

Liquid Burner Development for Powerplant Fire Test

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Project Objective:

- Develop the operating settings for NexGen burner for powerplant fire tests
 - NexGen burner should simulate previously FAA approved liquid burners
 - NexGen burner should be robust and repeatable

Approach:

- Sensitivities of NexGen burner settings on burner temperature and heat flux → to understand burner behavior and to derive the “accuracy” or “tolerance” for burner settings
- Derive the NexGen burner settings
 - Comparison of fire test results from different burners (Park, NexGen and ISO) ...*future work*
 - Comparison of fire test results from NexGen burner with different settings ...*future work*

Standard Liquid Burners

from

Power Plant Engineering Report NO. 3 A

Standard Fire Test Apparatus and Procedure (for flexible hose assemblies)

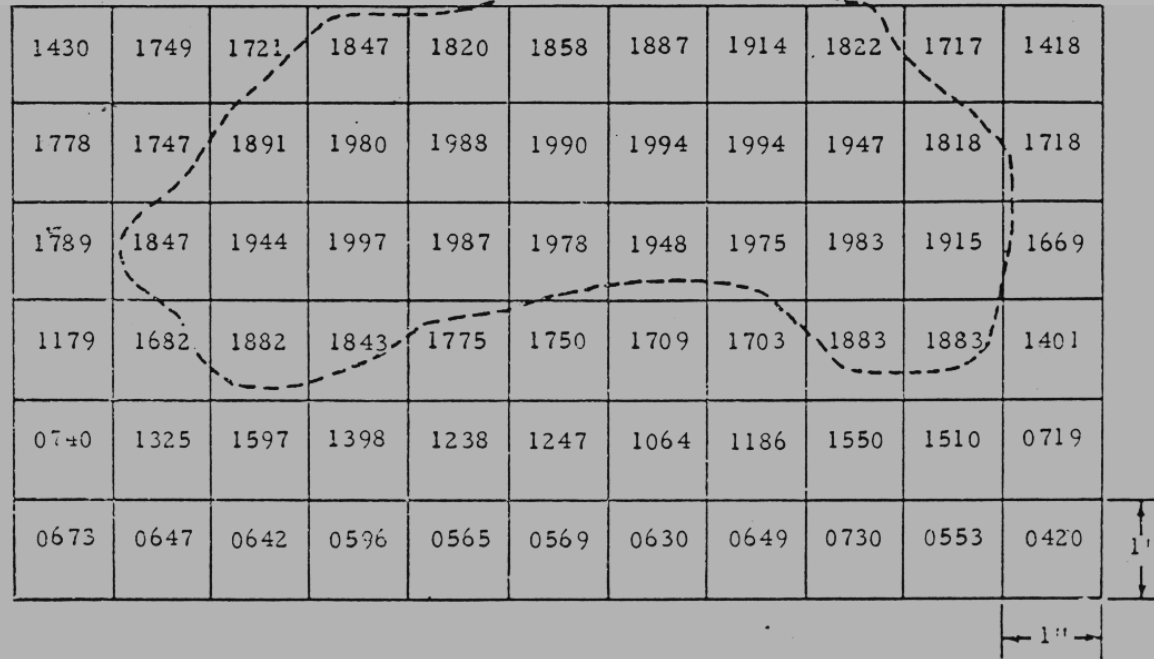
E. P. Burke and T. G. Horeff

March 1978

Individual TC Temperature Requirements: 1850~2150 F

<u>Burner</u>	<u>Rated Fuel Flow Capacity</u>	<u>Motor</u>	<u>Blower Wheel (inches)</u>	<u>Fuel Pump</u>	<u>Tube Extension (inches)</u>	<u>Heat Transfer to 1/2-in. Tube Btu/hr.</u>	<u>Total Thermal Energy (Btu/ft²/s)</u>	<u>Flame Oxygen Concentration (percent)</u>
Lennox OB-32 ¹	(Reference) only	1/4 hp 1,725 r/min.	3 5/8 x 6 1/8	Two Stage	4x12	4,574	9.8-10.8	5-11
Carlin 200 CRD	2.0-5.0 gal/hr.	1/4 hp 3,450 r/min.	3 1/2 x 5 1/4	Single Stage	4 1/8x11	4,545	9.3-11.2	6.5-8.5
Stewart Warner HPR-250	1.35-2.50 gal/hr.	1/7 hp 3,450 r/min.	3 9/16 x 5 3/8	Single Stage	4x13 5/32	4,646	9.3-10.1	9.2-9.5
Stewart Warner FR-600	2.0-6.0 gal/hr.	1/3 hp 3,450 r/min.	3 15/16 x 6 1/32	Two Stage	4x12 7/8	4,466	9.9-10.9	8.5-15.2

Typical Temperature Distribution



Liquid Burner Requirements

Park Burner (FAA Handbook & AC20-135)

	Fuel Rate (GPH)	Air Rate (ft ³ /min)	A/F ratio (equivalence ratio)	TC size	Temperature	Heat Flux (Btu/ft ² -s)
CH. 7 Seat Cushion	2	67	23.0 (0.67)	1/16" (0.010")	7 TC > 1750 F 5 TC > 1800 F AVE > 1800 F	10
CH. 11 & 12 Powerplant	?	?	?	1/16" (0.025")	AVE > 2000F	9.3
AC20-135 & AC33-17-1	?	?	?	1/16" to 1/8" (0.010" - 0.025")	1850 F - 2150 F TAVE > 2000 F	9.3

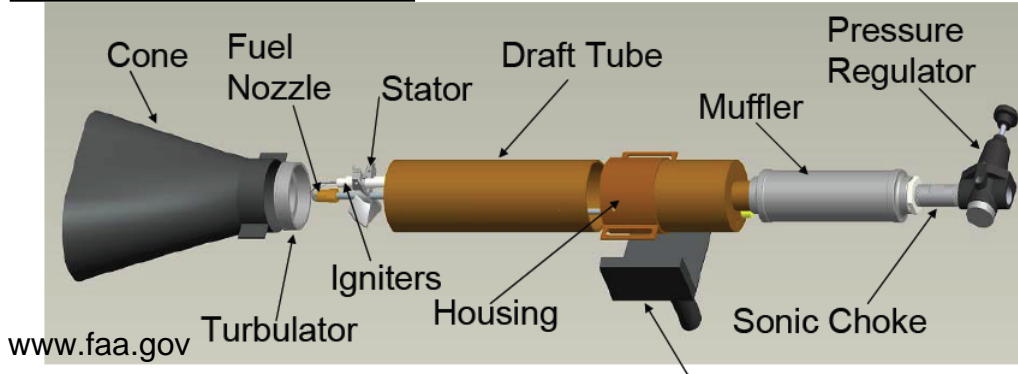
NexGen Burner

	Fuel Rate (GPH)	Air Rate (ft ³ /min)	A/F ratio (equivalence ratio)	TC size	Temperature	Heat Flux (Btu/ft ² -s)
Seat Cushion	2 .02	45-53 (35 to 45 psi)	14.4 – 17.3 (1.02 – 0.849)	1/8"	About 100 F less than Park Burner	10
Acoustic Insulation (AC-25.856-2A)	6	63	6.72 (2.16)	1/8"	1900 ±100°F	16.0 ±0.5
Powerplant	?	?	?	?	?	?

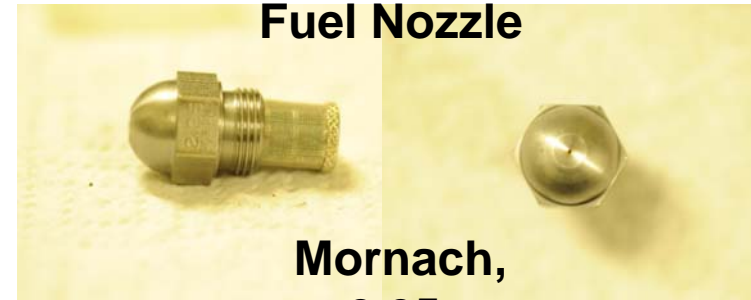
NexGen and Park Burner

NexGen Burner

Both fuel and air rate can be accurately metered and controlled



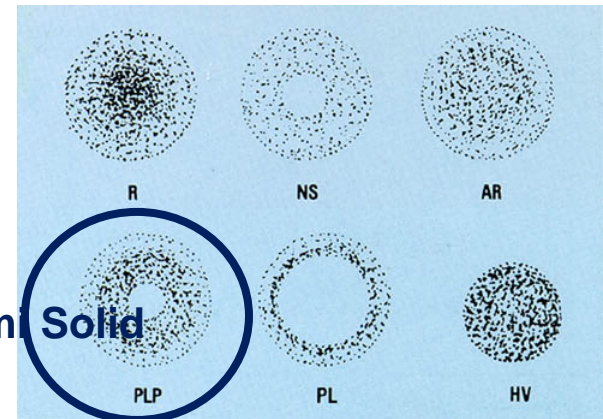
www.faa.gov



Fuel Nozzle

**Mornach,
2.25
80° PLP**

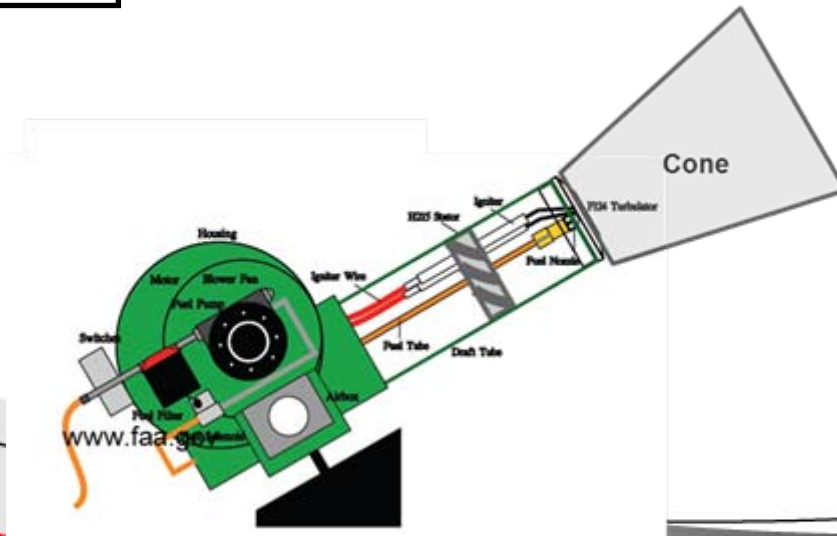
- ✓ 2.25: Max Flow Rate=2.25 gph
- ✓ 80°: Spray Angle=80°
- ✓ PLP: Spray Pattern



Semi Solid

Park Burner

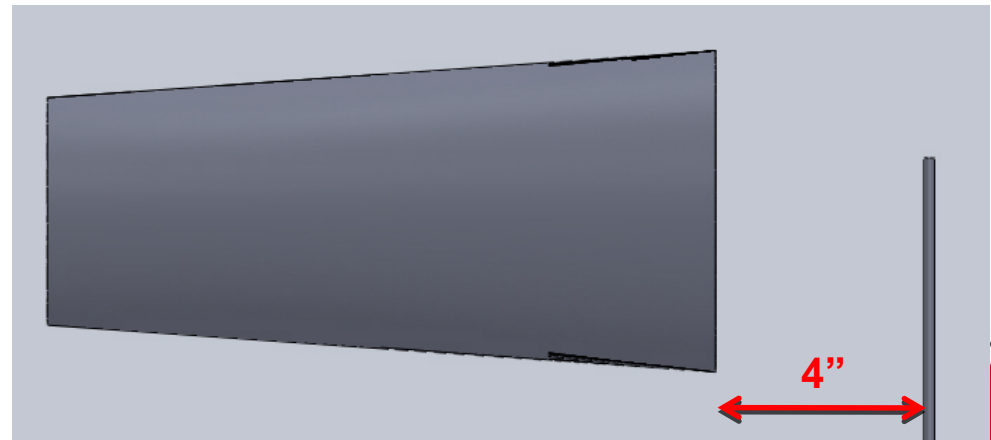
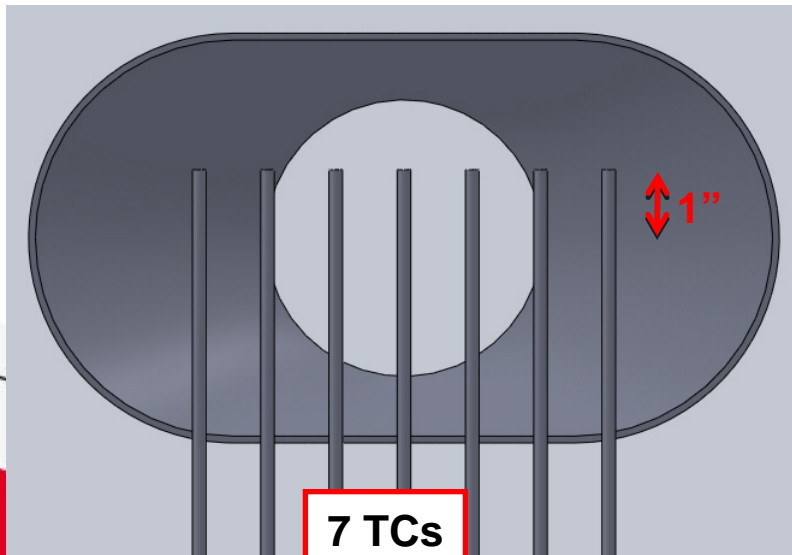
Only fuel rate can be accurately metered and controlled



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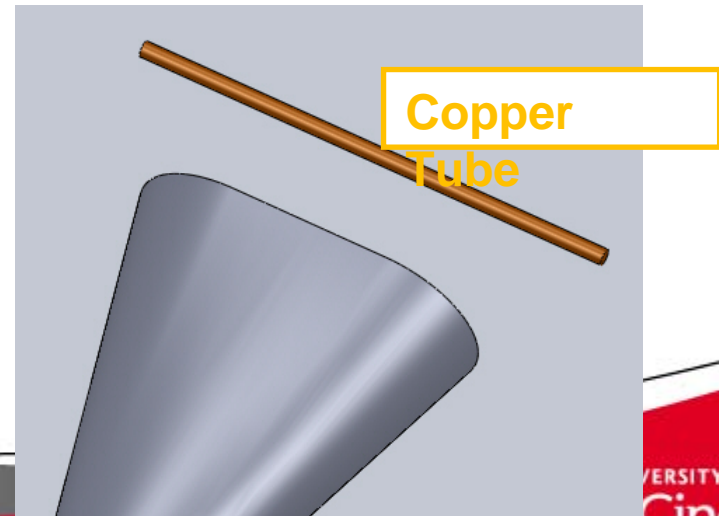
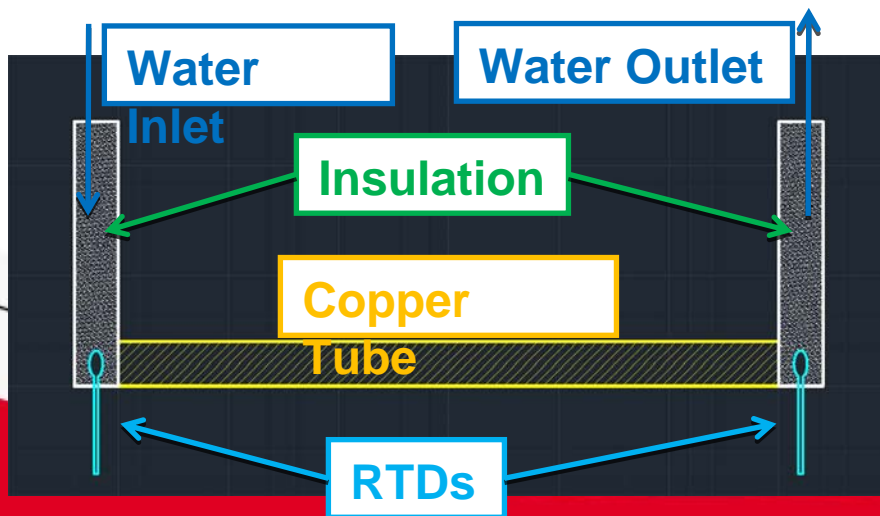
Temperature Calibration

- Thermocouple rack
 - 7 thermocouples
 - K-Type, 1/8 or 1/16 inch stainless steel sheath
 - Exposed bead, 1/4 inch exposed wire
 - Located at target plane (4 inch from burner exit) and 1 inch above burner horizontal center line
- Temp. Min. Avg. = **2000 F**, Individual = **2000±150 F**

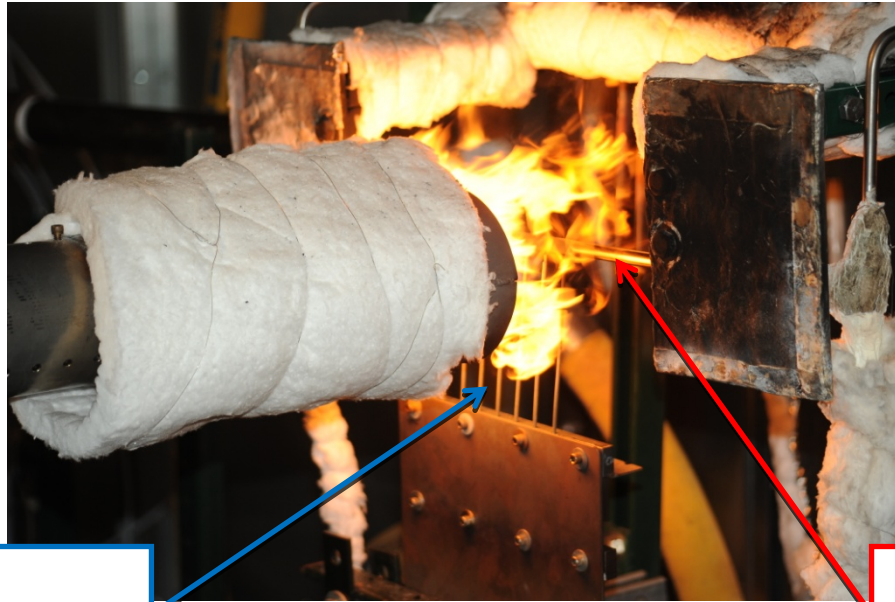


Heat Flux Calibration

- Copper tube used as heat transfer device
 - 1/2 inch outer diameter, 15 inch un-insulated length
 - Water flow rate maintained a 3.8 liter/ min
 - Center of copper tube located at target plane (4 inch from burner exit)
- Heat flux calculated from water flow rate and temperature rise across the tube
 - Exposed tube length for heat flux calculation is equal to burner horizontal size (11 inch)
 - Min. **9.3 Btu/ft²-s**

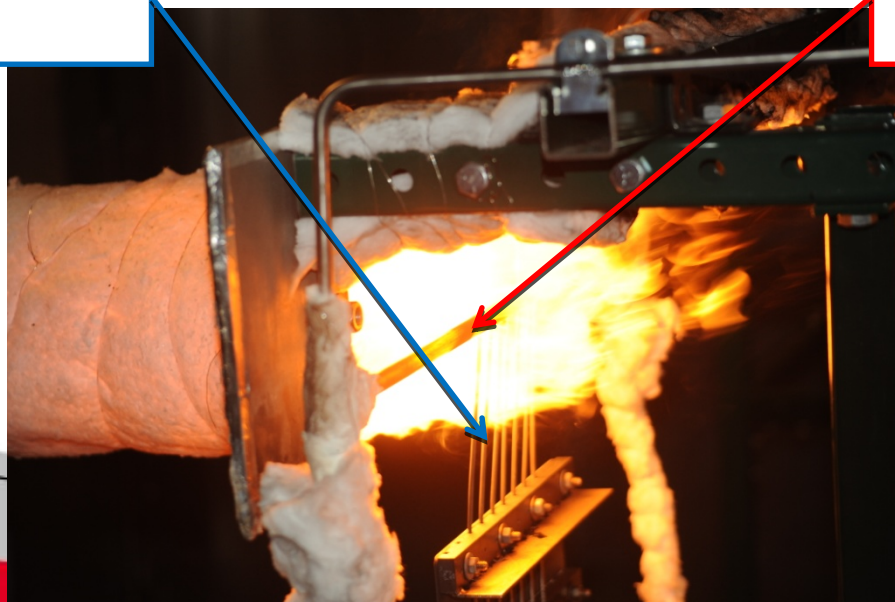


Temperature/ Heat Flux Calibration Rigs



**7 TCs Rack for
Temperature
Calibration**

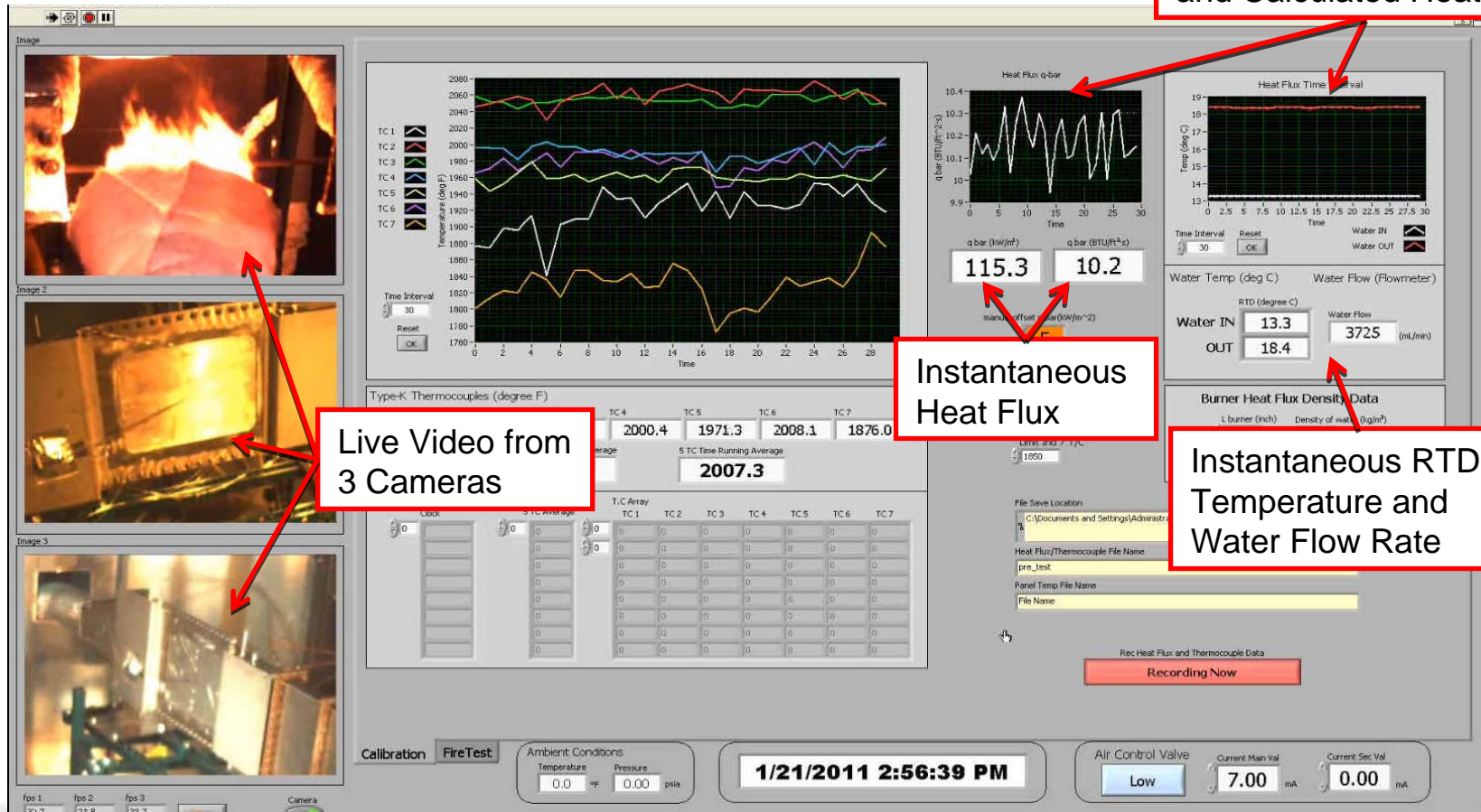
**Copper Tube for
Heat Flux Calibration**





Powerplant Calibration Requirements

Time Traces of RTDs and Calculated Heat Flux



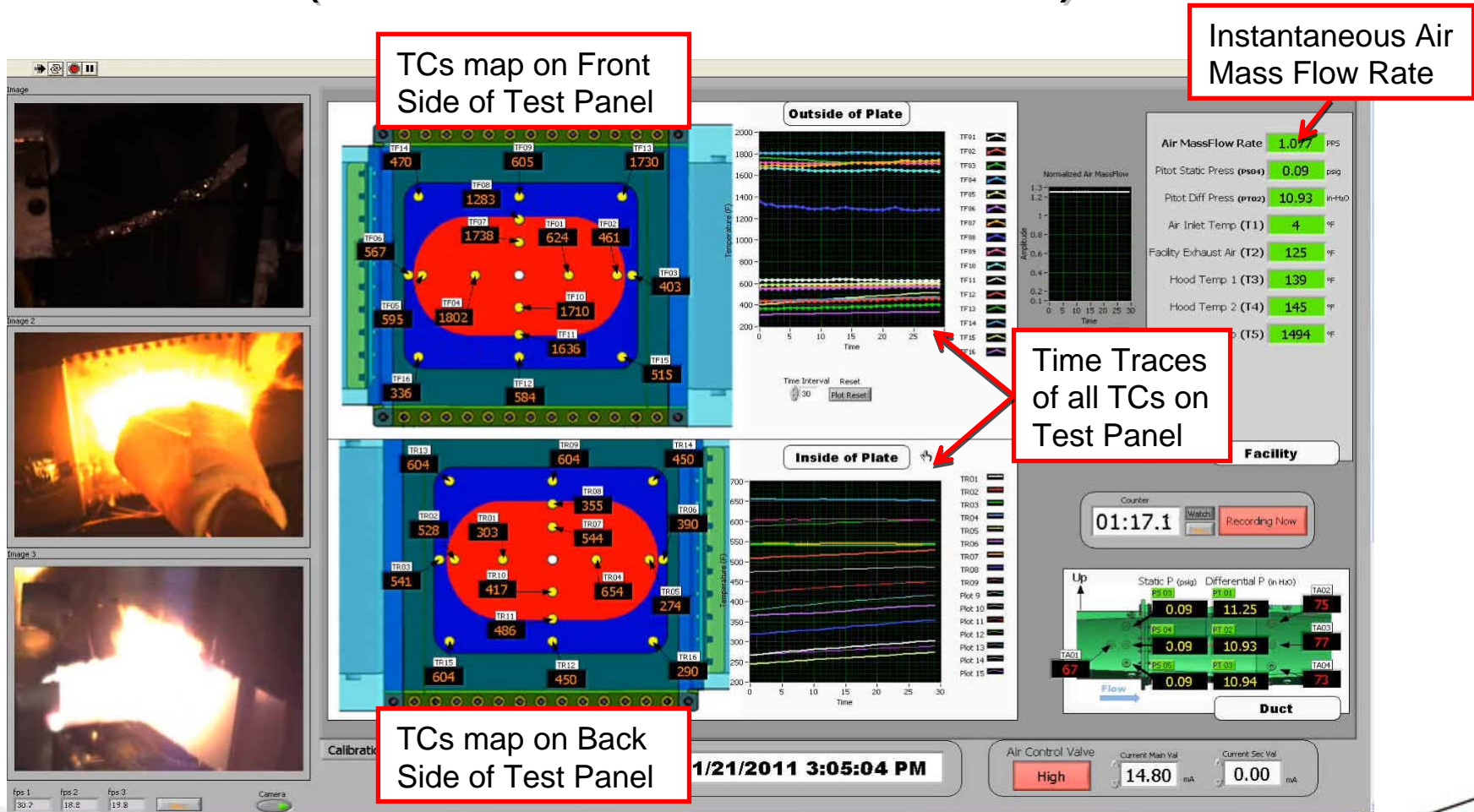
Live Video from 3 Cameras

Instantaneous Heat Flux

Instantaneous RTD Temperature and Water Flow Rate

Courtesy by HONDA R&D

Powerplant Fire Wall (engineering) Fire Test (with back side air flows)



Courtesy by HONDA R&D

Burner Configurations

Turbulator Modifications

TC Wire and Bead size

Back Side



Front Side



Original
w/o tab

Short Tabs (1"x3/4")



Long Tabs (1 1/8"x3/4")



K-type TC-Big

bead size 0.033 inch

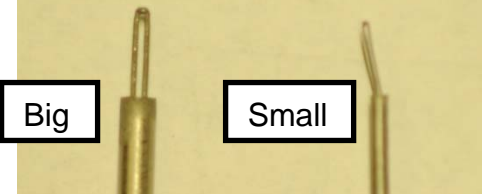
wire size 0.020 inch

K-type TC-Small

bead size 0.020 inch

wire size 0.012 inch

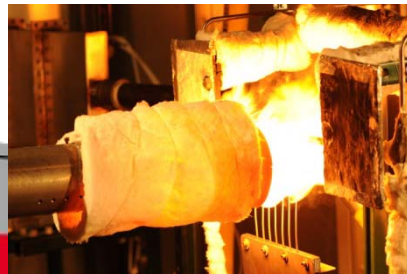
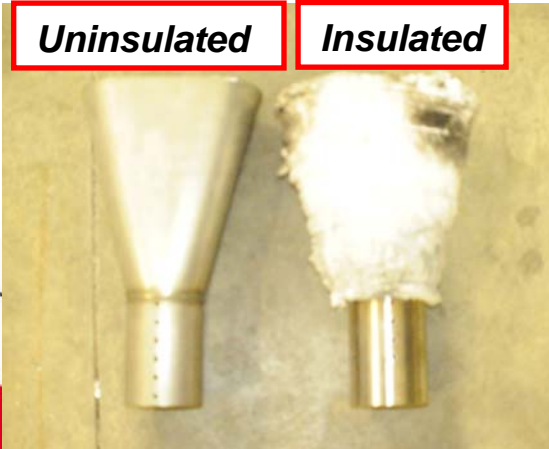
New



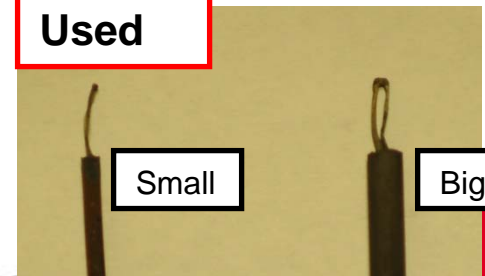
Insulation on Expansion Cone

Uninsulated

Insulated



Used



NexGen Burner w/o cone
Original v.s. Modified Turbulator

Record: 3000 fps/ Playback: 15 fps

Park Burner Performance

$$\theta = \frac{T_{max} - T_{min}}{T_{avg}}$$

	Insulation	TAB	TC	Fuel rate (GPH)	T_avg (F)	T_max (F)	Heat Flux (BTU/ft^2-s)	θ
						T_min (F)		
Park Burner	N	N	Big	3.70	1996	2157	12.95	17.23%
						1813		
	Y	N	Big	3.65	2025	2147	14.45	13.93%
						1865		
	N	Short	Big	3.22	2013	2095	11.10	11.30%
						1867		
	Y	Short	Big	2.86	2009	2098	12.07	10.59%
						1885		

- ✓ Original (unmodified):
 - ✓ Extremely hard to reach ave.2000 F temperature
 - ✓ Highly non-uniform temperature distribution
- ✓ With **Tabs** to improve aerodynamics and air/fuel mixing:
 - ✓ Reduce the required fuel rate to meet 2000 F Ave. temperature requirements
 - ✓ Reduce heat flux by about 20%
 - ✓ improve temperature distribution
- ✓ With **Thermal Insulation** to reduce heat loss from expansion cone:
 - ✓ Reduce the required fuel rate to meet 2000 F Ave. temperature requirements
 - ✓ Increase heat flux by about 12%
 - ✓ improve temperature distribution

NexGen Burner Performance

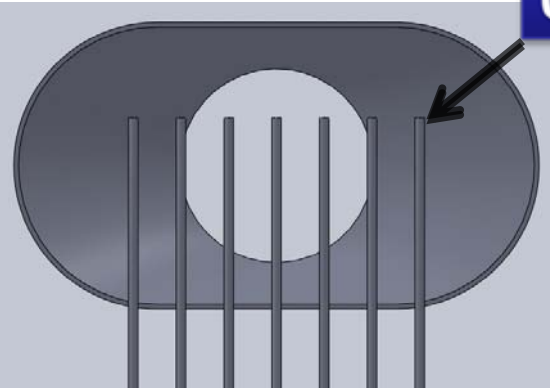
$$\theta = \frac{T_{max} - T_{min}}{T_{avg}}$$

	Insulation	TAB	TC	Jet-A (GPH)	Air (SCFM)	A/F	ϕ	T_adia(F)	T_avg (F)	T_max (F)	Heat Flux (BTU/ft^2-s)	θ
										T_min (F)		
NexGen Burner	N	Short	TC1	2.86	61.09	14.02	1.05	3647	2008	2095	10.51	10.53%
										1884		
	Y	Short	TC1	2.38	59.12	16.33	0.90	3469	2038	2092	11.40	5.11%
										1988		
	N	Long	TC1	2.44	55.18	14.86	0.99	3615	2007	2063	9.70	7.33%
										1916		
	Y	Long	TC1	2.26	55.18	16.05	0.91	3491	2002	2059	11.07	6.62%
										1927		

- ✓ Will use less fuel rate, compared to Park burner, to meet temperature requirements
- ✓ More uniform temperature distribution compared to Park Burner
- ✓ “Long Tabs” has lower heat flux and more uniform temperature
- ✓ Burner behavior is sensitive to details of TAB’s “geometry and configuration”
- ✓ Insulation on expansion cone reduces the fuel rate requirements and has more uniform temperature but increase the heat flux about 10%
- ✓ A/F ratio (equivalence ratio: 0.90 -1.05) is near stoichiometric and will generate the “highest” temperature fire (around 3500 F theoretical gas temperature)

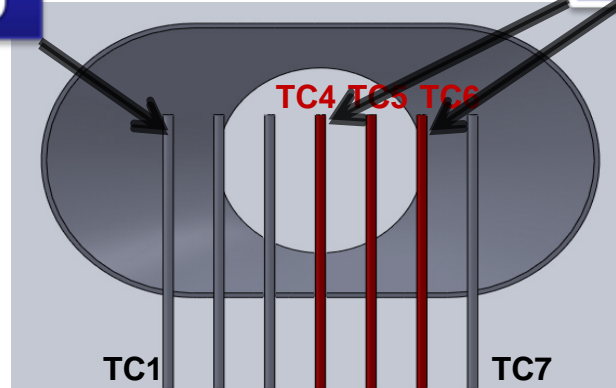
Different TC Configurations

Configuration 1

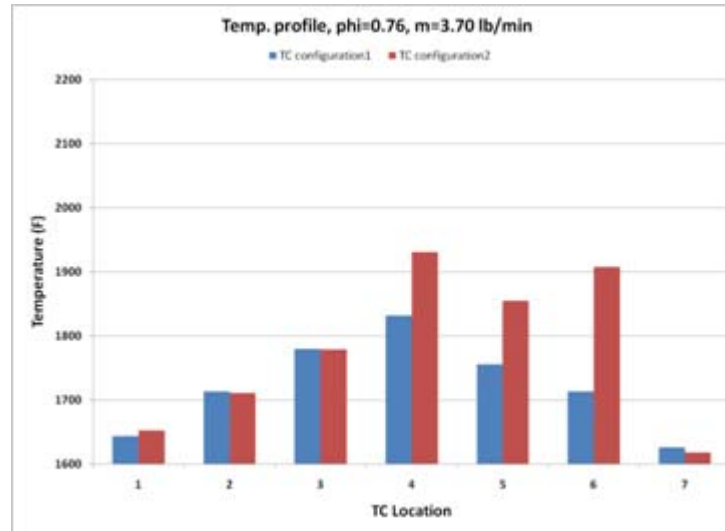
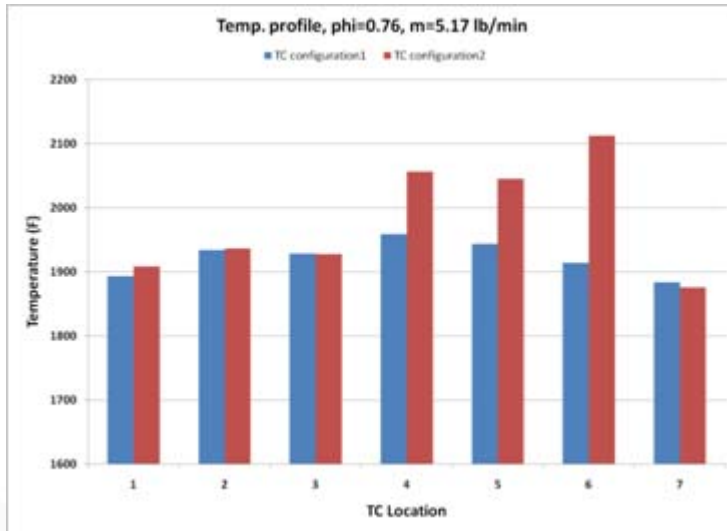
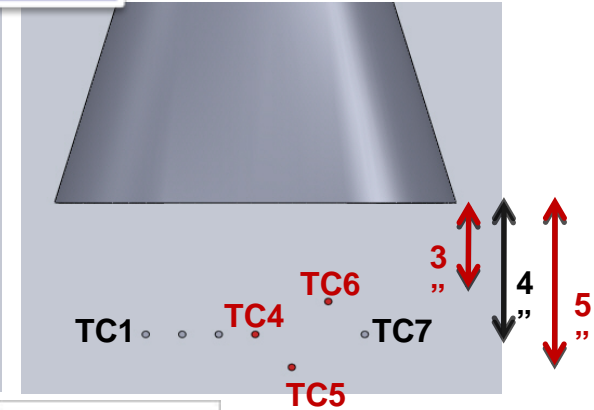


0.020"

Configuration 2:



0.012"



- ✓ Small TCs provide around 100 F higher temperature readings
- ✓ Distance from burner exit impacts on measured temperatures

Effects of TC Wire and Bead Size

	Insulation	TAB	TC	Jet-A (GPU)	Air (SCFM)	A/F	ϕ	T_adia(F)	T_avg (F)	T_max (F)	Heat Flux (BTU/ft ² -s)	θ
										T_min (F)		
NexGen Burner	N	Long	Big	2.44	55.18	14.86	0.99	3615	2007	2063	9.70	7.33%
										1916		
	N	Long	Small	2.35	55.18	15.43	0.95	3563	2021	2101	9.40	10.48%
										1889		
	Y	Long	Big	2.26	55.18	16.05	0.91	3491	2002	2059	11.07	6.62%
										1927		
	Y	Long	Small	2.20	55.18	16.50	0.89	3448	2008	2106	10.74	8.60%
										1933		

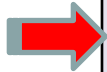
- ✓ Thermocouple wire size (bead size) can affect burner settings and heat flux
- ✓ With smaller thermocouple bead (wire size) the fuel rate can be reduced by about 3 % and heat flux is also reduce by about 3 %

Fuel Sensitivity, Air Sensitivity for NexGen Burner

Burner Arrangements: Inculated Cone, TAB- small, TC- small

Fuel Sensitivity, NexGen Burner

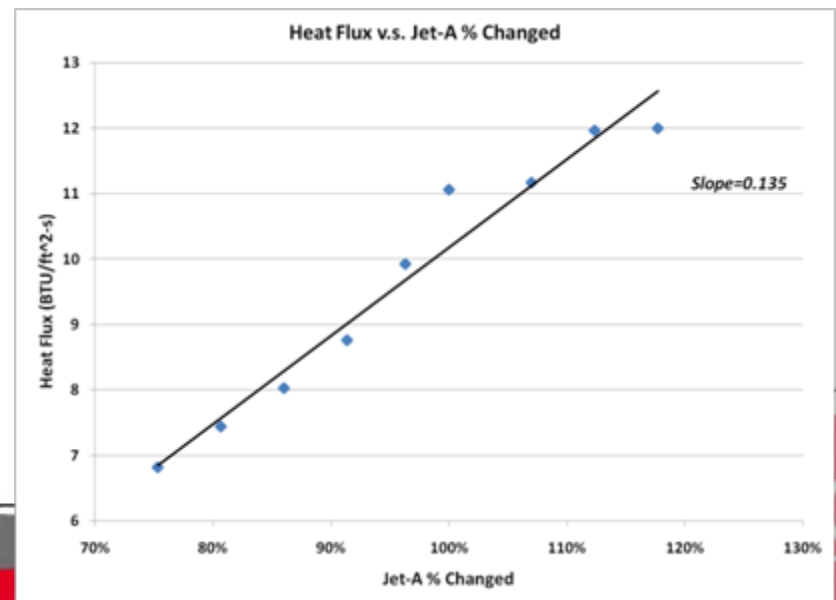
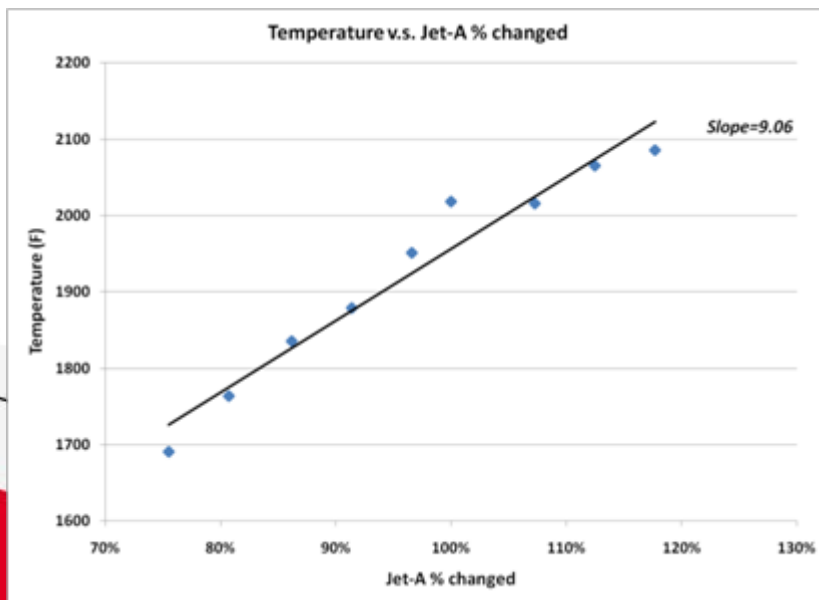
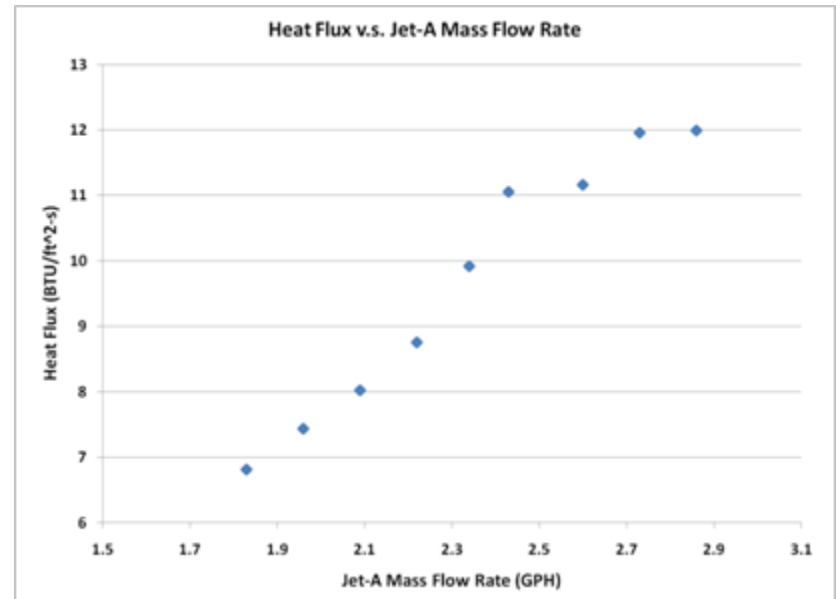
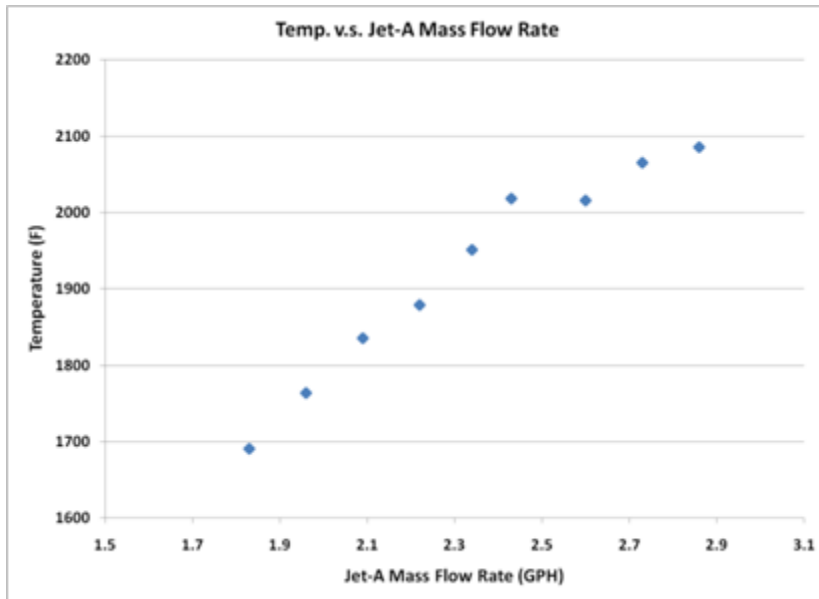
Jet-A (GPU)		Air (SCFM)	A/F	ϕ	T_adia(F)	T_avg (F)	T_max (F) T_min (F)	Heat Flux (BTU/ft ² -s)
1.83	75.5%	63.9	22.89	0.64	2735	1690	1809 1392	6.82
1.96	80.7%	63.9	21.41	0.69	2895	1763	1870 1525	7.44
2.09	86.2%	63.9	20.05	0.73	3019	1835	1934 1674	8.03
2.22	91.4%	63.9	18.91	0.78	3167	1878	1982 1700	8.76
2.34	96.6%	63.9	17.89	0.82	3277	1951	2049 1808	9.92
2.43	100.0%	63.9	17.28	0.85	3354	2018	2102 1868	11.06
2.6	107.3%	63.9	16.11	0.91	3491	2015	2135 1824	11.17
2.73	112.5%	63.9	15.36	0.96	3579	2065	2189 1864	11.96
2.86	117.7%	63.9	14.68	1.00	3626	2085	2215 1860	11.99



 **:baseline point**

- ✓ Air mass flow rate is fixed around 63.9 SCFM.
- ✓ Jet-A mass flow rate is changed from 1.83~2.86GPH, and ϕ is also changed from 0.64 to 1.

Fuel Sensitivity, NexGen Burner ...con't



Air Sensitivity, NexGen Burner

Jet-A (GPH)	Air (SCFM)		A/F	ϕ	T_adia(F)	T_avg (F)	Temp. Diff. %	T_max(F)	Heat Flux (BTU/ft ² -s)	Heat Flux Diff. %
								T_min(F)		
2.34	55.3	86.5%	15.48	0.95	3563	1892	-1.38%	2085	9.93	4.73%
								1463		
2.34	58	90.6%	16.20	0.91	3491	1938	1.00%	2096	9.29	-2.04%
								1605		
2.34	61	95.5%	17.09	0.86	3379	1948	1.55%	2071	9.08	-4.25%
								1704		
2.34	63.9	100.0%	17.89	0.82	3277	1943	1.29%	2055	9.28	-2.17%
								1719		
2.34	66.8	104.4%	18.68	0.79	3196	1909	-0.50%	2027	9.75	2.83%
								1659		
2.34	69.6	108.9%	19.48	0.75	3079	1881	-1.96%	2002	9.57	0.89%
								1629		

 :baseline point

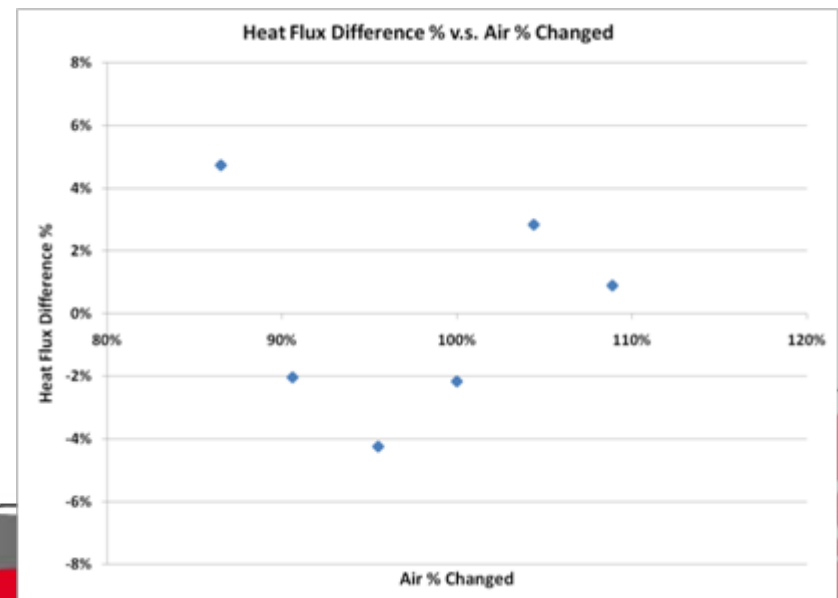
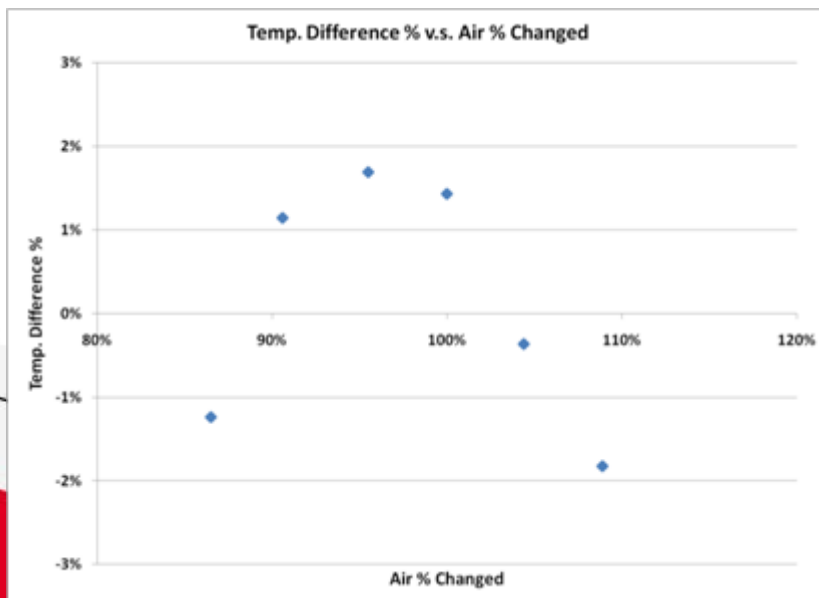
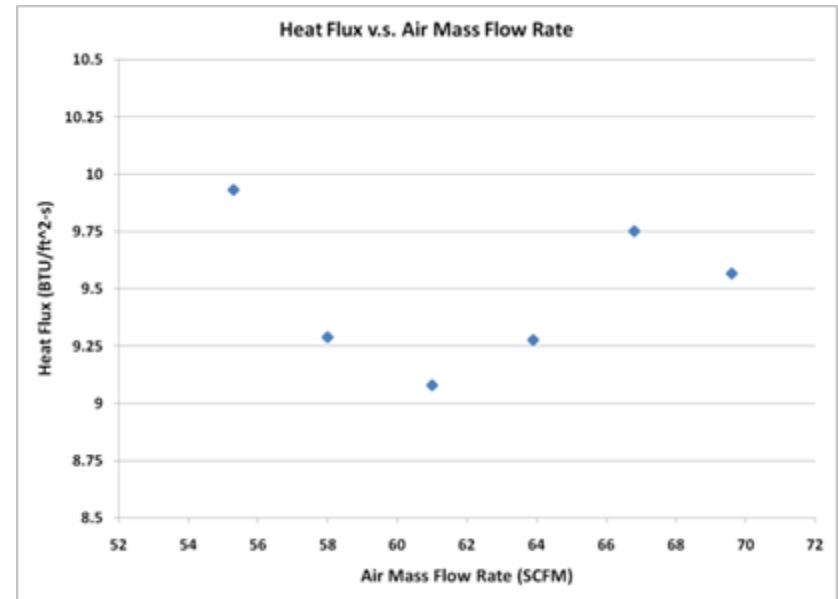
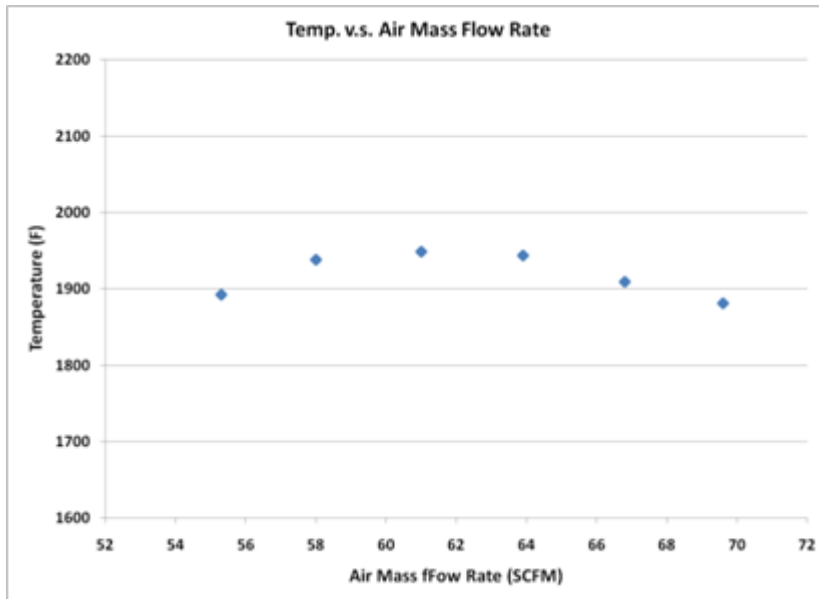
$$Temp. Diff. \% = \frac{T_{indi\ case} - T_{mean,6\ cases}}{T_{mean,6\ cases}}$$

$$Heat\ Flux\ Diff. \% = \frac{\dot{Q}_{avg,indi} - \dot{Q}_{avg,6\ cases}}{\dot{Q}_{avg,6\ cases}}$$

✓ Jet-A mass flow rate is fixed around 2.34 GPH

✓ Air mass flow rate is changed from 55.3~69.6 SCFM, and ϕ is changed from 1 to 0.75 as well.

Air Sensitivity, NexGen Burner ...con't

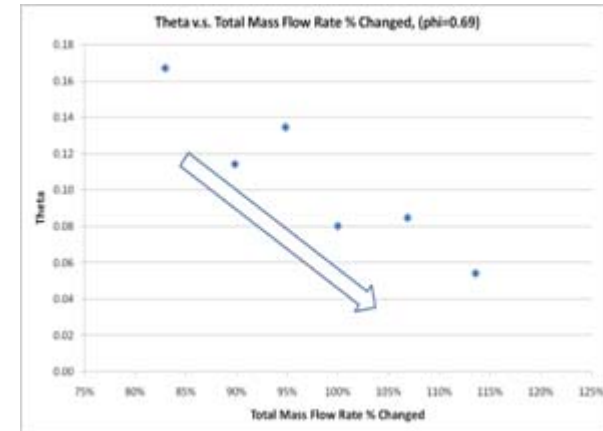
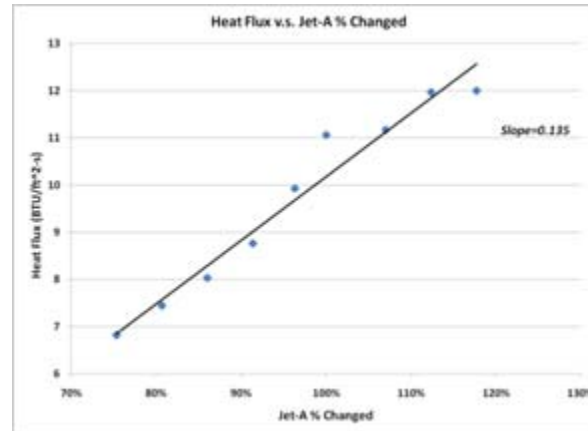
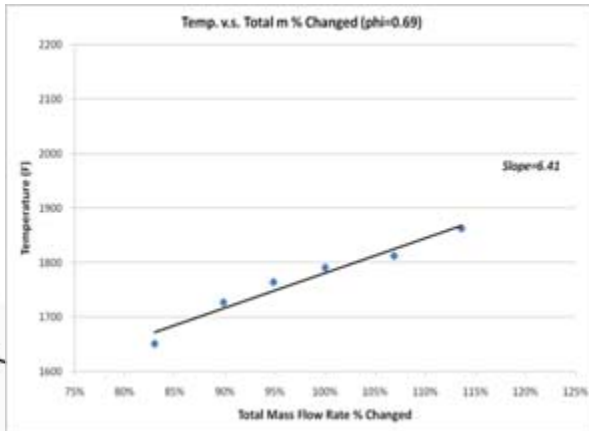


Mass Flow Rate Effect, NexGen Burner

Jet-A (GPH)	Air (SCFM)	Total Mass (lb/min)		A/F	ϕ	T_adia(F)	T_avg (F)	T_max(F)	Heat Flux (BTU/ft ² -s)	θ
								T_min(F)		
1.47	47.6	3.68	82.99%	21.23	0.69	2895	1651	1744	6.74	0.17
								1468		
1.6	51.6	3.99	89.86%	21.24	0.69	2895	1727	1806	7.53	0.11
								1609		
1.68	54.4	4.21	94.85%	21.22	0.69	2895	1764	1851	7.91	0.13
								1613		
1.77	57.3	4.44	100.00%	21.23	0.69	2895	1791	1857	8.39	0.08
								1714		
1.89	61.2	4.74	106.87%	21.24	0.69	2895	1812	1876	8.74	0.08
								1722		
2.00	65.2	5.04	113.58%	21.22	0.69	2895	1863	1914	9.48	0.05
								1813		

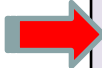


:baseline point

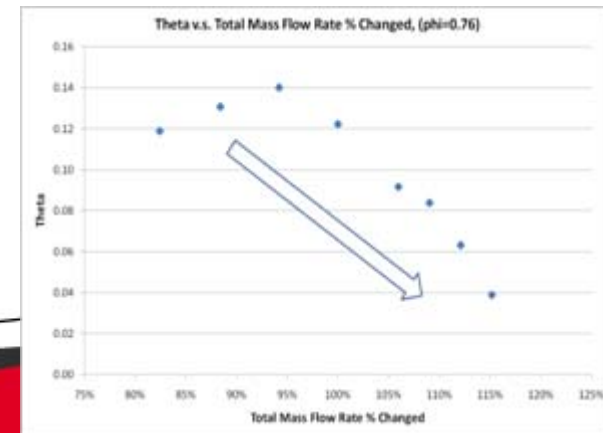
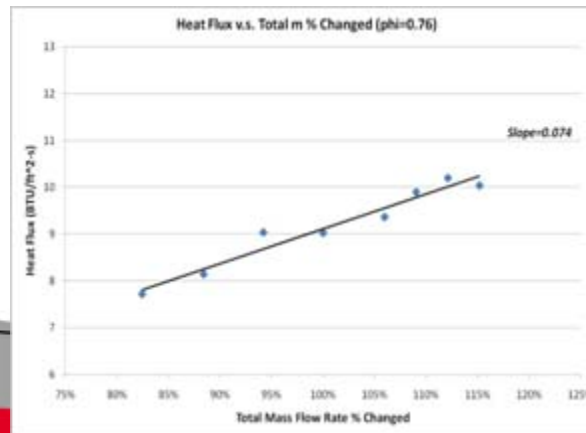
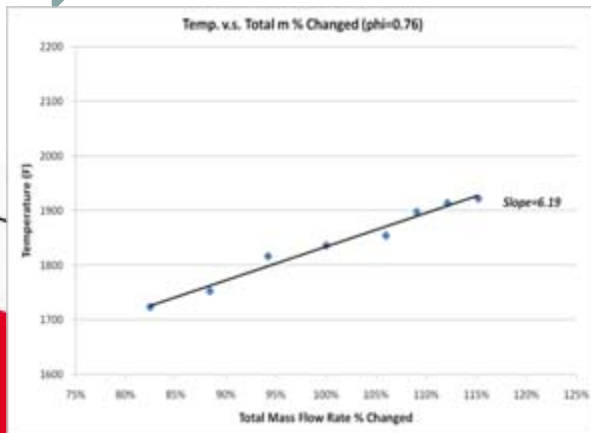


Mass Flow Rate Effect, NexGen Burner ...cont'd

Jet-A (GPH)	Air (SCFM)	Total Mass (lb/min)		A/F	ϕ	T_adia(F)	T_avg (F)	T_max(F)	Heat Flux (BTU/ft ² -s)	θ
								T_min(F)		
1.62	47.6	3.71	82.45%	19.37	0.76	3109	1723	1831	7.72	0.12
								1626		
1.73	51.2	3.97	88.41%	19.39	0.76	3109	1752	1854	8.14	0.13
								1625		
1.84	54.4	4.23	94.21%	19.38	0.76	3109	1816	1909	9.04	0.14
								1654		
1.96	57.8	4.50	100.00%	19.37	0.76	3109	1836	1933	9.03	0.12
								1708		
2.08	61.2	4.76	105.96%	19.39	0.76	3109	1854	1925	9.37	0.09
								1755		
2.14	63	4.90	109.02%	19.38	0.76	3109	1897	1962	9.90	0.08
								1803		
2.2	64.8	5.04	112.09%	19.38	0.76	3109	1914	1972	10.20	0.06
								1851		
2.25	66.6	5.18	115.16%	19.38	0.76	3109	1922	1958	10.04	0.04
								1883		



:baseline point

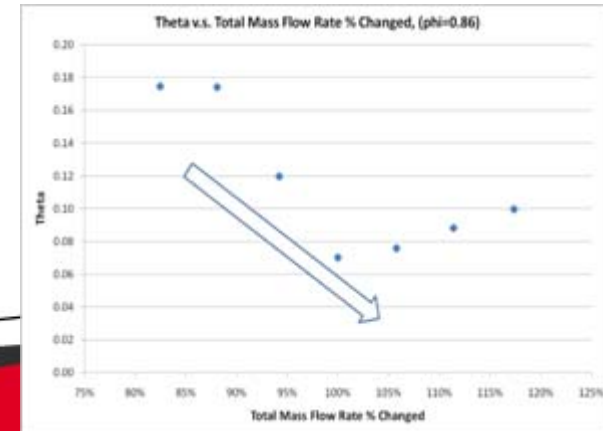
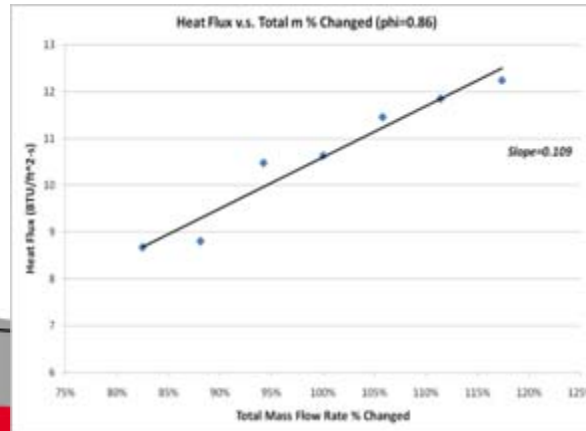
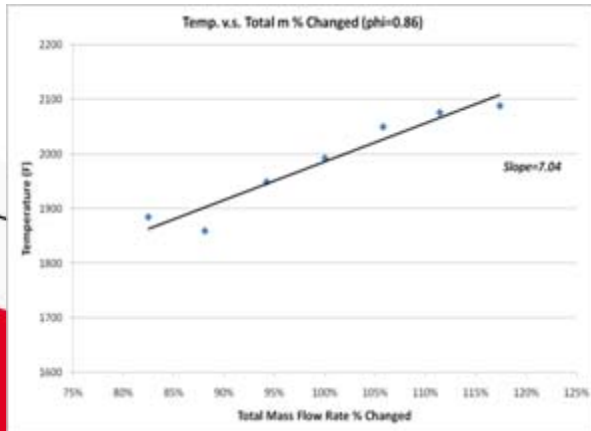


Mass Flow Rate Effect, NexGen Burner ...cont'd

Jet-A (GPH)	Air (SCFM)	Total Mass (lb/min)		A/F	ϕ	T_adia(F)	T_avg (F)	T_max(F)	Heat Flux (BTU/ft ² -s)	θ
								T_min(F)		
1.83	47.8	3.74	82.49%	17.11	0.86	3379	1884	1990 1661	98.54	0.17
1.96	51	3.99	88.11%	17.09	0.86	3379	1859	1957 1633	100.01	0.17
2.09	54.6	4.27	94.22%	17.12	0.86	3379	1948	2035 1802	119.06	0.12
2.22	58	4.53	100.00%	17.13	0.86	3379	1992	2047 1908	120.82	0.07
2.35	61.2	4.79	105.78%	17.14	0.86	3379	2049	2105 1950	130.19	0.08
2.48	64.6	5.05	111.40%	17.13	0.86	3379	2075	2134 1951	134.66	0.09
2.61	68	5.32	117.35%	17.12	0.86	3379	2088	2168 1960	139.12	0.10



:baseline point

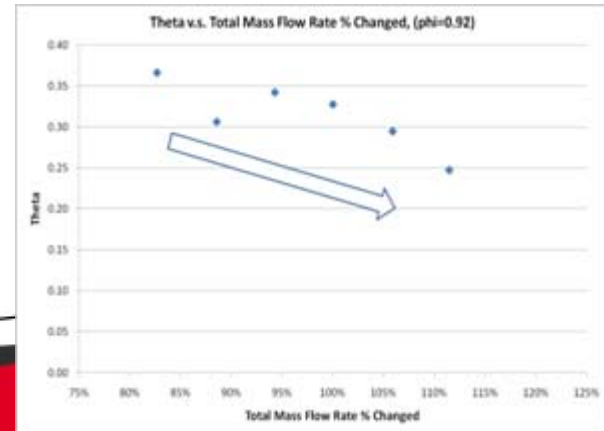
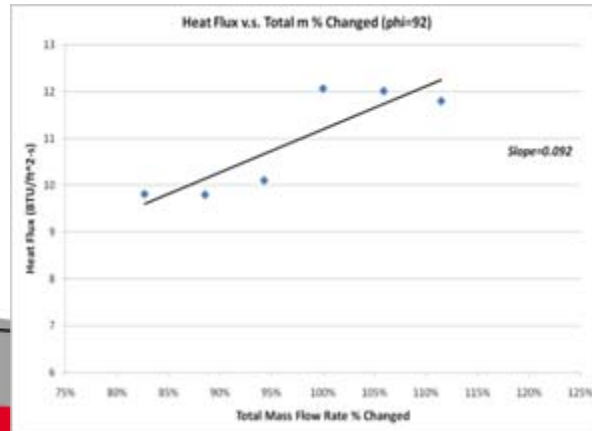
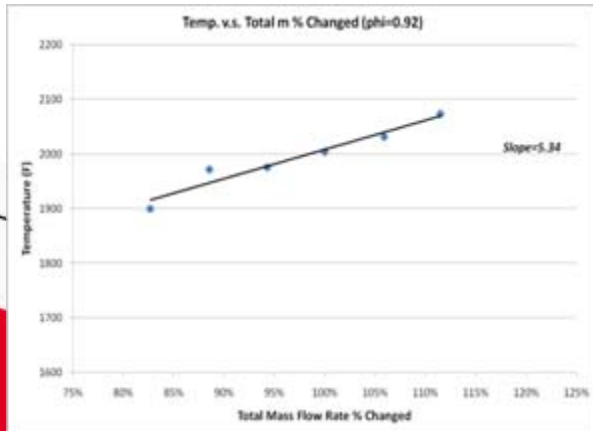


Mass Flow Rate Effect, NexGen Burner ...cont'd

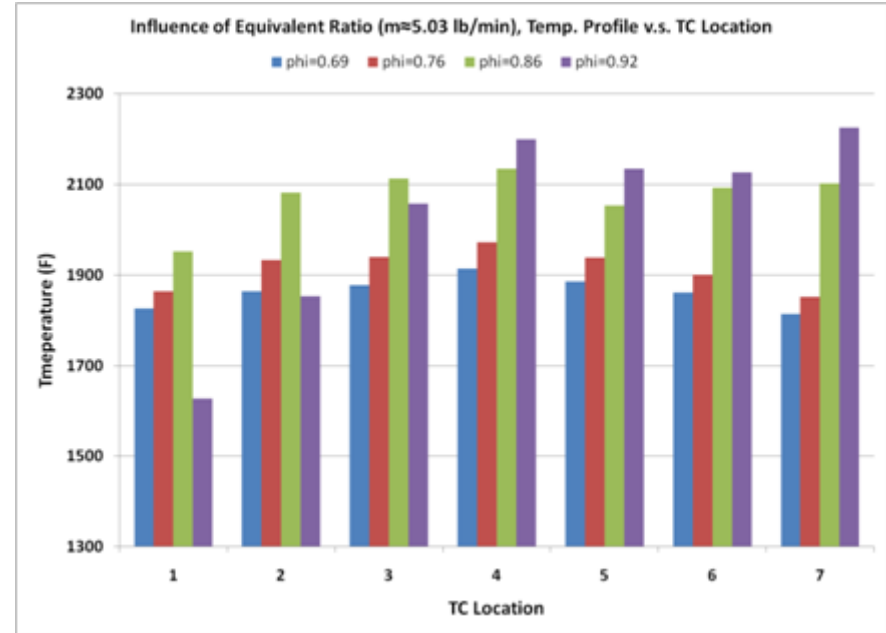
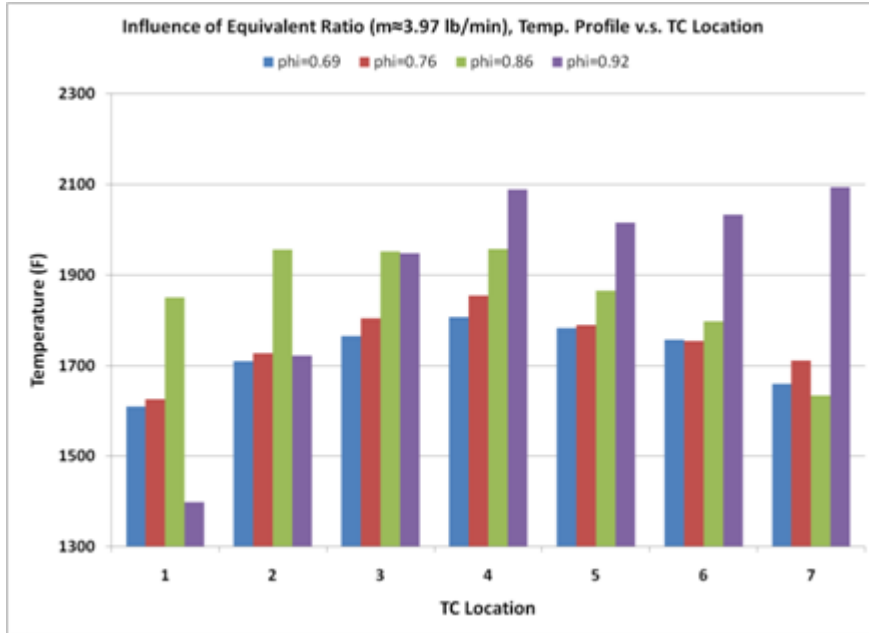
Jet-A (GPH)	Air (SCFM)	Total Mass (lb/min)		A/F	ϕ	T_adia(F)	T_avg (F)	T_max(F)	Heat Flux (BTU/ft ² -s)	θ
								T_min(F)		
2.05	49.9	3.92	82.68%	15.94	0.92	3511	1899	2094	111.49	0.37
								1398		
2.2	53.5	4.20	88.55%	15.94	0.92	3511	1972	2142	111.32	0.31
								1539		
2.35	56.9	4.47	94.28%	15.93	0.92	3511	1975	2182	114.79	0.34
								1507		
2.49	60.3	4.74	100.00%	15.95	0.92	3511	2004	2209	137.10	0.33
								1553		
2.63	63.9	5.02	105.88%	15.92	0.92	3511	2031	2225	136.49	0.29
								1627		
2.77	67.3	5.29	111.46%	15.92	0.92	3511	2073	2244	134.06	0.25
								1731		



:baseline point



Influence of Equivalent Ratio, ϕ , NexGen Burner



NexGen Burner

- Nexgen burner is more sensitive to fuel flow rate change than air flow rate change.
- TC bead size impacts on the measured “TC temperature” under the same flame.
- TC temperature is significantly lower than true flame temperature.
- Higher total mass flow rate can produce higher temperature and higher heat flux at the same A/F (equivalent ratio) conditions.
- Higher total mass flow rate can produce more uniform flame temperature.
- At the similar total mass flow rate conditions, fuel leaner operating (higher A/F ratio) provide more uniform temperature distribution.