14	0.22		.2
Dewar Fill Ratio	(%)	85	32	8.5	34	85	34	84	6.5	34	84	84	63	33
Dewar Saturation Pressure	(psig)	100	100	7.0	70	0 † 0	0 1	105	105	105	4.5	7.5	7.5	0 †1
Average Static Pressure at Nozzle (P <sub>n</sub> )	(bsig)		•	•	37.3		٠,		•	21.6	•			7.5
System Configuration		e .				-		<b>+</b> -						
Test No.		222	'A (	$\sim$	$\sim$ 0	V	N 10	V (	V	$\mathbf{r}$	$\sim$	$\sim$	$\sim$	$\sim 1$

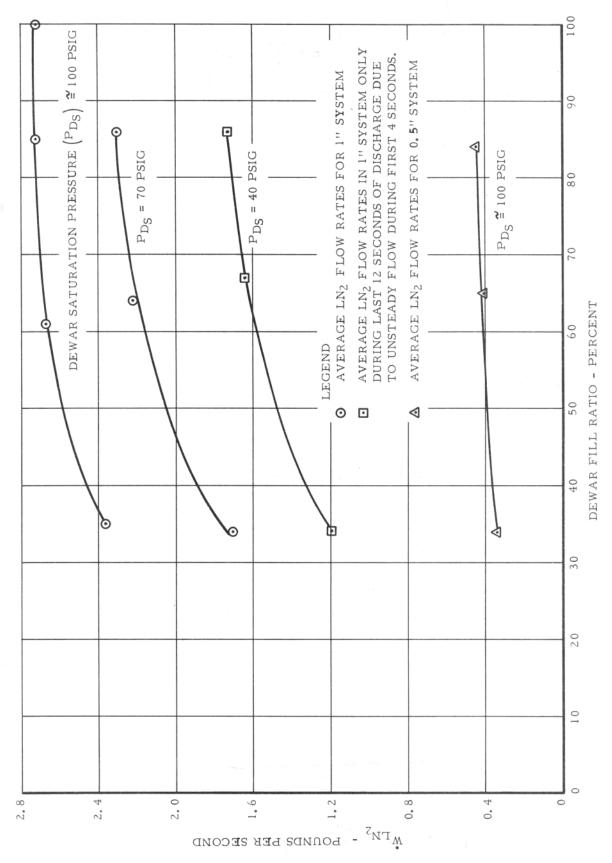


FIGURE 23 - NITROGEN FLOW RATES AS A FUNCTION OF DEWAR SATURATION PRESSURE AND DEWAR FILL RATIO FOR 1- AND 1/2-INCH-DIAMETER OPEN-END NOZZLE SYSTEMS

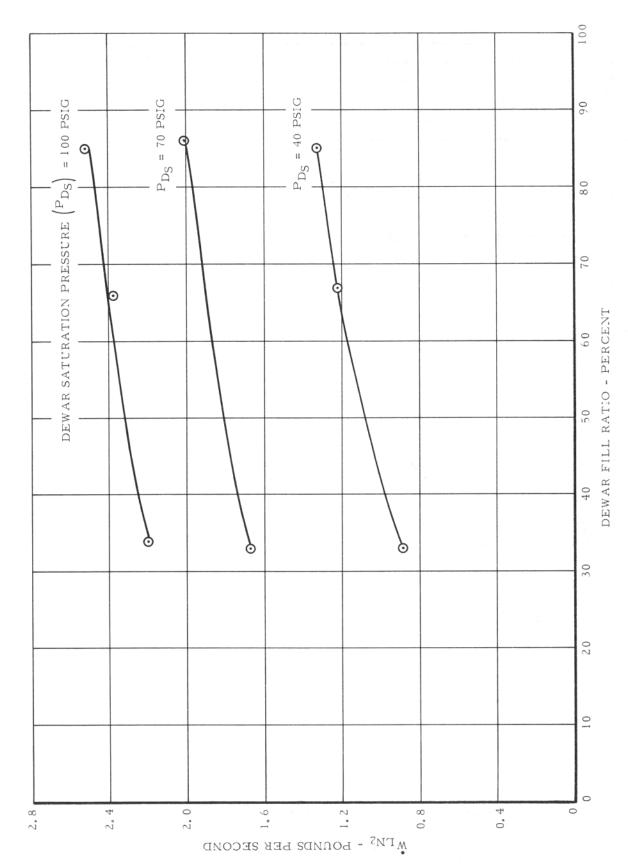


FIGURE 24 - NITROGEN FLOW RATES AS A FUNCTION OF DEWAR SATURATION PRESSURE AND DEWAR FILL RATIO FOR 1-INCH-DIAMETER PERFORATED TUBE DISCHARGE SYSTEM

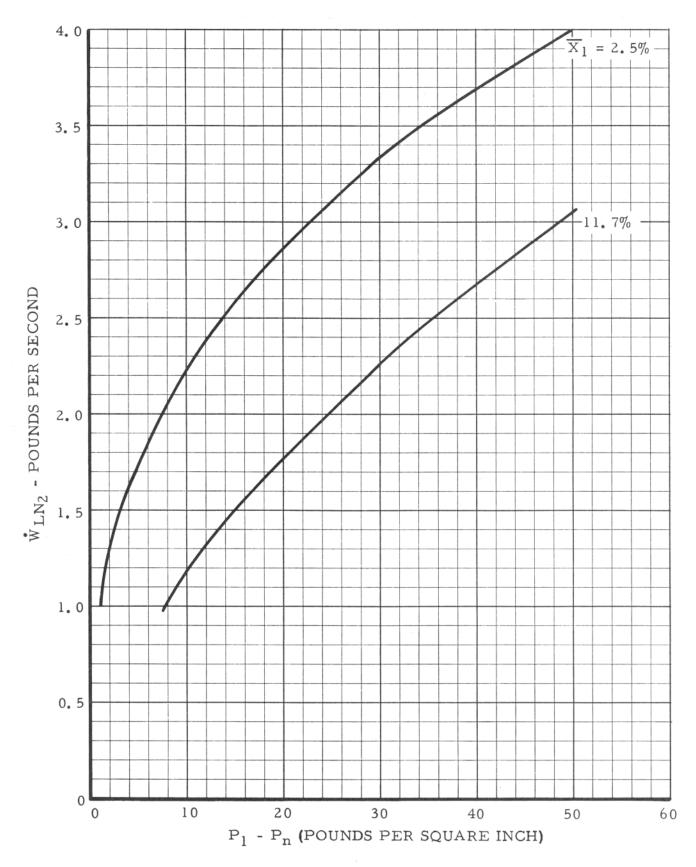


FIGURE 25 SYSTEM PRESSURE LOSS AS A FUNCTION OF NITROGEN FLOW RATE AND QUALITY

orifice. The relationship shown in Figure 26 was calculated from temperature-entropy data for nitrogen. As an approximation in the quality calculations, it was assumed that the thermodynamic process involved was an irreversible, steadyflow, adiabatic process, similar in anture to a throttling process. This figure was used to determine the nitrogen quality at locations in the distribution system where pressure measurements were taken. The quality curves shown in Figure 25 were developed by grouping all the data resulting from tests with nitrogen flows of 1 pound per second or greater, through the 21-foot l-inch tube system into qualities less than 8 percent and 8 percent or greater. The qualities shown are average values for the tests in each grouping. The curves represent the least square fit in the form of a power curve function for the combined pressure loss and nitrogen flow rate data in each grouping. This figure indicates that from a design standpoint, it is important to minimize the pressure losses in the distribution system. When a pressure loss occurs in a flow system, quantities of liquid nitrogen, proportional to the losses, flashes to a gas and increases the quality of the nitrogen. Pressure losses downstream in the flow system, therefore, become substantially greater due to the higher quality (x) of the nitrogen. The fill ratio, saturation pressure and the nozzle size or discharge type did not substantially affect this relationship. The tests included fill ratios from 15 to 109 percent, dewar pressures from 65 to 115 psig, AN-834-4 to -16 nozzles, and perforated tube-type discharges.

Figure 27 shows the nitrogen weight-flow density through the nozzle as a function of the dewar fill ratio, the saturation pressure, and the quality of the nitrogen entering the nozzle. Again the quality is based on Figure 26 and measured pressure losses between the dewar and the nozzle. This relationship was established from tests (1 through 203) with 1/2-, 3/4-, and 1-inch type by 21-foot-long distribution systems; open-tube nozzles and the perforated tube-type discharge; and dewar pressures grouped from 80 to 90 psig and from 90 to 110 psig. The curves represent averaged quality values and the least square fit in the form of a power curve function, for the combined initial fill ratio and nitrogen flow density data (3 seconds into the nitrogen discharge), for each of the quality and dewar pressure groupings. The significant factors shown in this figure are the effects of fill ratio and dewar pressure on the nitrogen discharge rate. If the duration of the discharge was longer than the 10 seconds used in these tests, the nitrogen flow density would be

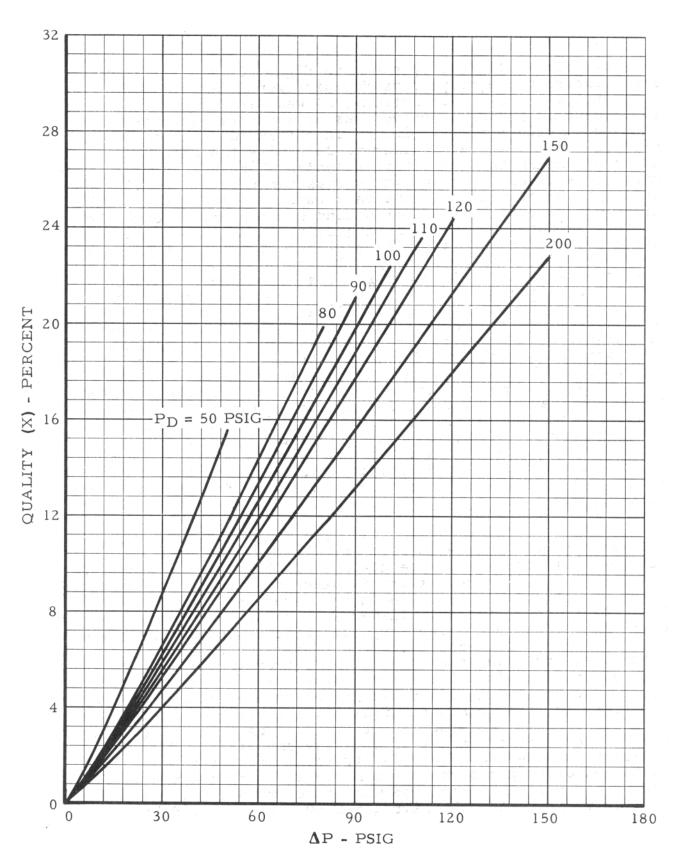


FIGURE 26 NITROGEN QUALITY AS A FUNCTION OF SYSTEM PRESSURES

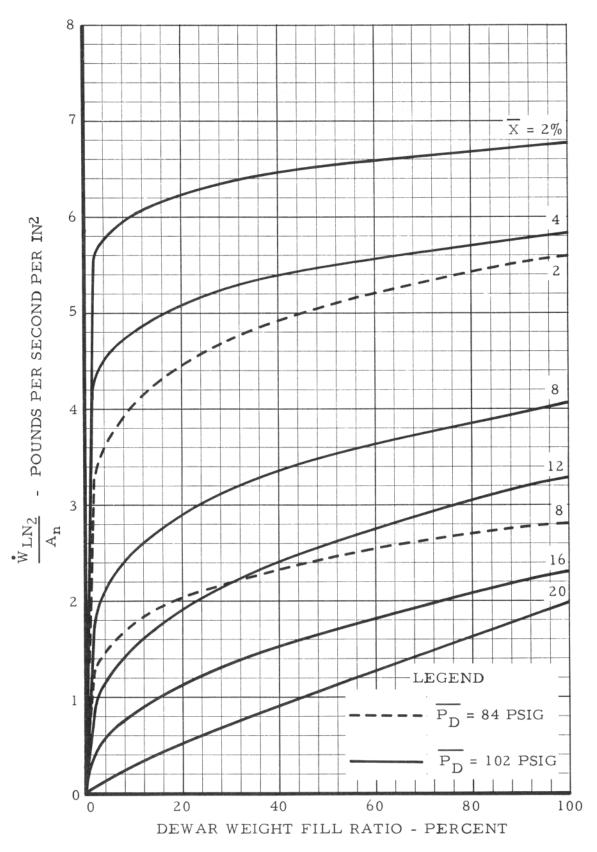


FIGURE 27 - NOZZLE DISCHARGE RATES FOR A 1-INCH-DIAMETE NITROGEN SYSTEM AS A FUNCTION OF DEWAR FII RATIO, SATURATION PRESSURE, AND QUALITY

expected to be substantially less at higher discharge rates, due to the larger changes between the initial and final fill ratios. However, if the capacity of the dewar is increased, then the nitrogen flow density would be expected to increase, due to the smaller difference between initial and final fill ratios.

# CONCLUSIONS

Based upon the results of the evaluation of cryogenic nitrogen as an aircraft powerplant fire-extinguishing agent, it is concluded that:

- 1. The use of cryogenic nitrogen as an effective aircraft powerplant fire-extingushing agent is feasible from a functional standpoint.
- 2. The flashing of saturated cryogenic nitrogen in a distribution system increases the pressure losses in the lines and decreases the transfer rate substantially.
- 3. For equal length and diameter distribution systems, the location of the discharge valve and distribution line pressurization has no appreciable effect on the LN2 transfer and fire-extinguishment capabilities of the system.
- 4. The rate at which the nitrogen is discharged is critical with respect to the effectiveness of the extinguishing system.
- 5. A long-duration LN<sub>2</sub> system discharge can provide a greater safety advantage than a conventional short-duration halogenated agent system discharge with respect to cooling potential reignition sources and reducing the vaporization rate of any fuel remaining within the nacelle after extinguishment.
- 6. Although no operational problems were encountered with the engine or components during the discharge of the low-temperature nitrogen within the test installations, additional testing will be required to completely define the effects of an inadvertent system discharge on an aircraft engine installation.
- 7. Fire-extinguishing protection for a low-airflow nacelle which has received damage in the form of large air leakages or openings in the cowling is feasible with a nitrogen system without substantially increasing the quantity of nitrogen required. The increase in the quantity of agent required for this added protection will, however, be more pronounced for a system with long distribution lines than for one with short distribution lines.

- 8. The type of discharge from the nozzle, whether liquid or gaseous, is not critical from the standpoint of extinguishing effectiveness.
- 9. For a low-flow nacelle, the volume of the compartment has little effect on the nitrogen discharge requirements.

# APPENDIX A

DESCRIPTION AND RESULTS OF PRELIMINARY FEASIBILITY STUDY

# DEPARTMENT OF TRANSPORTATION FEDERAL AVIATION ADMINISTRATION NATIONAL AVIATION FACILITIES EXPERIMENTAL CENTER ATLANTIC CITY, NEW JERSEY 08405 PROPULSION SECTION, NA-542

April 1969

NA-542

DATA REPORT NO. 54

EXTINGUISHING AIRCRAFT POWERPLANT FIRES
WITH LIQUID NITROGEN
PROJECT NO. 520-001-15X

James E. Demaree

#### ABSTRACT

The effectiveness of liquid nitrogen (LN $_2$ ) as a fire-extinguishing agent for the protection of aircraft powerplant installations was investigated under full-scale simulated low altitude flight conditions at the Federal Aviation Administration's National Aviation Facilities Experimental Center (NAFEC), Atlantic City, New Jersey. An LN $_2$  discharge system was developed and used to extinguish fires in the compressor and accessory compartment of an aft pod, side-mounted powerplant nacelle. The minimum quantitites and discharge rates required to extinguish test fires were determined for LN $_2$  as a function of nacelle ventilation rates. Comparative tests were conducted to determine the relative effectiveness of LN $_2$  to the fire-extinguishing agent currently being used on the majority of commercial United States transport aircraft.

Preliminary test results indicated that (1)  $\mathrm{LN}_2$  is effective in extinguishing fires in aircraft powerplant compartments; (2) the quantity of  $\mathrm{LN}_2$  expected to be available from a  $\mathrm{LN}_2$  fuel tank inerting system would be sufficient to extinguish the fires; and (3) on aircraft where a large quantity of  $\mathrm{LN}_2$  is available, an  $\mathrm{LN}_2$  fire extinguisher system could provide greater in-flight powerplant fire protection than the limited quantity of agent available in a conventional high rate of discharge system.

#### INTRODUCTION

## Purpose

Project No. 520-001-15X was undertaken to provide fundamental desgin criteria for aircraft powerplant fire-extinguishing systems which utilize a liquid nitrogen ( $LN_2$ ) supply common to other aircraft systems and to determine the relative effectiveness of  $LN_2$  with other fire-extinguishing agents.

## Background

The high performance of present day aircraft encourages the use of systems that may provide more than a single service in aircraft operation. LN<sub>2</sub> is being considered as a multi-service system in providing (1) inerting capability to the ullage space in aircraft fuel tanks; (2) fuel "scrubbing" to remove foreign matter such as water and oxygen; (3) galley cooling; and (4) fire protection to all potential fire zones such as powerplant, auxiliary power units (APU), and cargo-baggage area.

One proposed  $LN_2$  system for a four-engine, transport-type aircraft would incorporate a Dewar with a 300-pound  $LN_2$  capacity. During a typical trans-continental flight, approximately 200 pounds of  $LN_2$  would be used in support of associated systems on the aircraft. The remaining 100 pounds, at the terminal point, could be utilized for fire protection. The weight/cost of such an installation thus encourages the utilization of  $LN_2$  for more than one function.

During the past 25 years, the problem of providing protection against in-flight aircraft powerplant fires has been a formidable one. To provide adequate protection, several factors are involved regarding an acceptable agent used in fire extinguishment. Primary objectives of suitable fire extinguishing agents are:

- 1. Equally suitable for hydrocarbon and electrical fires.
- 2. Toxicity level should be below a range considered injurious to human health.
- 3. Be effective when stored at temperatures which may range from  $-65^{\circ}F$  to approximately  $500^{\circ}F$ .
- 4. Storage life, in both the aircraft and in ground climatic conditions, be for extended duration.
- 5. The final cost and production capability be within the economic range of the customers.

- 6. The corrosive characteristics of the agent should be as low as present technology will permit.
- 7. The pressure-temperature characteristics should be such to provide adequate storage and in-flight containment capabilities.

 $\rm LN_2$  appears to meet most of the above requirements. However, there is a lack of technical knowledge on the effectiveness of  $\rm LN_2$  in extinguishing fires.

#### DISCUSSION

### Test Installation

Tests were conducted in a 5-foot Fire Test Facility, described in the SRDS Technical Facilities at NAFEC, Handbook RD P 6000.2, paragraph 7-1 to 7-9. This facility is powered by two J-57 turbojet engines which produces airflow through a 64-inch-diameter by 16-foot-long test section. The number 2 engine nacelle from a C-140 aircraft was mounted in this test section as shown in Figure 1. A JT-12 turbojet engine was housed in this two-zoned aircraft nacelle. This engine utilizes a compressor bleed arrangement which discharges all of the compressor bleed air through a series of holes around the circumference of the engine compressor case into the nacelle void space and not overboard as in most conventional turbojet and turbofan installations. Further information concerning this air flow and its importance during the tests will be discussed under test results in this report.

The facility with this engine installation was normally limited to simulated level flight conditions from sea level to 5,000 feet and velocities from 0 to 350 knots on a standard day.

### Test Equipment

The primary objective of this project was to evaluate the effectiveness of  $\mathrm{LN}_2$  as a powerplant fire-extinguishing agent; therefore, the major modification to the test installation was to the fire-extinguishing agent distribution system. The conventional system was not utilized and a distribution system utilizing four fog nozzles which broke up the liquid particles of  $\mathrm{N}_2$  was installed. This system is shown in Figure 2.

The  $\rm LN_2$  used during this testing was stored in a portable storage unit adjacent to the building and was transferred to a 300-pound capacity Dewar, shown in Figure 3. Flow duration of  $\rm LN_2$  from the Dewar and the flow rate were controlled by hand-operated ball valves. Flow of  $\rm LN_2$  was routed from the Dewar through these valves and through a 1-inch outside-diameter line to the powerplant nacelle where the  $\rm LN_2$  was discharged through the four fog nozzles into the 13-cubic-foot forward accessory compartment.

The Dewar used during this program was designed to permit filling with subcooled LN, from the storage unit and saturation with gaseous nitrogen  $(GN_2)$ . Storage pressure in the Dewar was maintained between 100 psig and 140 psig during the test program.

### Test Instrumentation

Standard wind tunnel instrumentation, utilized with the facility, consisted of static pressure pickups at various stations along the test section and read-out on water and mercury manometers in the control room.

The operation of both drive engines and the test engine was monitored with standard aircraft powerplant instrumentation. The power setting for these engines was set up using rotor speed and turbine discharge pressure. Airflow velocity through the test section was indicated on a Mach meter.

Ambient air temperature within the powerplant nacelle of the test engine was measured using 28-gauge chromel-alumel thermocouples. The output signals of these thermocouples were recorded on potentiometer-type recorders. Engine case temperature, at selected locations, was measured by chromel-alumel thermocouples spot welded directly to the engine case and were recorded on recording potentiometers in the control room.

Three instrumentation stations were used on the  $\rm LN_2$  system. A load cell was utilized to record weight of the  $\rm LN_2$  and container on an oscillograph.

Line pressure/temperature signals at the Dewar and at a point where the  $\rm LN_2$  system entered the powerplant nacelle were recorded on an oscillograph. These combined signals enabled calculations to be made of  $\rm LN_2$  discharge rate, total flow and time/temperature and pressure values as the  $\rm LN_2$  flowed through the line and into the nacelle. The test fire sequence and duration were manually controlled in the control room.

### Test Procedures

Tests simulated flight conditions under which a fire could occur and test conditions were primarily set to control the amount of bleed air flowing into the nacelle. Airflow into the nacelle was a function of engine compressor speed and test section Mach number. The calculated bleed air flow was based on information contained in the manufacturer's JT-12 engine handbook. The combined values of ram air velocity in the test section and compressor rotation speed provided bleed air flows from 0.4 pounds per second to 2.9 pounds per second. The airflow provided by the blast tubes ranges from 0.1 to 0.2 pounds per second dependent on test section velocity. These airflow values are presented in Table I for each test conducted.

Basically three test schedules were used throughout the test program as described in Table II. The fire duration prior to engine cutoff was decreased to minimize fire damage to the nacelle. Test section air velocity was adjusted during the test engine power reduction to control the amount of bleed air flowing into the nacelle.

All test fires within the powerplant nacelle resulted from spray releasing and spark igniting JP-4 jet fuel. Fuel flow to the fire was decreased from 0.4 gallon per minute to 0.1 gallon per minute (at 20 psig) during the first fire tests.

The fire tests were conducted under conditions as outlined in Table II. The test section Mach number was established by the operation of the drive engine after setting the test engine at the required power. The test sequence was initiated and the test fire allowed to burn for a predetermined duration; then the test engine was shut down as in an emergency procedure. The test section velocity was maintained at a desired value determined by pretest planning. The LN<sub>2</sub> flow was controlled by manual operation of both the throttling valve and the on-off valve.

### Results

Thirty-four tests were conducted during this first phase of a program designed to investigate the various parameters involved with the proposed use of  $LN_2$  as a fire-extinguishing agent. They were conducted during a period from September 3, 1968, through November 5, 1968.

The ability to utilize  $\mathrm{LN}_2$  as an effective fire-extinguishing agent appears to be predicated on the rate of flow rather than a duration of flow. Fires were successfully extinguished in 2 to 3 seconds when  $\mathrm{LN}_2$  flow rate was above approximately 1.4 pounds per second and the maximum airflow in the compartment was maintained. As the airflow value decreased, the  $\mathrm{LN}_2$  flow rate required for extinguishment decreased, as noted in Table I.

The fire detectors shown in Figure 4 were utilized as flame sensors only in determining positive ignition time of the fuel and evidence of extinguishment time either by the supply of LN $_2$  or the back-up  $^{\rm CO}_2$  system. These times were recorded on an oscillograph, and were used in determining extinguishing time as indicated in Table I.

The LN $_2$  discharge rate necessary for extinguishment is shown in Figure 5, as a function of nacelle airflow. Assuming a complete mixing and disregarding oxygen consumption by the fire, 5-percent, 10-percent, and 15-percent oxygen concentrations in the air are also shown. With two exceptions, when the oxygen value dropped below 15 percent test fires were extinguished.

A cooling effect was apparent during the  $\mathrm{LN}_2$  discharge. This effect, although not fully investigated during this phase of the project, is considered to be beneficial in decreasing the probability of reignitions by cooling potential hot-surface ignition sources. This, together with a probable excess in the amount of  $\mathrm{LN}_2$  available, could provide a greater degree of protection than most conventional powerplant fire-extinguishing systems.

Only two comparative tests were conducted with Bromotrifluoromethane (CBrF<sub>3</sub>). Although the minimum quantity of agent required for extinguishment was not determined, it is estimated that LN<sub>2</sub> requires approximately three to four times more agent for extinguishment as compared to CBrF<sub>3</sub>.

The effectiveness of  $LN_2$  as a fire-extinguishing agent is considered due to cooling and oxygen dilution and not to a chemical reaction as in the case of most agents being used on today's aircraft for powerplant fire protection.  $LN_2$  boils at -320°F at one atmosphere, has a heat of vaporization of 85 British Thermal Units per pound and each pound expands to 13.8 cubic feet of gas at  $70^{\circ}$ F and one atmosphere. In comparison, CBrF<sub>3</sub> boils at -72°F at one atmosphere, has a heat of vaporization of 48 British Thermal Units per pound and each pound expands to 2.6 cubic feet of gas at  $70^{\circ}$ F and one atmosphere.

The data presented in this report represent the first phase of project to determine the effectiveness of LN<sub>2</sub> as an extinguishing agent. The effects of line lengths and size, nozzle configuration, cooling during fire extinguishment, LN<sub>2</sub> storage container pressure and rates of discharge are items under consideration for future investigation.

TABLE I

LN2 FIRE EXTINGUISHING TEST DATA SUMMARY

Fire	sec	NA	NA	NA	:	;	6	3.9	4.3	;	1	1	;	;	0.8	2.5	2.7
Fire	Dalled Talled	Yes	Yes	Yes	No	No	Yes	Yes	Yes	No	No	No	No	No	Yes	Yes	Yes
LN2 Flow Duration	1	15.7	10.5	10.0	7.6	10.0	12.9	6.6	10.1	8.3	7.8	9.85	9.3	10.85	11.15	5.1	4.8
LN2 Flow Rate	lbs/sec	3.18	2.48	1.00	0.21	0.35	1.01	1.06	0.89	96.0	0.83	0.86	0.97	06.0	06.0	1.96	1.56
Fuel Flow Press.	psig	55	30	20	20	20	20	20	20	20	20	20	20	20	20	20	20
Fuel	gal/min	0.42	0.30	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10
Fire Location	(3)	A	В	В	В	g	В	В	В	Ø	g	В	Ø	В	В	g	В
Nacelle Air Flow	lbs/sec (2)	08.0	2.42	2.12	2.22	2.30	2.09	2.68	2.69	2.74	2.77	2.81	2.81	2.68	2.81	2.83	2.79
Test	(1)	A	Ą	В	В	В	В	В	В	*2	B*	В	<b>19</b> *	g	*	P.	B*
Test No.		1.	2.	ů.	. 4	5.	• 9	7.	8	.6	10.	11.	12.	13.	14.	15.	16.

Fire		<b>6.7</b>	5	2.6		1	1	2.7			;	1	;	2.1	2.15	2.6
Fire Extinguished	o o	0	Yes	Yes	No	No	No	Yes			No	No	O	Yes	Yes	Yes
Flow Duration	3.05		5.05	2.95	5.35	5.8	5.2	5.05			7.2	7.3	7.3	7.3	7.45	7.0
Flow Rate lbs/sec	1.64	D	1,39	1.61	0.51	0.57	0.53	0.79			0.21	0.55	0.55	1,1	0.94	0.86
Flow Press.	20	Ι 0 Δ	20	20	20	20	20	20	20	20	20	20	20	20	20	20
Fuel Flow gal/min	0.10		0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10	0.10
Fire	(c) g	E S I	æ	В	В	В	В	В	В	В	В	В	В	В	В	В
Nacelle Air Flow 1bs/sec	2.78	H	2.87	2.63	1,55	1.50	1.08	1.09	2,56	2.50	0.85	0.80	09.0	0.58	0.57	0.37
Test Sequence (1)	B≯		B*	B*	B*	B*	B*	B*			B*	B∻	O	S	2	O O
Test No.	17.	18.	.61	20.	21.	22.	23.	24.	,) 25.	3) 26.	27.	28.	29.	30.	31.	32.
	Test Nacelle Fire Fuel Flow Flow Flow Fire Sequence Air Flow Location Flow Press. Rate Duration Extinguished  (1) lbs/sec gal/min psig lbs/sec sec	Test Nacelle Fire Fuel Flow Flow Flow Fire Sequence Air Flow Location Flow Press. Rate Duration Extinguished  (1) 1bs/sec (2) (3) 8al/min psig 1bs/sec sec  B* 2.78 B 0.10 20 1.64 3.05 ver	Test         Nacelle         Fire         Fuel         Flow         Flow         Flow         Fire           (1)         1bs/sec         (3)         gal/min         psig         1bs/sec         sec           B*         2.78         B         0.10         20         1.64         3.05         Yes           T         E         S         T         V         O         I         D	Test         Nacelle         Fire         Fuel         Flow         Flow         Flow         Fire         Fire           (1)         1bs/sec         (3)         gal/min         psig         1bs/sec         sec           B*         2.78         B         0.10         20         1.64         3.05         Yes           B*         2.87         B         0.10         20         1.39         5.05         Yes	Test         Nacelle         Fire         Fuel         Flow         Flow         Flow         Flow         Flow         Fire           (1)         1bs/sec         (3)         gal/min         psig         1bs/sec         sec         sec           B*         2.78         B         0.10         20         1.64         3.05         Yes           B*         2.87         B         0.10         20         1.39         5.05         Yes           B*         2.63         B         0.10         20         1.61         2.95         Yes	Test         Nacelle         Fire         Fuel         Flow         Flow         Flow         Fire         Fire           (1)         1bs/sec         (3)         gal/min         psig         1bs/sec         sec         sec           (1)         1bs/sec         (3)         gal/min         psig         1bs/sec         sec         sec           B*         2.78         B         0.10         20         1.64         3.05         Yes           B*         2.87         B         0.10         20         1.39         5.05         Yes           B*         2.63         B         0.10         20         1.61         2.95         Yes           B*         1.55         B         0.10         20         0.51         5.35         No	Test         Nacelle         Fire         Fuel         Flow         Flow         Flow         Fire         Fire           (1)         1bs/sec         (3)         8a1/min         psig         1bs/sec         sec         Fire           (2)         (3)         (3)         8a1/min         psig         1bs/sec         sec           B*         2.78         B         0.10         20         1.64         3.05         Yes           B*         2.87         B         0.10         20         1.39         5.05         Yes           B*         1.55         B         0.10         20         1.61         2.95         Yes           B*         1.50         B         0.10         20         0.51         5.35         No           B*         1.50         B         0.10         20         0.57         5.8         No	Test         Nacelle         Fire         Fuel         Flow         Flow	Test         Nacelle         Fire         Fuel         Flow         Flow         Flow         Fire         Fire           (1)         1bs/sec         (3)         gal/min         psig         1bs/sec         sec         Friending Fried           B*         2.78         B         0.10         20         1.64         3.05         Yes           B*         2.87         B         0.10         20         1.39         5.05         Yes           B*         1.55         B         0.10         20         0.51         5.35         No           B*         1.50         B         0.10         20         0.57         5.8         No           B*         1.08         B         0.10         20         0.57         5.8         No           B*         1.09         B         0.10         20         0.57         5.8         No           B*         1.09         B         0.10         20         0.79         5.05         Yes	Test         Nacelle         Fire         Fuel         Flow         Flow         Flow         Fire           (1)         1bs/sec         (3)         8a1/min         Flow         Fress.         Rate         Duration         Extinguished           B*         (2)         (3)         1.64         3.05         Yes           B*         0.10         20         1.64         3.05         Yes           B*         0.10         20         1.61         2.95         Yes           B*         0.10         20         1.61         2.95         Yes           B*         0.10         20         0.51         5.35         No           B*         0.10         20         0.51         5.35         No           B*         0.10         20         0.57         5.8         No           B*         0.10         20         0.53         5.2         No           B*         0.10         20         0.53         5.05         Yes           B*         0.10         20         0.79         5.05         Yes	Test No.         Rest Note elle (1)         Fire (1) <td>Test No.         Test No.         Nacelle Air Flow 10x Acation (1)         Fire Incompleted (1)         Fire Incompleted (2)         Fire Incompleted (3)         Fire Incompleted (3)&lt;</td> <td>Test No.         Aiselle (1)         Fire (1)         Fire (2)         Fire (3)         Fire (3)</td> <td>Test No.         Nacelle (1)</td> <td>Test No.         Nacelle (10)         Fire Indication (10)         Fire Indicated (10)</td> <td>Test No.         Sequence of Asia Period         Fire Incompany         Fire</td>	Test No.         Test No.         Nacelle Air Flow 10x Acation (1)         Fire Incompleted (1)         Fire Incompleted (2)         Fire Incompleted (3)         Fire Incompleted (3)<	Test No.         Aiselle (1)         Fire (1)         Fire (2)         Fire (3)         Fire (3)	Test No.         Nacelle (1)	Test No.         Nacelle (10)         Fire Indication (10)         Fire Indicated (10)	Test No.         Sequence of Asia Period         Fire Incompany         Fire

(Continued)

Fire	sec	2,55	1
Fire	1	Yes	No
LN2 Flow Duration	sec	7.3	7.25
LN <sub>2</sub> Flow Rate	lbs/sec	0.62	0.14
Fuel Flow Press.	psig	20	20
Fuel	gal/min	0.10	0.103
Fire Location	(3)	Д	Ą
Nacelle Air Flow	lbs/sec (2)	0.37	0.37
Test Sequence	(1)	O	O
Test No.		33.	34.

Test Sequence described in Table

. Average Nacelle Bleed Airflow During  ${\tt LN}_2$  Discharge

Fire location "A" is located 4 inches forward of firewall at 4 o'clock. Nozzle directed to spray forward,  $5^{\rm O}$  up and  $5^{\rm O}$  to the right in a horizontal plane.

Fire location "B" is located 7-3/4 inches forward for firewall at 4 o'clock. Nozzle directed to spray forward and 5 to the right in a horizontal plane.

Test conducted using 1 pound of  $\mathrm{CB}_{\mathrm{R}}\mathrm{F}_3$ . Fire extinguished.

5. Test conducted using 1 pound of  $\mathrm{CB}_\mathrm{R}F_3$ . Fire extinguished.

 $\mathbf{B}^{\star}$  LN<sub>2</sub> flow duration less than 10 seconds. Note actual time.

## TABLE II

### SCHEDULE OF EVENTS

## Schedule "A"

0	Initiate Test Sequence
+5	Ignition On
+10	Fuel On
+20	Chop or Abort
+30	LN <sub>2</sub> On & Ignition Off
+45	LN <sub>2</sub> Off - CO <sub>2</sub> On (if required)
	Schedule "B"
0	Initiate Test Sequence
+10	Ignitor On
+15	Fuel On
+20	Chop
+30	LN <sub>2</sub> On
+40	LN <sub>2</sub> Off - CO <sub>2</sub> On (if required)
	Schedule "C"
0	Initiate Test Sequence
+10	Ignitor On
+15	Fuel On
<b>⊦1</b> 7	Chop
<b>⊦</b> 30	LN <sub>2</sub> On & Ignition Off
-38	LN <sub>2</sub> Off & CO <sub>2</sub> On (if required)

TABLE III
DATA SUMMARY

						Load	
Run No.	Time	т1	P <sub>1</sub>	T <sub>2</sub>	P <sub>2</sub>	Cell	т3
	(1)	(°F)	(psig)	(°F)	(psig)	(1b)(2)	(3)
1.	0	+18	111	+74	-0.5	321.5	
	3.0	-304	82	-300	72.0	313.5	
	15.7	-299	82	-301	67.5	271.0	
	0	-3	109	+76	-1.0	226.0	
2.	3.0	-288	98	-292	86.5	218.0	
	10.5	-290	79	-300	62.5	200.0	
	0	-8	127	+73	-1.5	304.5	
3.	3.0	-285	108	-303	63.0	301.0	
	10.0	-292	89	-313	36.5	294.0	
	0	-63	118	+79	-1.0	291.0	
4.	3.0	-119	117	+47	3.5	290.5	
	9.35	-295	115	-59	1.5	289.0	
	0	-64	120	+79	-1.0	286.0	
5.	3.0	-190	117	-44	14.0	285.0	
	10.0	<b>-30</b> 5	109	-336	10.0	282.5	
	0	-77	109	+62	-0.5	277.5	
6.	3.0	-300	114	-312	43.5	271.5	
	12.9	-310	114	-321	35.0	264.5	
	0	-50	126	+9	-1.5	241.5	
7.	3.0	-273	117	+7	46.0	238.0	
	9.8	NA	108	+9	34.0	231.5	
	0	-32	118	+3	-2.0	227.5	
8.	3.0	<b>-2</b> 85	113	+3	36.5	224.5	
	10.1	-291	108	+3	27.0	218.0	
	0	-46	119	+3	2.0	218.5	
9.	3.0	-276	114	+3	38.0	215.5	
	8.3	-288	107	+3	80.5	210.5	
	0	-36	116	+3	2.0	210.5	
10.	3.0	-286	111	+3	36.0	207.5	
	7.7	-299	108	+3	28.0	203.0	

TABLE III
DATA SUMMARY (Continued)

Run No.	Ti	me	т1	P <sub>1</sub>	т2	P <sub>2</sub>	Load Cell	т3
	· ,	1)	(°F)	(psig)	(°F)	(psig)	(1b)(2)	(°F)(3)
11.			-45	117	+3	2.0	204.5	
	3	.0	-272	113	+3	37.5	201.0	
	9	.9	-286	106	+3	26.0	196.0	
10								
12.	0		-59	126	+36	2.5	198.0	
			-283	122	<b>-28</b> 8	43.5	194.5	
	9	.25	-297	115	-310	30.0	188.5	
1.0	0		-11	143	+35	2.0	190.0	
13.			-287	115	-300	37.0	187.0	
	10	.85	-297	108	-314	25.0	180.0	
1/	0		-45	117	+35	2.0	181.5	
14.			-288	113	-294	37.5	179.0	
	11	.15	-299	106	-314	26.0	171.5	
	0		-20	142	+32	2.0	172.0	
15.			-298	131	-292	79.5	159.0	
	5	.1 .	-299	126	<b>-29</b> 6	72.5	161.0	
16	0		+57	126	+60	2.0	156.5	
16.			285	116	-300	78.0	152.5	
	4	.8 -	· <b>2</b> 86	109	-303	72.5	149.0	
17	0		+34	124	+52	2.0	149.0	
17.			299	115	-300	74.5	144.0	
	3,	.05 -	<b>29</b> 8	115	-300	74.0	144.0	
10	0		+54	124	+62	2.5	145.0	
18.			296	118	-303	59.5	142.0	
	5.	.0 -	<b>29</b> 8	126	<b>-30</b> 5	55.0	139.5	
10	0		+37	123	+53	2.0	140.5	
19.	3.		297	117	-202	58.5	136.0	
	5.	05 -	301	115	<b>-30</b> 5	54.0	133.5	
20	0		+25	123	+49	1.5	135.5	
20.	2.	95 -	330	116	-303	57.5	130.5	

TABLE III
DATA SUMMARY (Continued)

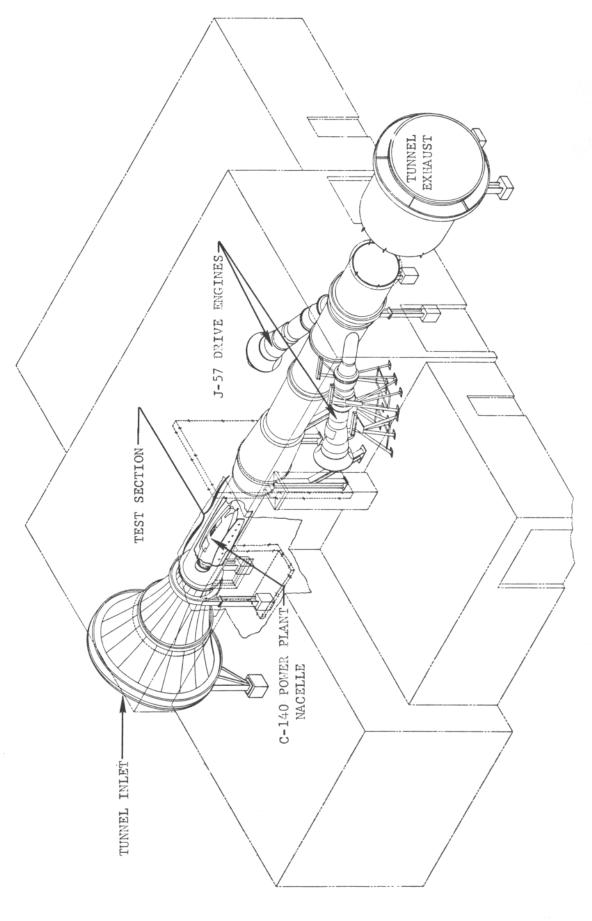
						Load	
Run No.	Time	T <sub>1</sub>	P <sub>1</sub>	T <sub>2</sub>	P <sub>2</sub>	Cel1	Т3
	(1)	(oF)	(psig)	(oF)	(psig)	(1b)(2)	(°F)(3)
	0	+57	129	+64	1.0	110.5	+166
21.	3.0	-254	127	-89	16.5	109.0	+137
	5.35	-293	126	-264	16.0	108.5	+110
	0	+27	128	+55	0.0	109.0	+164
22.	3.0	<b>-2</b> 85	125	-192	21.0	107.0	+125
	5.8	-293	124	-327	20.5	106.0	<b>+9</b> 5
	0	+40	126	+63	-0.5	107.0	NA(4)
23.	3.0	-279	123	-131	20.5	100.0	+212
	5.2	-289	122	-304	17.5	104.5	+160
24.	0	+21	125	+49	-0.5	104.0	
	3.0	-287	121	-302	17.0	102.5	+149
	5.05	-291	120	-250	12.0	100.0	+101
	0		NA	NA	NA	NA	+166
25.	3.0	NA	NA	NA	NA	NA	+137
	0	NA	NA	NA	NA	NA	+137
26.	0.25	NA	NA	NA	NA	NA	
	3.0	NA	NA	NA	NA	NA	+130
	0	+61	135	+67	0.0	242.5	
27.	3.0	-132	136	+40	3.0		
	7.2	-291	131	-26	3.0	241.0	
	0	+36	136	+69	-0.5	237.5	+291
28.	3.0	-269	131	-122	21.0		+185
	7.3	-302	129	<b>-33</b> 5	17.0	<b>233.</b> 5	+106
	0	-66	134	+53	-0.5	234.5	
29.	3.0	-267	130	-132	18.5		
	7.3	-302	129	-339	15.0	<b>230.</b> 5	+266
	0	+36	135	+57	-0.5	230.5	
30.	3.0	-294	127	-304	47.0		+305
	7.3	-302	121	-321	36.5	220.5	+130

TABLE III DATA SUMMARY (Continued)

						Load	
Run No.	Time	T <sub>1</sub>	P <sub>1</sub>	T <sub>2</sub>	P <sub>2</sub>	Cell	<sup>T</sup> 3
	(1)	(°F)	(psig)	(OF)	(psig)	(1b)(2)	(°F) (3)
	0	+16	132	+34	-0.5	223.0	
31.	3.0	-293	126	-303	36.5		+315
	7.45	-298	128	-321	30.0	216.0	+136
	0	+53	138	+34	-0.5	218.5	
32.	3.0	-286	133	-303	34.5		
	7.0	-295	128	-321	27.5	<b>212.</b> 5	+230
	0	-40	132	+35	0.0	207.5	
33.	3.0	-230	127	-127	16.0		
	7.3	-295	126	-108	13.0	204.5	+334
	0		130		-0.5	204.5	
34.	3.0		129		6.5		
	7.25		128		6.5	203.5	

### NOTES:

- Time in seconds after LN<sub>2</sub> flow was initated.
   Quantity of LN<sub>2</sub> remaining in Dewar.
   Nacelle ambient temperature measured at Nacelle Station 104, at 3:30 o'clock position.
- (4) NA = Not Applicable.



FIVE-FOOT FIRE TEST FACILITY

FIGURE 1

1-17

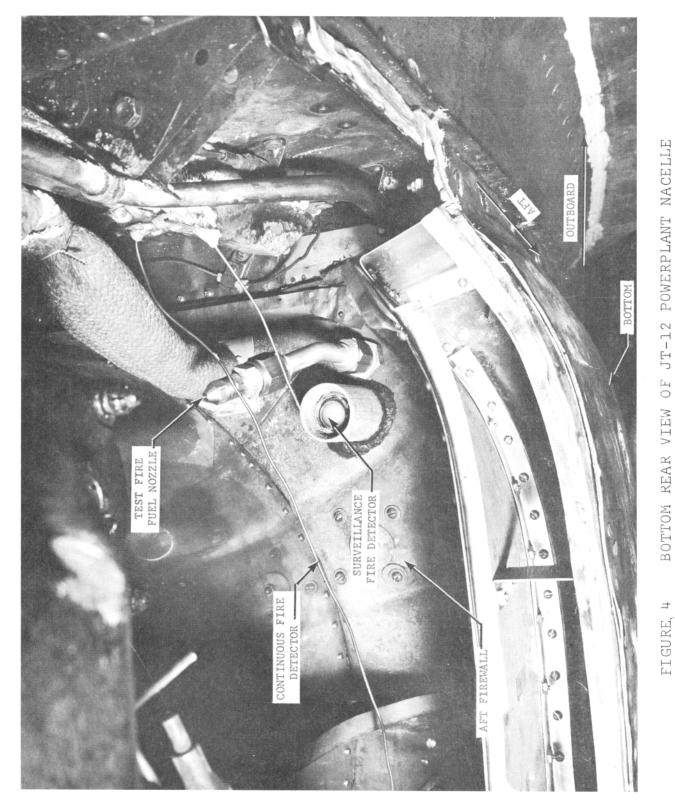


FIGURE 2 JT-12 POWERPLANT AND NACELLE SHOWING LN<sub>2</sub> DISCHARGE SYSTEM

WORK AREA SHOWING LN2 CONTAINER AND ASSOCIATED COMPONENTS

 $^{\circ}$ 

FIGURE



BOTTOM REAR VIEW OF JT-12 POWERPLANT NACELLE SHOWING FUEL NOZZLES AND FIRE SENSORS

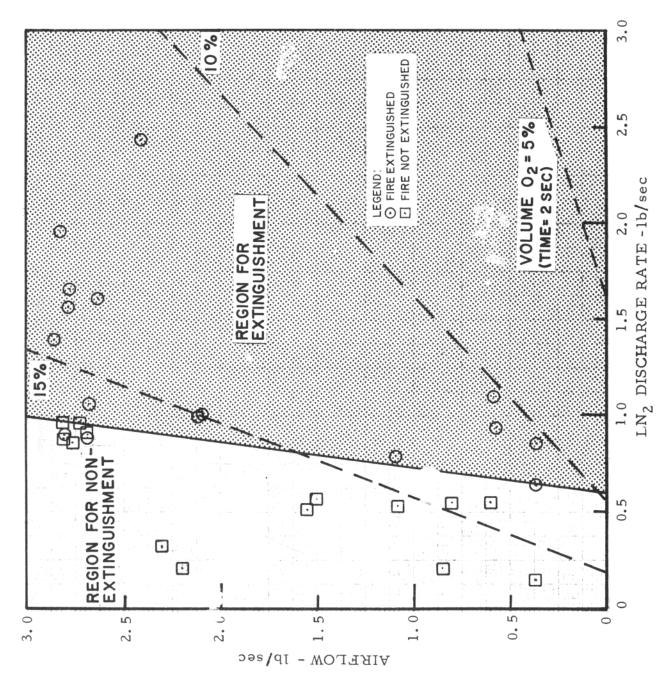
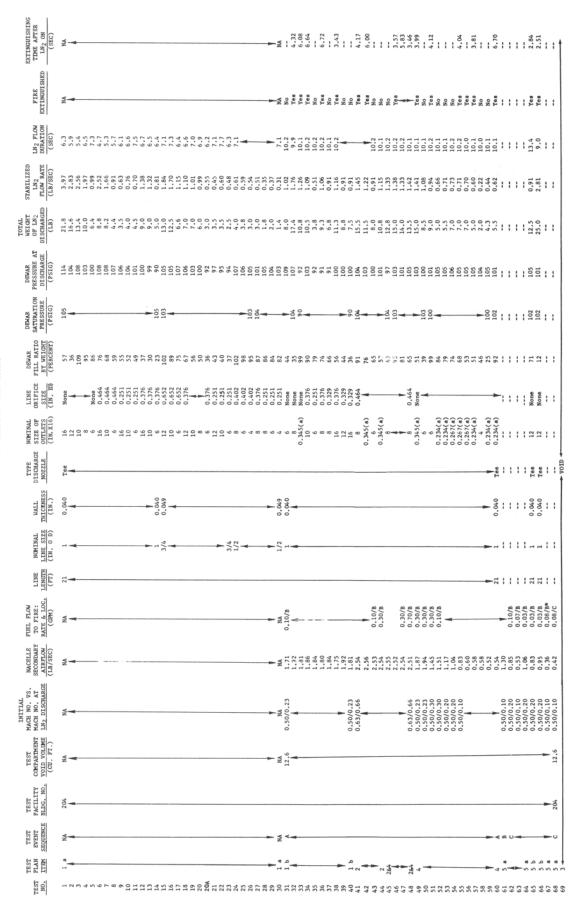


FIG. 5 LN<sub>2</sub> DISCHARGE REQUIREMENTS FOR FIRE EXTINGUISHMENT

# APPENDIX B SUMMARY OF TEST CONDITIONS AND TEST RESULTS



EXTINGUISHING TIME AFTER LN2 ON	(SEC)	NA 1.50 1.50 1.70 1.70 1.70 1.70 1.70 1.70 1.70 1.7	
FIRE EXTINGUISHED	₹	Y Y & B B B B B B B B B B B B B B B B B	
LN2 FLOW DURATION	(280) (10.9) (10	6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	
45	(18/SEC) 1,70 2,22 1,60 1,44 1,28 2,20 2,20 2,10 2,10 1,132 1,22 1,10 1,122 1,10 1,10 1,10 1,00 1,0	0.248 0.248 1.466 1.466 1.121 1.136 0.256(e) 0.256(e) 0.256(e) 0.126(e) 0.1	
I W W	(18) 25.5 6.6 6.7 2.7 2.8 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.6 6.7 2.5 6.7	2 2 1 1 1 1 1 1 1 1 2 1 2 1 1 1 1 2 1 2	
DEWAR PRESSURE AT DISCHARGE	(FSIG)  69  69  60  100  100  100  100  100  1	4 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
NO. SI	(PSIG) 70 70 70 70 70 70 70 70 70 70 70 70 70	100 100 100 100 100 100 100 100 100 100	
0.641	(PERFORM)  24  25  26  27  28  28  28  28  28  28  28  28  28	233 244 252 253 253 253 253 253 253 253 253 253	
	None None	None 0.464 0.464 0.464 0.464 0.464 0.464 0.464 0.464 0.464 0.464	
NOMINAL SIZE OF OUTLETS	(78.X15) 16 16 0.458(b) 0.645(b)	0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
TYPE DISCHARGE NOZZLE	0E(e)  0E(e)  1	0 H 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	
S	0.065 0.065	0.035	
	(IN. 0 p)	172	
<b>2</b> 1	32.5	33.5	
FUEL FLOW TO FIRE: RATE & LOC.	S N N	0.46/B	
NACELLE SECONDARY AIRFLON	(18/SEC)	N A A A A A A A A A A A A A A A A A A A	
INITIAL MACH NO. VS. MACH NO. AT LN2 DISCHARGE	ž <del>-</del>	NA N	
TEST COMPARTMENT VOID VOID VOID	(GU. PT.)	∑ Z	
TEST FACILITY BLDG, NO.	200	50 €	
TEST EVENT SEQUENCE	V V	× × × × × × × × × × × × × × × × × × ×	
TEST	0.	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	
TEST NO.	208 209 211 210 210 211 211 211 211 211 222 222	233 5 23 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	

NOTES: (a) Inside diameter in inches
(b) Total Outler Area in fig.
(c) Rates are average Lby flow rates
(d) PT Perforated Tube
(e) Oge Open End
(f) PN = POS Nozzle

APPENDIX C
TEST EVENT SCHEDULES

1 - 1

### TEST EVENT SCHEDULE A

TIME (sec)	EVENT
0	Stabilize Tunnel Velocity and JT-12 Power
10	Spark Ignition On
15	Spray Release Fuel
20	Retard JT-12 to Cutoff
25	Ignition Off
30	Initiate LN <sub>2</sub> Discharge
40	Fuel Spray Off Terminate LN <sub>2</sub> Discharge CJ <sub>2</sub> if Required

## TEST EVENT SCHEDULE B

TIME(sec)	EVENT
0	Stabilize Tunnel Velocity and Jt-12 Power
15	Spark Ignition On
20	Fuel Spray On
25	Spark Ignition Off
50	Retard JT-12 to Cutoff
60	Fuel Spray Off Initiate LN2 Discharge if Specified
90*	Terminate LN <sub>2</sub> Discharge CO <sub>2</sub> if Required

<sup>\*</sup> Approximate Time. Actual Termination to occur when four thermocouples in fire indicate  $\leq$  500°F.

## TEST EVENT SCHEDULE C

TIME (sec)	EVENT
0	Stabilize Tunnel Air Velocity and JT-12 Power
15	Spark Ignition On
20	Fuel Spray On
25	Spark Ignition Off
20	Chop JT-12 When Fire Detector Alarms
60	Fuel Spray Off Initiate LN2 Discharge if Specified
90*	Terminate LN <sub>2</sub> Discharge CO <sub>2</sub> if Required

<sup>\*</sup> Approximate time. Actual termination to occur when four thermocouples in fire indicate  $\leq$  500°F.

## TEST EVENT SCHEDULE D

TIME (	sec)		EVENT
0		Start Fan	
15		Spark Ignition	On
18		Fuel Spray On	
25		Spark Ignition	Off
40		LN <sub>2</sub> On	
50		LN <sub>2</sub> Off	
55		Fuel Spray Off CO <sub>2</sub> if Require	

## TEST EVENT SCHEDULE D1

TIME (sec)	EVENT
0	Fan On
18	Spark Ignition and Fuel Spray On
20	Spark Ignition Off
40	LN <sub>2</sub> On
50	LN <sub>2</sub> Off
5 5	Fuel Spray Off CO2 if Required
	TEST EVENT SCHEDULE D2
TIME (sec)	EVENT
0	Fan On
15	Spark Ignition and Fuel Spray On
15	Spark Ignition and Fuel Spray On Spark Ignition Off
18	Spark Ignition Off

## TEST EVENT SCHEDULE E

TIME (sec)	EVENT
0	Event Recorder and Oscillograph On
5 10 10	Spark Ignition On Fuel Spray On Fuel Spray On
15	Chop JT-12 Spark Ignition Off
25	Initiate LN2 Discharge
45	Fuel Spray Off Terminate LN2 Discharge CO2 if Required

## TEST EVENT SCHEDULE F

TIME (sec)	EVENT		
0	Event Recorder On		
5	Fuel Flow On		
10	Oscillograph On		
15	LN <sub>2</sub> On		
5 5	LN2 Off		
90	Fuel Flow Off		
105	Recorders Off		

## APPENDIX D

NITROGEN FLOW RATE CALIBRATIONS AS A FUNCTION OF NOZZLE, ORIFICE, AND TUBE SIZE

					. 4
					*
					•
					w

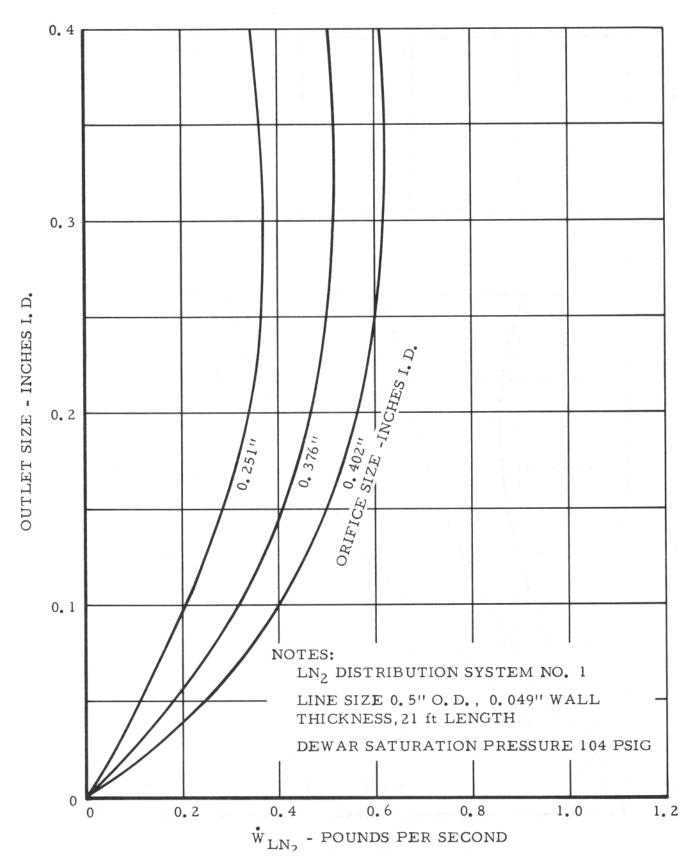


FIGURE 4-1 - NITROGEN FLOW RATE CALIBRATION FOR 21 FEET OF 1/2-INCH TUBING

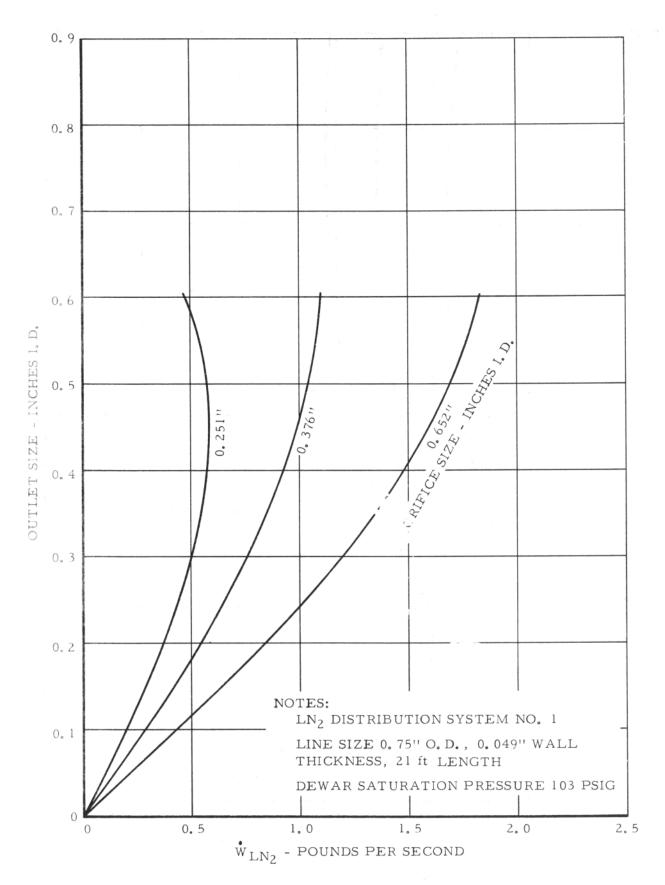


FIGURE 4-2 - NITROGEN FLOW RATE CALIBRATION FOR 21 FEET OF 3/4-INCH TUBING

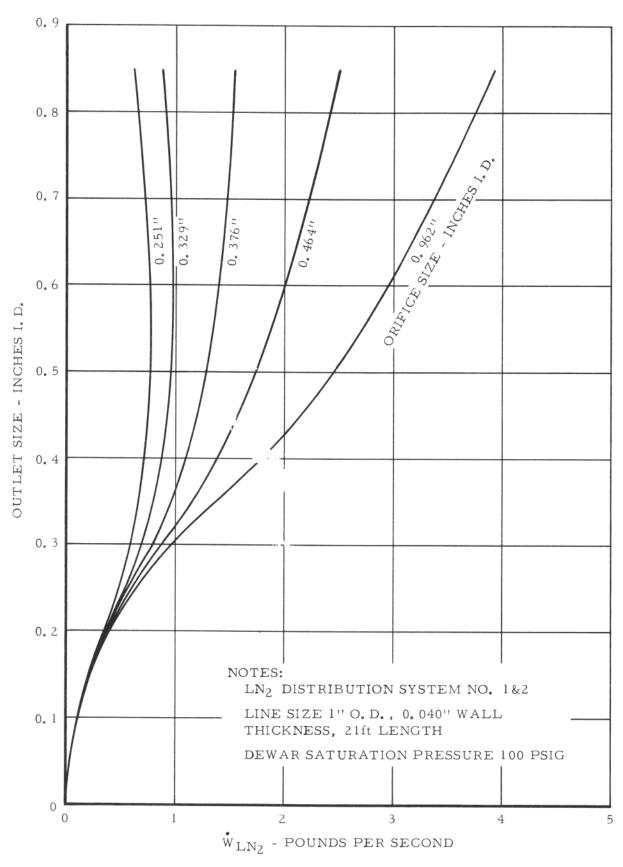


FIGURE 4-3 - NITROGEN FLOW RATE CALIBRATION FOR 21 FEET OF 1-INCH TUBING

#### APPENDIX E

LOCATION AND DESCRIPTION OF FUEL-TO-FIRE NOZZLES USED IN THE JET ENGINE TEST INSTALLATION

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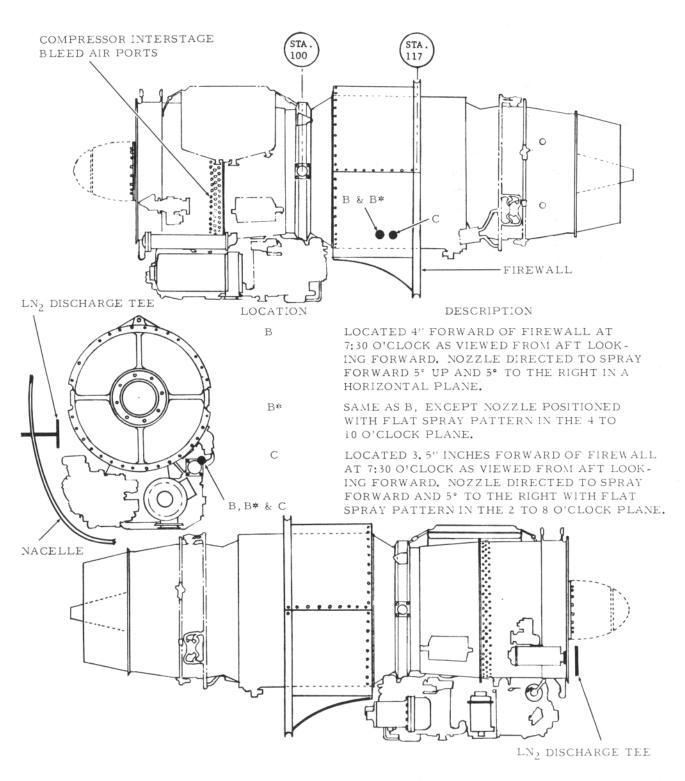


FIGURE 5-1 LOCATION OF FUEL-TO-FIRE NOZZLES AND LN2 DISCHARGE NOZZLE IN TEST ENGINE INSTALLATION

TABLE 5-1 -FUEL SPRAY NOZZLE DESCRIPTION

USED ON TESTS	44 thru 51 69 thru 86 96 THRU 169 191 THRU 255	64 THAU 68	31 THRU 43 52 THRU 62 87 THRU 95 189 and 190	63 170 THRU 188
ATOMIZATION	COARSE	FINE	EXTRA FINE	EXTRA FINE
SPRAY ANGLE	300	o O †	81°	62°
SPRAY PATTERN	FULL CONE	FLAT, FAN TYPE	HOLLOW CONE	HOLLON CONE
NOZZLE TYPE	1/8GG-3007 1/8GG-3004 1/8GG-3014	T-400050 T-400025	54-21	68-21

#### APPENDIX F

TABULAR PRESENTATION OF DISTRIBUTION SYSTEM PRESSURES, TEMPERATURES, AND NITROGEN FLOWS FOR AN 80-FOOT-LONG PRESSUR-IZED LINE, AND 80- and 21-FOOT UNPRESSURIZED LINES\*

\*See Figure 6b for location of pressure and temperature probes.

#### TEST 46

Length: 21-ft Unpressurized

Valve: At Dewar

Flow Rate: 1.38 lbs/sec Nozzle: AN-834-16 tee

w/AN-894-8 reducer bushings

Fill Ratio: 95% @ 103 psig Orifice: 0.464 inch

Time After LN <sub>2</sub> Discharged	Pl	T <sub>1</sub>	P <sub>2</sub>	T <sub>2</sub>	WLN2	
(sec)	(psig)	(°F)	(psig)	(°F)	(lbs)	
0 1 2 3 4 5 6 7 8 9	Vacuum 84 87 89 87 85 83 82 82 80	+68 -20 -100 -165 -211 -277 -281 -279 -279 -282 -282	Vacuum 80 80 81 86 80 78 77 75 74	+68 -34 -192 -281 -289 -295 -296 -296 -298 -298	94.0 93.2 92.0 90.8 89.4 88.6 87.2 85.8 84.4 82.6 81.4	

### TEST 193

Length: 21-ft Unpressurized Valve: At Dewar

Flow Rate: 3.32 lbs/sec Nozzle: AN-834-16 tee

w/AN-894-12 reducer bushings

Fill Ratio: 102% @110 psig Orifice: None

After LN <sub>2</sub> Discharged	Pl	$T_1$	P <sub>2</sub>	T <sub>2</sub>	WLN2	
(sec)	(psig)	(o F.)	(psig)	(°F)	(1bs)	
0		+54		+54	102	
1	49.9	<b>-</b> 90	39.8	-247	99.2	
2	52.4	-214	42.3	-310	96.0	
3	51.4	-325	41.0	-312	92.7	
4	50.9	-328	40.0	-311	89.8	
5	50.2	-330	38.0	-312	86.6	
6	48.9	-328	36.0	-314	83.2	
7	48.1	-328	35.5	-313	80.2	
8	47.6	-328	35.0	-312	77.2	
9	46.4	-330	34.0	-314	73.6	
10	45.4	-331	33.5	-314	70.6	

TEST 246

Length: 80-ft Pressurized Valve: Near Discharge Nozzle Fill Ratio: 95% @ 100 psig

Flow Rate: 0.81 lbs/sec Nozzle: AN-834-16 tee Orifice: None

Time						
After						
LN <sub>2</sub>	P <sub>1</sub>	$T_1$	P <sub>2</sub>	T <sub>2</sub>	WLN2	
Discharged			_	-	1117	
(sec)	(psig)	(°F)	(psig)	(°F)	(lbs)	
0	104	68	AMB	88	94.2	
1	108	-115	18	74	93.6	
2	95	-305	14	67	93.0	
3	104	-305	20	60	92.4	
4	101	-305	18	53	91.4	
5	103	- 305	21	46	90.8	
6	102	-305	22	30	90.4	
7	100	-305	21	16	89.4	
8	100	-305	22	2	88.4	
9	99	-305	25	-30	87.6	
10	98	-305	26	-84	86.8	
11	95	-306	20	-126	86.0	

TEST 247

Length: 80-ft Pressurized Flow Rate: 1.33 lbs/se Valve: Near Discharge Nozzle Nozzle: AN-834-16 tee Fill Ratio: 96% @ 100 psig Orifice: None

Flow Rate: 1.33 lbs/sec

Time After						
LN <sub>2</sub> Discharged	P <sub>1</sub>	Tl	P <sub>2</sub>	T <sub>2</sub>	W <sub>LN2</sub>	
(sec)	(psig)	(°F)	(psig)	(°F)	(lbs)	
0	102	74	AMB	82	95.6	
1	104	-116	20	74	94.2	
3	95	-300	16	67	94.2	
3	103	-301	22	60	93.6	
4	99	-301	20	53	93.0	
5	101	-301	24	46	92.4	
6	101	-301	25	28	92.0	
7	98	-301	24	12	90.2	
8	98	-301	26	- 7	89.2	
9	98	-301	29	-55	88.4	
10	9 7	-301	30	-119	87.6	
11	94	-302	30	-191	86.2	
12	92	-302	29	-307	84.6	
13	9 2	-304	27	-317	83.0	
14	91	-304	27	-321	81.6	
15	91	-304	26	-322	79.8	

### TEST 248

Length: 80-ft Pressurized Length: 80-ft Pressurized Flow Rate: 1.14 lbs/sec Valve: Near Discharge Nozzle Nozzle: AN-834-16 tee

w/AN-894-12 reducer bushings

Fill Ratio: 102% @ 105 psig Orifice: None

						Time
						After
	WLN <sub>2</sub>	T <sub>2</sub>	P <sub>2</sub>	$\mathtt{T}_{\mathtt{l}}$	P <sub>1</sub>	LN <sub>2</sub>
	21. 2		_		ed _	Discharge
	(lbs)	 (°F)	(psig)	(°F)	(psig)	(sec)
	99.0	91	AMB	67	107	0
	98.8	82	34	-117	109	1
	98.4	74	29	-293	100	2
	98.0	74	3 7	-293	107	3
	97.6	67	35	-295	104	4
	97.0	60	37	-295	104	
	96.2	39	41	-296	104	5
	95.2	22	41	-296	103	7
	94.4	2	40	-296	101	8
	94.2	-29	43	-296	99	9
	93.2	-93	44	-296	98	10
	91.6	-153	44	-296	97	11
	90.0	-250	45	-296	96	12
	88.4	-307	44	-296	96	13
	87.0	- 30 7	44	-296	95	14
	85.4	- 30 7	42	-296	95	15
	84.0	-307	41	-296	94	16
	82.4 80.1 79.8	-307 -307 -319	40 38 37	-296 -296 -305	9 4 9 4 9 3	17 18 19

TEST 249

Length: 80-ft Pressurized Valve: Near Discharge Nozzle Flow Rate: 1.25 lbs/sec
Nozzle: AN-834-16 tee
w/AN-894-12 reducer bushings

Fill Ratio: 99% @ 110 psig Orifice: None

Time						
After						
LN <sub>2</sub>	Pl	$T_1$	P <sub>2</sub>	T <sub>2</sub>	$W_{LN_2}$	
Discharged		1. 1 <u></u>			2112	
(sec)	(psig)	(°F)	(psig)	(°F)	(lbs)	
0	108	11.2	AMD	0.0	0.0	
1		43	AMB	86	99.6	
1 2	108	-134	33	81	98.4	
3	100	-302	30	7.5	97.2	
	109	-302	38	72	96.4	
4	103	-305	34	61	95.4	
5	107	-305	40	54	95.0	
5 6 7	105	-305	42	35	94.2	**
	103	-305	40	14	93.4	
8	104	- 305	43	- 7	92.2	
9	103	-305	46	-47	91.6	
10	102	-305	45	-100	90.2	
11	100	-305	45	-161	88.8	
12	97	-305	45	-265	87.6	
13	97	-305	44	-315	86.2	
14	96	-305	43	-319	84.6	
15	96	-305	41	-320	82.8	
16	96	-305	41	-320	81.6	
17	95	-305	39			
18	94	-305	38	-320	80.2	
19	94	-305		-321	78.4	
			37	-322	77.2	
20	93	-306	35	-323	74.6	

TEST 250

Length: 80-ft pressurized Valve: Near Discharge Nozzle

Flow Rate: 1.19 lbs/sec

Nozzle: AN-834-16 tee
w/AN-894-12 reducer bushings
Orifice: None

Fill Ratio: 63% @105 psig

Time After LN <sub>2</sub> Discharged (sec)	Pl (psig)	T <sub>1</sub>	P <sub>2</sub>	T <sub>2</sub>	W <sub>LN<sub>2</sub></sub>	
0 1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20	110 111 103 108 105 106 106 104 102 101 99 96 95 93 93 93 92 90 90 89	53 -147 -300 -300 -300 -300 -300 -300 -300 -30	AMB 36 31 35 39 22 41 43 44 44 42 41 41 41 41 41 41 41 41 41 41 41 41 41	74 67 60 60 46 39 28 9 -13 -51 -123 -192 -293 -313 -313 -313 -313 -313	62.6 62.4 661.8 660.2 598.8 577.6 553.8 555.3 555.3 555.3 555.4 487.6 442.0 441.0	

TEST 251

Length: 80-ft Pressurized Valve: Near Discharge Nozzle

Flow Rate: 0.73 lbs/sec
Nozzle: AN-834-16 tee
w/AN-894-8 reducer bushings
Orifice: None

Fill Ratio: 99% @ 105 psig

Time After						
LN <sub>2</sub> Discharged	Pl	Tl	P <sub>2</sub>	T <sub>2</sub>	$W_{LN_2}$	
(sec)	(psig)	(°F)	(psig)	(°F)	(lbs)	
0	109	74	2	91	97.6	
1	115	-107	75	91	97.0	
1 2 3	98	-249	6 2	82	96.6	
	113	-297	77	82	96.2	
4	107	-301	73	82	95.6	
5	108	-300	73	74	95.0	
6	115	-300	81	74	94.2	
7	105	-300	75	60	93.4	
8	99	-300	67	53	92.6	
	109	-300	80	46	92.0	
10	107	-300	79	29	91.4	
11	105	-300	77	16	91.0	
12	105	-300	75	3	90.2	
13	105	-300	79	-20	90.0	
14	105	-300	8 3	-63	89.2	
15	104	- 300	81	-103	88.8	
16	102	-300	78	-138	88.4	
17	101	-300	77	-163	87.2	
18	101	-300	79	-230	86.2	
19	101	-300	79	-298	85.0	
20	101	-300	79	-300	83.8	

### TESTS 252

Length: 80-ft Unpressurized Valve: At Dewar

Flow Rate: 1.24 lbs/sec
Nozzle: AN-834-16 tee
w/AN-894-12 reducer bushings

Fill Ratio: 97% @ 105 psig Orifice: None

Time						
After			<u>_</u>			
LN <sub>2</sub>	$P_1$	$\mathtt{T}_{\mathtt{l}}$	P <sub>2</sub>	T <sub>2</sub>	$W_{LN_2}$	
Discharged		(0.5)		/ > = \	The state of the s	
(sec)	(psig)	(°F)	(psig)	(°F)	(lbs)	
0	8	91	AMB	96	96.4	
1	114		28			
		-132		96	95.8	
2	121	-292	3 3	91	94.8	
3	123	-293	39	8 2	94.6	
4	120	-295	38	74	93.2	
5	120	-295	39	60	92.0	
6 7	121	-295	44	46	91.0	
7	118	-295	42	26	90.0	
8	116	-295	42	8	89.2	
9	116	-295	46	-25	88.2	
10	115	-295	49	-90	87.6	
11	113	-295	47	-163	86.6	
12	111	-295	47	-270	84.6	
13	111	-295	46	-304	83.0	
14	111	-295	45	-307	81.6	
15	110	-295	42	-309	80.0	
16	109	-295	42	-309	78.4	
17	109	- 295 - 295				
			41	- 30 9	77.0	
18	108	-295	38	-309	75.2	
19	107	-295	38	-309	73.6	
20	107	-295	38	-309	72.4	

#### APPENDIX G

TABULATION OF NITROGEN FLOW PARAMETERS FOR VARIOUS SYSTEM COMPONENTS AND DISCHARGE NOZZLE CONFIGURATIONS\*

\*See Figure 22 for location of pressure and temperature probes.

Test No. 204

Time	Pl	Tı	Pa	Pb	Pc	Pd	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig	) (psig)	(pisg)	(°F)
1	84	NR*	76	71	68	62	30	NR
1 2 3	90		83	80	77	73	42	1417
3	90	1	81	52	7.5	70	44	Ť
4	87		78	7.5	7 2	6 7	42	
5	8.5		76	7 3	70	6 5	42	
6	84		7 5	7.2	69	6 4	39	
7	82		76	70	6 7	61	37	
8	81		7 4	69	64	60	33	1.00
9	8.0		7 4	6 9	6 3	58	32	
10	79		7 3	68	6 3	58	3 2	
11	78		7 3	68	6 3	58	3 3	
12	77		7.3	6.8	63	58	3 2	
13	77		7 2	68	63	58	3 2	
14	77		7 2	6.7	6 3	5 8	32	
15	7.6		71	66	61	57	31	
16	75		70	6 5	60	56	31	
17	74		70	6.5	60	5 6	30	
18	74		6 8	63	5 8	54	28	1
19	73	V	66	61	56	5 2	27	<b>Y</b>
20	71	NR	61	5 7	5 2	48	24	NR
2 2 2 2 2								

Test No. 205

Time (sec)	P <sub>1</sub> (psig)	T <sub>1</sub>	P <sub>a</sub> (psig)	P <sub>b</sub> (psig)	P <sub>c</sub> (psig)	P <sub>d</sub> (psig)	P <sub>2</sub> (psig)	T <sub>2</sub>
1 2 3 4 5 6 7 8 9 10 11 12 14 15 16 17 18 19	104 110 108 105 102 100 97 95 93 91 89 87 83 82 80 77 75 73	NR	96 109 98 99 99 99 99 99 88 97 66 67 66 64	95 96 97 87 88 88 88 76 76 66 66 59	83 93 89 88 79 75 74 71 86 53 75 54	88 88 88 77 74 77 69 66 66 64 19 23 55 55 55	41 53 53 50 34 42 41 40 39 38 36 35 32 29 28 26	NR
20	71	NR	6 2	5 7	5 2	49	25	NR

TEST No. 206

Time	Pl	Tl	Pa	Pb	Pc	Pd	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(pisg)	(°F)
1	100	7.00	0.5	0.0				
1	103	-138	95	88	85	80	47	-99
2	100	-273	9 4	91	8 8	8 3	50	-297
3	100	-292	91	86	8 3	78	48	-302
4	9 7	-294	8 7	83	79	73	47	-302
5	9 3	-295	8 3	80	76	71	45	-302
6	89	-296	8 0	76	72	6 7	41	- 30 2
7	86	-296	78	73	6.7	6.2	37	-303
8	83	-297	76	70	6.5	60	34	-305
9	80	-298	74	68	63	5.8	3 3	-305
10	77	-299	72	66	61	5 7	31	-305
11	7 5	-299	70	6 5	60	5.5	30	-306
12	73	-300	6 8	6 2	59	5.5	30	-306
13	6 2	-302	52	42	39	35	17	-312

TEST No. 207

Time	Pl	Tl	Pa	$P_{\mathbf{b}}$	Pc	Pd	$P_2$	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(pisg)	)°F)
1 2 3 4 5 6 7 8 9	71 74 74 73 71 70 68 68 67 66	-78 -246 -288 -297 -299 -299 -299 -299 -299	64 67 66 64 63 62 61 61 61	(psig) 64 67 65 63 62 61 60 59 58	(psig) 62 65 62 60 59 58 56 54 53	(psig) 58 62 59 56 54 53 52 51 50 49	(pisg) 30 35 34 35 34 32 30 27 26	-50 -297 -316 -315 -315 -315 -315 -315 -315
11 12 13	66 65 64	-299 -299 -302	6 0 6 0 6 0	5 8 5 8 5 7	5 3 5 2 5 2	49 48 48	2 5 2 5 2 5	-317 -317 -317
14 15 16	6 4 6 3 6 2	-300 -302 -302	6 0 5 9 5 8	5 7 5 6 5 5	5 2 5 1 5 1	48 47 47	2 5 2 5 2 4	-316 -317 -318

TEST No. 208'

Time (sec)	Pl (psig)	(°F)	Pa (psig)	Pb (psig)	P <sub>C</sub> (psig)	P <sub>d</sub> (psig)	P <sub>2</sub> (psig)	T <sub>2</sub>
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16	61 60 58 54 53 51 50 48 47 45	-70 -227 -286 -299 -303 -305 -305 -305 -305 -305 -307 -307 -307	54 55 55 55 55 55 55 55 55 55 55 55 55 5	5 7 2 1 9 7 6 5 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	51 54 49 46 44 44 41 39 39 37 36 36	48 51 474 45 43 41 39 38 37 36 35 35 34 33	23 28 25 26 25 24 24 22 20 19 17 16 15 15	-67 -247 -321 -319 -319 -319 -319 -319 -319 -319 -322 -322 -322 -322 -322

Time (sec)	P <sub>l</sub> (psig)	(°F)	P <sub>a</sub> (psig)	P <sub>b</sub> (psig)	P <sub>C</sub> (psig)	P <sub>d</sub> (psig)	P <sub>2</sub> (psig)	(°F)
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16	72 74 74 72 70 68 67 66 65 64 63 62 61 60	-93 -257 -292 -298 -300 -302 -302 -302 -302 -302 -302 -302	657642109999887655555555555555555555555555555555	6 7 5 3 1 0 9 8 6 7 6 6 5 5 4 3 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	62 65 60 57 56 57 55 52 55 50 49	59 65 55 55 55 55 55 55 55 55 55 55 55 55	3 35 35 35 36 30 28 22 55 25 25 25 25 24 23	-58 -305 -315 -315 -315 -317 -317 -317 -317 -317 -317 -317 -317

TEST No. 210

Time	Pl	Tl	Pa	Pb	Pc	$P_d$	P <sub>2</sub>	T2
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
		-						
0.7	43	-10	39	3 8	35	3 2	13	-46
1	46	-51	40	4 2	41	3 7	16	-46
1.5	45	-115	3 8	31	28	2.5	10	-63
2	48	-190	42	43	4 2	40	21	-154
3	49	-284	43	42	41	3.8	19	-324
4	48	- 30 3	41	41	40	3 7	20	-325
5	47	-309	41	40	38	35	20	-325
6	47	-312	40	39	37	3 5	20	-324
7	45	-312	39	38	36	33	19	-323
8	44	-312	3 7	3 7	3 5	3 2	18	-323
9	43	-312	37	3 7	35	31	18	-323
10	43	-314	37	3 7	34	31	17	-325
11	43	-314	3 7	36	34	31	16	-324
12	43	-314	3 7	36	34	31	15	-325
13	43	-314	3 7	36	33	31	13	-326
14	42	-314	37	36	33	31	13	-326
15	42	-314	37	36	33	31	13	-326
16	42	-314	37	36	3 3	31	13	-326
			2 : :			31		

Time	Pl	Tl	Pa	РЪ	$P_{\mathbf{G}}$	$P_d$	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
0.7 1 1.5	43 47 45 48	18 -62 -123 -218	39 40 37 42	28 40 30 44	38 36 34 41	3 4 3 3 2 4 4 0	27 14 2 19	-43 -48 -72 -209
3	48	-336	42	41	39	37	18	-325
4	47	-309	41	41	39	37	20	= 332
5	46	-316	40	40	38	3 5	19	-333
6	4.5	-318	39	39	36	34	19	-330
7	44	-319	38	38	35	3 3	19	-331
8 9	43 43	-320 -320	3 7 3 6	37 36	3 5 3 4	32	18 17	-331 -331
10	42	-320	3 7	37	34	31	16	-332
11	42	-320	3 7	36	3 3	31	15	-333
12	42	-320	3 7	36	3 3	31	14	-333
13	42	-320	3 7	36	3 3	30	13	-333
14	41	-323	3 7	36	32	30	12	-333
15	41	-320	37	36	32	3 0	13	-333
16	41	-320	36	36	32	30	13	-333

TEST No. 212

Time	Pl	Tl	Pa	$P_{\mathrm{b}}$	Pc	Pd	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
0.7	39	-30	35	38	35	34	15	- 7.7
1	42	-62	36	36	3.5	3 2	13	-80
1.5	41	-121	36	3 3	3.0	26	11	-93
2	41	-196	36	40	39	37	21	-149
3	36	-287	36	3 7	3.5	3 3	15	-318
4	36	-310	35	36	3 3	31	16	-335
5	35	-319	34	34	3 2	30	15	-336
6	34	-322	3 3	3 3	31	29	15	-334
7	32	-322	3 2	32	29	2.7	14	-335
8	32	-324	31	31	28	26	14	-335
9	31	-322	30	30	28	2.5	13	-335
10	30	-323	30	29	27	24	13	- 336
11	29	-324	29	29	26	23	11	-337
12	28	-324	29	28	2 5	23	10	-338
13	28	-324	29	28	26	23	9	-336
14	28	-324	28	27	24	22	8	-338
15	2.7	-324	28	27	24	22	7	-338
16	26	-324	28	2 7	23	21	7	-338

TEST No. 213

Time	Pl	Tl	Pa	$P_{b}$	Pc	$P_d$	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1	89	NR	79	73	6.8	63	32	NR
2	98	A	8 7	87	84	80	48	A
3	98	T	8 7	8 5	81	77	49	T
4	96		8 5	82	78	73	49	
5	93		8 2	80	76	71	48	
6	91		81	79	75	69	44	
7	90		81	77	71	66	41	
8	88		8 0	76	6 9	6.5	3 7	
	86		7 8	74	68	64	36	
10	85		7 7	7 3	6.8	6 3	36	
11	8 4		77	7 3	67	6.3	36	1
12	83		76	72	66	6 2	36	
13	8 2		7 5	7 2	66	62	36	
14	81		7 4	70	6.5	61	34	
15	80	A	7 3	6 9	6 4	60	3 3	A
16	7 8	ΝR	70	66	62	5 7	32	NR

TEST No. 214

Time	Pl	Tl	Pa	Pb	Pc	Pd	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1	91	-91	84	85	84	81	6 3	-43
1 2	102	-257	96	96	96	94	81	-263
3	103	-286	95	93	91	89	7.7	-300
4	101	-292	93	90	88	85	72	-300
5	99	-293	91	88	86	8.3	70	-301
6	96	-294	88	85	8 3	79	66	-302
7	94	-296	86	83	81	76	6 2	-304
8	92	-296	85	81	78	74	5 8	- 304
9	91	-296	85	79	74	70	5 3	- 305
10	88	-297	83	77	7 3	69	49	-306
11	8 7	-297	81	76	7 2	6 7	47	- 30 7
12	8.5	-297	8 0	75	71	66	46	-308
13	84	-297	7.9	74	7.0	6 5	44	-308
14	83	-297	7 8	7 3	6 9	6.5	4 3	-309
15	82	-297	77	73	6 9	64	43	-309
16	81	-297	76	7.2	6 8	6 3	4 2	-309

Time	Pl	Tl	Pa	Pb	Pc	Pd	P <sub>2</sub>	T2
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1	NR	-96	89	9 2	92	89	6 9	-49
2	<b>A</b>	-259	94	9 3	92	90	7 8	-261
3		-290	9 3	89	89	86	7 3	-301
4	1	-295	91	87	86	8 3	71	- 302
5		-297	88	85	8.3	80	6 7	-303
6		-297	86	82	80	76	64	-304
7		-298	83	80	78	73	60	-305
8		-298	82	78	75	71	56	-306
9		-299	81	7 5	71	67	51	-307
10	- A 1	-300	79	74	69	6.5	47	-308
11		-300	78	73	6.8	6.5	45	-309
12		-303	77	72	67	63	44	-309
13		-300	76	70	66	6 2	43	-310
14		-300	75	70	6.5	61	42	-310
15 .	₩	-300	73	69	64	60	41	-310
16 '	NR	-302	71	66	62	58	39	-311

TEST No. 216

				A CONTRACTOR OF THE SECOND				
Time	Pl	Tl	Pa	РЪ	Pc	Pd	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1	NR	-107	94	95	94	0.2	0.0	r 0
	1					92	69	-5,9
2	<b>A</b>	-264	96	9 3	9 3	91	80	-252
3		-291	9 4	91	90	8 8	74	-300
4		-295	91	88	87	84	71	-301
5		-298	8 8	8.5	8 3	80	68	-302
6		-298	85	81	79	75	63	-303
7		-300	81	78	76	71	59	-305
8		-300	79	75	73	6 9	54	-307
9		-300	77	72	68	6.5	49	-308
10		-300	7 5	70	6.5	62	45	-308
11		-302	7 3	6 9	64	61	43	-310
12		-303	71	6 7	63	5 9	41	-310
13		- 30 2	70	65	61	58	39	-309
14		-303	6.7	6 3	59	56	38	-311
15		-303	6 7	63	59	56	37	-311
16	<b>V</b>	-305	60	54	53	49	38	-311
17	NR	-306	45	39	3 7	35	27	-308

# TEST No. 216A

Time	Pl	Tl	Pa	Pb	Pc	Pd	P <sub>2</sub>	Τ2
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1	87	-86	80	76	73	70	5 3	-37
2	8.7	-251	81	8 0	80	77	6.5	-188
3	87	-288	80	8 0	79	77	67	-303
4	8.5	-296	78	77	72	73	62	-306
5	8.3	-299	76	74	7 3	70	60	-304
6	80	-299	73	71	70	66	56	-306
7	77	-302	70	6 8	66	62	5 2	-308
8	74	-302	68	66	6.5	60	49	-308
9	72	-303	67	64	6 2	5 8	4.5	- 30 9
10	71	-303	66	6 2	59	55	41	-310
11	69	-304	64	61	56	52	38	-312
12	68	-304	63	59	56	52	36	-312
13	66	-304	6 2	58	54	51	35	-312
14	65	-305	60	57	5 3	49	33	-312
				, , ,			0.0	312

TEST No. 217

Time	Pl	Tl	Pa	Pb	Pc	Pd	P <sub>2</sub>	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
0.7	NR	-42	64	5 8	6 0	56	46	-46
1		-92	65	5 8	6 0	57	48	-43
1.5		-179	64	5 9	6 4	60	50	-62
2		-255	71	7 4	7 3	71	61	-198
4 5		-293 -302 -303	71 69 69	69 69 67	6 8 6 8 6 6	65 65 63	5 3 5 6 5 4	- 306 - 309 - 309
6		- 306	67	66	6 3	61	51	-310
7		- 306	65	64	6 2	59	47	-311
8		- 307	64	63	6 0	57	46	-311
9		- 30 7	63	61	6 0	56	44	-312
10		- 30 7	63	61	6 0	56	42	-312
11		- 30 8	63	60	5 7	53	39	-313
12		-308	6 2	6 0	5 5	52	3 7	-314
13		-309	6 2	5 9	5 5	52	3 5	-314
14		-309	6 2	5 8	5 4	51	3 4	-315
15	NR	-308	61	58	54	51	3 4	-315

# TEST No. 217A

_Time	Pl	Tl	Pa	$P_{b}$	PG	$P_d$	P2	$T_2$	
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	
0.7	61	-36	58	5.8	5 7	5 3	43	6.6	
1	63	-83	56	53	51			-66	
1.5	64	-188	55			48	36	-50	
				50	53	49	43	-60	
2	6.8	-252	63	66	6 3	6 2	49	-209	
3	68	-288	6 3	64	64	6.2	49	-301	
4	6 8	-299	62	6 2	61	59	49	-310	
5	6 7	-303	61	60	6.0	58	48	-310	
6	6.5	-304	59	59	5.8	5.6	4.7	-311	
7	64	-306	58	5 7	56	5.3	44	-312	
8	62	-305	56	5.5	53	50	42	-312	
9	60	-305	54	54	5 2	49	39	-313	
10	6.0	-305	54	54	52	48	39	-313	
11	59	-307	54	5 3	51	48	36	-315	
12	59	-306	54	52	49	47	34		
								-315	
13	59	-306	54	5 2	48	4 5	3 2	-316	
14	5 8	-307	53	51	48	45	30	-317	

TEST No. 218

Time P <sub>1</sub> (sec) (psig	T <sub>1</sub> (°F) ( <sub>F</sub>	P <sub>a</sub> P <sub>b</sub> psig)	P <sub>c</sub> P <sub>d</sub> (psig)	P <sub>2</sub> (psig) (	T <sub>2</sub>
0.7 70 1 74 1.5 73 2 76 3 74 4 73 5 72 6 70 7 68 8 66 9 65 10 63 11 63 12 61 13 59 14 59	-33 -80 -172 -244 -288 -302 -305 -307 -308 -308 -309 -309 -309 -309 -309	68 70 68 63 67 59 71 72 69 67 68 67 66 65 65 65 61 59 60 58 59 57 58 55 57 54 56 53 55 52	68 65 61 57 57 53 72 70 67 66 66 64 64 61 62 59 60 57 58 54 57 52 55 51 52 49 49 46 49	50 - 43 - 60 - 53 - 54 - 52 - 49 - 47 - 44 - 43 - 39 - 36 - 34 - 32 - 31 -	34 42 53 165 292 312 313 314 317 316 318 318 318 320 320
15 57	-312	54 51	47 45	29 -	321

Time	Pl	Tl	Pa	Pb	Pc	Pd	P <sub>2</sub>	Т2	
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	
0.8 1.5 2 3 4 5 6 7 8 9 10 11 12 13 14 15	49 48 47 50 50 50 48 45 44 43 43 43	-21 -43 -127 -211 -273 -297 -307 -312 -314 -315 -316 -316 -316 -316 -316 -317	442 445 445 445 445 443 440 338 388 388 388 388 388 388	42 42 41 42 47 47 44 41 41 41 41 41 41 41 41 41	40 37 39 41 465 43 41 338 337 37 36	3347 3347 347 34442 4097 3355 34442 337 333333344	28 25 28 37 30 36 38 35 31 30 28 27 25 25	-45 -36 -29 -117 -267 -298 -319 -320 -320 -321 -322 -321 -322 -322 -323	

TEST No. 220

Time	P <sub>1</sub>	Tl	P <sub>a</sub>	Pb	Pc	Pd	P2	Т2	
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	_
0.7	44	-51	41	45	43	40	29	-11	
1	46	-78	41	41	3 7	3 5	24	-94	
1.6	44	-170	38	39	3 5	3 3	22	-74	
2	47	-232	42	46	46	45	3 7	-127	
3	48	-285	43	4.2	42	40	32	-294	
3.7	45	-299	40	40	59	34	23	-322	
4	46	-305	41	44	43	42	36	-288	
5	46	-309	42	4 3	42	40	34	-318	
6	45	-314	40	41	40	39	3 2	-319	
7	44	-316	39	40	40	3 7	31	-326	
8	43	-316	38	39	37	36	30	-320	
9	42	-316	3 7	38	3 7	3.5	28	-320	
10	41	-316	3 7	37	36	34	28	-321	
11	41	-317	36	3 7	35	3 3	27	-321	
12	40	-317	35	36	35	3 2	26	-321	
13	40	-317	35	35	34	3 2	2.5	-322	
14	39	-318	35	3.5	34	3 2	23	-322	
15	39	-318	35	35	33	31	23	-322	

TEST No. 221

_Time	Pl	Tl	Pa	РЪ	Pc	Pd	P <sub>2</sub>	T <sub>2</sub>	
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	-
0.5	33	+12	35	46	4 7	lı C	11.7	170	
1	42	-38	37	34		46	41	+18	
					32	30	22	-57	
1.6	39	-120	3 3	36	34	30	21	-37	
2	42	-185	3 8	40	40	3 8	34	-131	
2.5	41	-219	37	3 8	37	35	27	-212	
3	40	-256	35	39	38	36	26	-167	
3.5	39	-281	33	3 7	35	3 3	22	-158	
4	40	-295	36	39	38	36	30	-264	
5	39	-307	36	36	3.5	3 3	27	-307	
6	39	-315	35	3.7	3 7	35	30	-319	
7	39	-317	34	35	34	3 3	28	-321	
8	3.8	-317	34	34	34	3 2	27	-321	
9	3 7	-318	33	34	33	31			
10	36						26	-321	
		-320	3 2	3 3	3 2	30	24	-323	
11	36	-320	32	3 2	31	29	24	-322	
12	35	-320	31	3 2	30	2 8	2 3	-323	
13	34	-320	30	31	30	27	22	-324	
14	34	-320	29	30	29	26	21	-324	

TEST No. 222

Time	Pı	Tı	Pa	Pb	D	Da	, D	m	
(sec)	(psig)	(°F)	(psig)	(psig)	P <sub>C</sub> (psig)	Pd (psig)	(psig)	(°F)	
			.18,	(10-8)	(bore)	(borg)	(barg)	( 1)	
1	95	-101	88	88	8.8	83	63	- 99	
2	105	-258	97	9 7	95	93	73	-291	
3	105	-287	96	93	91	88	73	-303	
4	103	-293	94	91	89	8.5	71	-303	
5	100	-294	92	89	87	83	70	-303	
6	98	-294	90	87	85	81	68	-303	
7	96	-295	89	86	84	79	6.5	-304	
8	95	-295	88	83	80	75	62	-305	
9	93	-295	86	81	77	72	58	-306	
10	91	-295	86	80	76	72	5.5	-306	
11	90	-295	84	79	75	71	53	-307	
12	88	-296	83	78	74	69	52	-307	
13	87	-296	82	77	73	68	50	-308	
14	86	-296	81	75	72	68	50	-307	

Time P <sub>1</sub> T <sub>1</sub> P <sub>a</sub> P <sub>b</sub> P <sub>c</sub> P <sub>d</sub> P <sub>1</sub> T <sub>2</sub> (sec) (psig) (°F) (psig) (psig) (psig) (psig) (psig) (°F)  1 105 -146 98 96 95 90 70 -92 2 105 -270 98 97 97 94 76 -286 3 103 -293 95 92 90 86 70 -306 4 100 -297 93 89 88 83 69 -305 5 97 -298 89 85 84 80 67 -306 6 93 -299 86 83 82 76 64 -306 7 90 -300 83 80 78 73 60 -307 8 87 -300 81 76 73 68 55 -308 9 84 -300 79 73 70 65 51 -308 10 81 -300 77 71 68 64 48 -311 11 79 -302 75 70 67 62 46 -311 12 77 -303 72 68 64 61 44 -312 13 75 -303 71 65 63 59 42 -312 14 72 -304 68 64 61 57 41 -313										
(sec) (psig) (°F) (psig) (psig) (psig) (psig) (psig) (°F)  1					Pb	Pc	Pd	Ρħ	$T_2$	
2 105 -270 98 97 97 94 76 -286 3 103 -293 95 92 90 86 70 -306 4 100 -297 93 89 88 83 69 -305 5 97 -298 89 85 84 80 67 -306 6 93 -299 86 83 82 76 64 -306 7 90 -300 83 80 78 73 60 -307 8 87 -300 81 76 73 68 55 -308 9 84 -300 79 73 70 65 51 -308 10 81 -300 77 71 68 64 48 -311 11 79 -302 75 70 67 62 46 -311 12 77 -303 72 68 64 61 44 -312 13 75 -303 71 65 63 59 42 -312	(sec)	(psig)	(°F)	(psig)	(psig)	(psig)		(psig)	(°F)	_
7 90 -300 83 80 78 73 60 -307 8 87 -300 81 76 73 68 55 -308 9 84 -300 79 73 70 65 51 -308 10 81 -300 77 71 68 64 48 -311 11 79 -302 75 70 67 62 46 -311 12 77 -303 72 68 64 61 44 -312 13 75 -303 71 65 63 59 42 -312	2 3 4 5	105 103 100 97	-270 -293 -297 -298	9 8 9 5 9 3 8 9	97 92 89 85	97 90 88 84	94 86 83 80	70 76 70 69 67	-286 -306 -305 -306	
9 84 -300 79 73 70 65 51 -308 10 81 -300 77 71 68 64 48 -311 11 79 -302 75 70 67 62 46 -311 12 77 -303 72 68 64 61 44 -312 13 75 -303 71 65 63 59 42 -312	7	90	-300	83	80					
10 81 -300 77 71 68 64 48 -311 11 79 -302 75 70 67 62 46 -311 12 77 -303 72 68 64 61 44 -312 13 75 -303 71 65 63 59 42 -312										
12 77 -303 72 68 64 61 44 -312 13 75 -303 71 65 63 59 42 -312				77	71	6 8	64	48	-311	
14 72 -304 68 64 61 57 41 -313	13	75	-303	72	68	64	61	44	-312	
	14	72	-304	68	64	61	5 7	41	-313	

TEST No. 224

Time	Pl	Tl	Pa	Pb	Pc	$P_d$	Pl	$T_2$
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
0.6 1.4 2 3 4 5	64 75 74 81 80 78 77	-47 -133 -206 -262 -290 -295 -297	62 69 65 74 73 72 70 69	68 73 65 77 74 72 70 68	65 72 63 75 72 70 69 67	61 69 60 73 69 67 65	43 50 42 59 54 55 54 53	-50 -58 -70 -263 -306 -306 -306
8 9 10 11 12 13 14	75 73 71 71 70 69 69	-298 -298 -300 -300 -300 -300 -300	6 6 5 5 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	67 66 64 63 62 62 61	6543668888555555	61 60 58 56 54 54 54 54	51 50 48 45 43 42 40 39	-306 -306 -307 -308 -309 -309 -309 -310

Time	Pl	Tl	Pa	Рь	Pc	Pd	Pl	$T_2$	
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	-
0.7 1.6 2 3 4 5 6 7 8 9 10 11 12 13	3746542197533109	-80 -138 -231 -267 -292 -300 -305 -305 -305 -305 -305 -307	9 5 6 6 6 6 9 7 5 5 3 2 0 9 9 8 7 5 5 5 5 5 5 5 5 5 5 5 5 5	7646655555544444 4444	66466555554444444444444444444444444444	58400742187643097 555544444333	3 4 2 8 3 0 5 4 4 2 1 9 8 8 6 4 2 0 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	-87 -94 -97 -213 -307 -310 -310 -310 -310 -311 -311 -311 -311	

TEST No. 226

Time	Pl	Tl	Pa	Pb	Pc	Pd	Pl	T <sub>2</sub>	_
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	
1	NR	-57	3 3	30	27	24	14	-70	
1.5	<b>A</b>	-166	3 2	3 3	30	28	17	-58	
2		-234	36	3 7	3 7	35	3.0	-179	
3		-257	36	37	36	35	24	-194	
4		- 300	35	38	36	3.5	28	-311	
5		-305	35	36	3 5	32	24	-316	
6		-309	34	36	3 5	3 3	26	-316	
7		-310	3 3	34	3 3	3 2	26	-317	
8		-312	3 2	3 3	3 2	30	24	-317	
9		-314	31	32	31	29	23	-317	
10		-314	31	3 2	31	29	23	-317	
11		-314	31	3 2	31	28	24	-317	
12		-314	31	31	30	28	22	-317	
13	Y	-314	30	31	30	27	22	-319	
14	NR	-314	30	31	30	27	22	-319	

	Time	Pı	$T_1$	Pa	Pb	Pc	Pd	Pl	T <sub>2</sub>	
•	(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	
	, ,	1 0		1 0,	.16,		, 1, -0,	(P0)	` - /	
	1	44	-67	41	36	33	31	20	-53	
	1.6	42	-185	38	32	31	28	18	-46	
	2	45	-238	42	44	42	40	29	-118	
	3	44	-281	41	41	39	37	26	-271	
	4	43	-300	41	41	39	3 7	28	-303	
	5	42	-307	40	38	3 7	35	27	-317	
	6	41	-309	39	38	36	35	27	-317	
	7	40	-313	39	37	35	34	27	-317	
	8	39	-313	3 8	36	34	32	26	-317	
	9	39	-312	3 7	35	34	31	26	-317	
	10	38	-314	36	34	33	30	25	-318	
	11	37	-315	36	3 3	32	28	24	-318	
	12	3.7	-315	3.5	33	32	28	23	-318	
	13	36	-315	35	32	31	2.8	23	-318	
	14	35	-315	34	31	30	28	21	-318	

TEȘT No. 228

Time	Pl	$T_1$	Pa	Рь	Pc	Pd	Pı	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1 1.5 2.6 3.6 4.7 5 6 7 8 9 10	95 99 104 103 104 102 104 103 103 102 100 98 97 96 95	-168 -285 -298 -303 -304 -305 -305 -305 -306 -306 -306 -306	77 82 89 87 91 88 93 92 93 92 86 82 80 78	77 77 92 79 92 80 94 86 90 87 78 74 72	70 70 84 74 85 76 91 87 84 75 69 67	64 63 77 67 79 70 85 75 83 78 68 62 60 59	14 14 18 15 19 17 22 20 23 24 24 24 24	 +30 +16 +6 -36 -57 -192 -320 -322 -322 -323
12	95	-306	78	71 71	6 6 6 6	5 9 5 8	19 18	-323 -323
13 14	95	-306 -306	7 8 7 8	7 2 7 0	6 7 6 6	59 59	19	-323
15	95	-306	77	70	6 5	5 8	19	- 323 - 323
	1. 5 (2.60 A.E.) 1							

_Time	Pl	Tl	Pa	$P_{b}$	Pc	$P_d$	Р	$T_2$
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1 2 3 4 5 6 7 8	92 95 97 98 97 95 95	-114 -285 -293 -292 -294 -294 -294 -294	74 80 84 89 87 83 83 78	(psig) 73 77 80 88 91 88 74 71	(psig) 67 71 74 84 90 87 70 65	(psig) 61 65 68 78 83 82 59 56	(psig)  13 15 16 20 22 23 16 18	(°F) 10 -24 -103 -277 -312
9 10 11 12 13 14	93 92 92 91 90	-295 -295 -295 -296 -296 -296	76 76 76 75 73	69 69 68 66 67	64 64 64 62 61	56 56 57 57 54	20 19 20 19 17 16	-312 -313 -312 -312 -312 -312

TEST No. 230

1       108       -161       93       89       83       78       19          2       109       -285       93       89       83       77       19          3       108       -291       93       93       87       81       20       25         4       108       -292       96       92       88       82       21       7         5       108       -292       94       85       80       75       20       -87         6       106       -293       94       90       87       78       26       -217         7       105       -293       90       83       77       65       21       -311         8       103       -293       85       79       73       65       26       -308         9       102       -293       83       75       70       62       22       -310         10       102       -293       83       76       71       63       21       -310         12       100       -293       82       75       69       61       20       -311	Time (sec)	P <sub>1</sub> (psig)	T <u>1</u>	Pa (psig)	Pb (psig)	P <sub>C</sub> (psig)	P <sub>d</sub> (psig)	P <sub>l</sub> (psig)	T <sub>2</sub>
	2 3 4 5 6 7 8 9 10 11 12 13	108 109 108 108 108 106 105 103 102 102 101 100	-285 -291 -292 -293 -293 -293 -293 -293 -293 -293	93 93 93 96 94 90 85 83 83 82 81	89 89 93 92 85 90 83 75 76 75 74	83 83 87 88 80 87 77 73 70 71 70 69	78 77 81 82 75 78 65 62 63 62 61 61	19 19 20 21 20 26 21 26 22 21 20 20	25 7 -87 -217 -311 -308 -310 -310 -311 -311

-	Time	Pl	T	P <sub>3</sub> (psig)	РЪ	Pc	Pd	Pl	T <sub>2</sub>	
	(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	
	1	42	-51	30	30	27	24	0		
	2	45	-257	37	42	37	34	3		
	3	45	-295	3.5	33	29	26	1		
	4	48	-299	43	45	43	41	6		
	5	47	-303	37	34	32	28	2	24	
	6	42	-305	34	33	29	26	1	16	
	7	47	-305	41	41	39	3.5	4	-16	
	8	41	-307	3 2	39	. 37	33	2	-30	
	9	47	-306	42	42	39	36	5	-142	
	10	42	-307	34	29	26	23	1	-137	
	11	46	-307	41	42	40	38	8	-309	
	12	41	-307	31	27	23	21	1	-255	
	13	45	-307	39	39	37	33	7	-319	
	14	41	-307	31	29	27	23	1	-291	

TEST No. 232

Time	Pl	Tl	Pa	Pb	PC	$P_d$	Pl	T <sub>2</sub>
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)
1	71	-124	56	54	49	44	7	
2	73	-287	59	55	50	45	8	
3	75	-294	6.5	69	65	60	13	
4	77	-295	69	62	58	5 2	11	21
5	71	-297	60	62	57	52	10	3
6	76	-297	70	69	66	62	15	-69
7	7.2	-297	63	78	54	51	11	-106
8	72	-297	6.5	64	63	59	16	-293
9	70	-297	5 9	57	5 2	45	10	-262
10	70	-298	59	54	49	40	9	-316
11	69	-299	57	50	45	39	10	-317
12	69	-298	5 5	50	45	39	10	-316
13	69	-299	5.5	50	45	39	11	-317
14	69	-298	56	50	46	40	10	-316

TEST No. 233

Time	Pl	Tl	Pa	Pb	Pc	Pd	Pl	T <sub>2</sub>	
(sec)	(psig)	(°F)	(psig)	(psig)	(psig)	(psig)	(psig)	(°F)	-
7	70	0.6	57	F.C.	<b>.</b>	11.0	7		
+		-86		56	50	46	1		
2	71	-290	59	57	5 2	48	8		
3	71	-298	60	5 7	52	48	8		
4	73	-300	64	63	58	54	10		
5	72	-300	63	60	58	53	10	23	
6	73	-302	66	66	64	60	13	-27	
7	71	-303	6 2	64	60	56	12	-29	
8	70	-303	62	67	66	63	19	-135	
9	70	-303	60	51	48	44	10	-291	
10	69	-303	58	5 2	49	44	12	-316	
11	69	-303	56	50	46	41	11	-317	
12	6 8	-303	54	49	44	36	10	-317	
13	68	-303	54	50	46	41	12	-317	
14	67	-305	54	49	45	39	10	-319	

TEST No.	P <sub>l</sub> (psig)	Pn (psig)	T <sub>1</sub> (°F)	T <sub>2</sub>
123456789011234567890A 1234567890A 122223456789033233456789011234545	87 82 99 99 90 91 91 91 91 91 91 91 91 91 91 91 91 91	35 66 87 91 92 166 93 25 75 77 68 73 68 74 75 76 82 16 72 82 53 16 72 82 53 16 72 82 53 16 84 54 74 85 86 86 86 86 86 86 86 86 86 86 86 86 86	-278 NR* -175 -180 -180 -172 -180 -177 -80 -187 -187 -187 -187 -187 -133 -125 -61 -66 -47 -48 -48 -48 -48 -48 -48 -49 -130 -130 -130 -130 -130 -130 -130 -130	-290 NR -283 -274 -300 -286 -275 -290 -287 -290 -288 -289 -288 -289 -288 -288 -256 +1 +2 +88 -257 -288 -288 -288 -288 -288 -288 -288 -28

\*NR = Not Recorded

TEST No.	P <sub>l</sub> (psig)	Pn (psig)	T <sub>1</sub>	T2 (°F)
4478901234567890123456789	91 87 90 95 95 99 97 98 98 98  98 100 101 100 NA** NA NA NA NA NA NA NA	85 81 82 93 91 95 97 96 95  96 98 100 99 NA NA NA NA NA NA NA NA	-155 -148 -162 -140 -124 -40 -103 -138 -144121 -46 -66 -147 NA	-263 -256 -256 -223 -150 -162 +4 -42 -81 -7819 +21 -63 -38 NA
70 71 <b>7</b> 2 73 74 75 76	107 85 26 62 93 70	102 54 30 28 64 45 N2 Discharge	-193 -279 -144 -228 -239 -295	   -239 -247
77 78 79 80 81 82 83 84 85 86 87 88 89	89 84 89 89 86 56 90 90 64 92 88 81 93 88	82 Discharge 60 55 62 87 86 61 90 93 59 92 89 81 93 85	-281 -285 -276 -181 -158 -90 -115 -96 -126 -130 -145 -141 -170 -195	-285 -286 -285 -288 -311 -283 -279 -259 -284 -277 -283 -283 -314 -312

\*\*NA = Not Applicable

TEST	P <sub>l</sub>	P <sub>n</sub>	T <sub>1</sub>	T <sub>2</sub>
No.	(psig)	(psig)	(°F)	
91 92 93 94 95 96 97 99 100 102 103 104 105 107 108 109 111 112 113 114 115 116 117 118 119 121 123 124 125 127 129 131 133 134 135 136 137 137 137 137 137 137 137 137 137 137	53 53 77 59 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 10 96 96 96 96 96 96 96 96 96 96 96 96 96	46539291982541114672344444004267988767767855863019858635555555555555555555555555555555555	-145 -1833370981453752494942644946201885345462029756 -124052494946018839586299756 -124052494946018839586299756 -124052494946018839586299756	-301 -299 -285 -299 -293 -288 -296 

TEST No.	(psig)	(psig)	T <sub>1</sub>	(°F)
NO.  226 227 228 229 230 231 233 234 235 238 239 241 244 244 244 244 244 244 244 244 244	Psig) 44 104 97 108 45 75 71 66 94 109 104 99 87 90 84 75 99 88 106 109 103 107 109 108 115 113 85 108	24 26 19 16 20 21 8 8 63 99 86 75 77 65 20 22 37 38 877 39 54 95	-257 -281 -304 -293 -291 -295 -294 -298 -303 -306 -306 -311 -321 -321 -309 -314 -312 -313 -301 -302 -307 -303 -309 -309 -309 -309 -309 -309 -309	-194 -271 +30  +25  -309 -303 -305 -312 -318 -320 -306 -315 -312 -313 +78 +61 +61 +73 +72 +59 +85 +70 -307
255	100	87	-330	-313

#### APPENDIX I

#### REFERENCES

- 1. Klueg, E. P., "Liquid Nitrogen as a Powerplant Fire Extinuishant," Fire Technology, National Fire Protection Association, August 1969.
- 2. Klueg, E. P. and Demaree, James E., "An Investigation of In-Flight Fire Protection With a Turbofan Powerplant Installation," Federal Aviation Administration Final Report NA-69-26 (DS-68-26), April, 1969.